

An input-output based analysis of convective velocity in turbulent channels

Chang Liu and Dennice F. Gayme †

Department of Mechanical Engineering, Johns Hopkins University, Baltimore, MD 21218, USA

(Received ?; revised ?; accepted ?. - To be entered by editorial office)

This paper employs an input-output based approach to analyze convective velocities and the transport of fluctuations in turbulent channel flows. The convective velocity for a fluctuating quantity associated with streamwise–spanwise wavelength pairs at each wall-normal location is obtained through the maximization of the power spectral density associated with the linearized Navier-Stokes equations with a turbulent mean profile and delta-correlated Gaussian forcing. We first demonstrate that the mean convective velocities computed in this manner agree well with those reported previously in the literature. We then exploit the analytical framework to probe the underlying mechanisms contributing to the local convective velocity at different wall-normal locations by isolating the contributions of each streamwise–spanwise wavelength pair (flow scale). The resulting analysis suggests that the behavior of the convective velocity in the near-wall region is influenced by large-scale structures further away from the wall. These structures resemble Townsend’s attached eddies in the cross-plane, yet show incomplete similarity in the streamwise direction. We then investigate the role of each linear term in the momentum equation to isolate the contribution of the pressure, mean shear, and viscous effects to the deviation of the convective velocity from the mean at each flow scale. Our analysis highlights the role of the viscous effects, particularly in regards to large channel spanning structures whose influence extends to the near-wall region. The results of this work suggest the promise of an input-output approach for analyzing convective velocity across a range of flow scales using only the mean velocity profile.

Key words: Convective velocity; Turbulent channel; Taylor’s hypothesis; Linearized Navier-Stokes

1. Introduction

Taylor’s frozen turbulence hypothesis (Taylor 1938) and its variants have proven invaluable in the study of high Reynolds number wall-bounded turbulent flows (Marusic *et al.* 2010; Smits *et al.* 2011; LeHew *et al.* 2011). However, its underlying assumption that the motion of turbulent fluctuations can be modelled as passive advection by the local mean velocity is known to break down in certain flow regimes (Lin 1953; Dennis & Nickels 2008; Squire *et al.* 2017). In order to

† Email address for correspondence: dennice@jhu.edu

compensate for these known errors, the local mean velocity is often replaced by a convective velocity that better represents the spatio-temporal development of the fluctuations (Zaman & Hussain 1981; Hutchins *et al.* 2011). This convective velocity can be computed from simulation data (Kim & Hussain 1993; del Álamo & Jiménez 2009; Chung & McKeon 2010; Lozano-Durán & Jiménez 2014; Geng *et al.* 2015; Renard & Deck 2015) or obtained from spatio-temporally resolved experimental measurements (Krogstad *et al.* 1998; LeHew *et al.* 2010; LeHew *et al.* 2011; de Kat & Ganapathisubramani 2015). However, questions remain regarding how to obtain an accurate estimate of this quantity, particularly in situations where the relevant data are unavailable; e.g., in experiments using hotwire measurements or planar PIV. In addition, there is not yet a full understanding of the mechanisms contributing to the convective velocity in each region of the flow. Such knowledge is required both to characterize the transport properties of fluctuating quantities, and to identify when direct application of Taylor’s hypothesis with the mean velocity is insufficient.

An early work by Lin (1953) suggests that Taylor’s hypothesis works well when the mean flow is approximately spatially uniform and when turbulence intensities are low, but breaks down in regions of high shear. Lumley (1965) further refines this spatial uniformity requirement, suggesting that weak interactions between eddies of different sizes are also needed to ensure the validity of Taylor’s hypothesis. Geng *et al.* (2015) provide support for the break down of Taylor’s hypothesis in highly sheared regions of the flow by explicitly computing the contribution of advection by the mean flow (Taylor’s hypothesis) to convective velocity in the viscous sublayer using Direct Numerical Simulation (DNS) data from channel flows at $Re_\tau = 205$ and 932. In particular, they compute the average amplitudes of different terms in the momentum equation through DNS and illustrate that advection by the mean flow provides less than 50% of the streamwise momentum flux in the viscous sublayer. Taylor’s hypothesis has also proven to inadequately describe the convection of large-scale components of the flow. Dennis & Nickels (2008) compare the spatial evolution of a turbulent flow inferred from the temporal information using Taylor’s hypothesis with those obtained using Particle Image Velocimetry (PIV) at a wall parallel plane sufficiently removed from the wall so that the assumptions of Lin (1953) are satisfied. The authors find that even though the PIV fields are qualitatively similar, several large-scale features are not reproduced in the fields generated using Taylor’s hypothesis.

The validity of Taylor’s hypothesis across a range of flow scales is explored by Fisher & Davies (1964) who use two-point space-time correlation for statistically stationary turbulence to compute the convective velocity of streamwise velocity fluctuations as a function of streamwise spatial and temporal separation; i.e., a streamwise (or temporal) scale-dependent convective velocity. Fisher & Davies (1964) and subsequently Favre *et al.* (1967); Zaman & Hussain (1981) show that the convective velocity computed in this manner does not coincide with the local mean velocity and can be strongly dependent on the streamwise spatial or temporal separation. A similar phenomenon has also been observed in measurements of wall-pressure (Willmarth & Wooldridge 1962).

This scale dependence, particularly the increasing deviation of the convective

velocity from the mean flow as spatial separation is increased, was identified as a possible explanation for the known discrepancy between the convective and mean velocities near the wall. In particular, the larger convective velocity versus the local mean velocity in the near-wall region has been attributed to faster moving structures centered further away from the wall whose influence extends to the wall due to their large size (Dinkelacker *et al.* 1977; Kreplin & Eckelmann 1979; Farabee & Casarella 1991; Kim & Hussain 1993; Hutchins *et al.* 2011). del Álamo & Jiménez (2009) find that fast and wide streamwise elongated structures, coherent up to the core region, provide a consistent contribution to the energy-weighted average convective velocity close to the wall. They relate these structures to the large modes (Bullock *et al.* 1978; del Álamo *et al.* 2004) reminiscent of Townsend’s ‘inactive’ eddies (Townsend 1961; Bradshaw 1967) and the very large-scale motions (Guala *et al.* 2006; Balakumar & Adrian 2007; Hutchins & Marusic 2007; Monty *et al.* 2007), which have been shown to modulate small-scale structures (Mathis *et al.* 2009*a,b*; Ganapathisubramani *et al.* 2012; Yang & Howland 2018). Although these works provide evidence that scale interactions contribute to the breakdown of Taylor’s hypothesis at the wall, a full understanding of the underlying mechanisms across the full range of flow regimes has yet to be realized.

In this paper, we explore the mechanisms underlying the convective velocity of fluctuating quantities in wall-bounded turbulence using a spatio-temporal transfer function that enables us to isolate the contributions and interactions across the full range of flow scales. This approach allows us to compute quantities for a range of Reynolds numbers given an associated turbulent mean velocity profile. Our analytical framework is based on stochastically-forced Linearized Navier-Stokes (LNS) equations, which have a long history in the study of wall-bounded shear flows; e.g., in characterizing energy amplification associated with stochastic disturbances (Farrell & Ioannou 1993; Bamieh & Dahleh 2001) and isolating the most sensitive input-output paths (Jovanović & Bamieh 2005). The LNS equations have also proven useful in characterizing coherent structures (Smits *et al.* 2011; McKeon 2017; Jiménez 2018). For example, low rank approximations of the resolvent operator (McKeon & Sharma 2010) have been used to explain the scalings of the very large-scale motions and to reconstruct the packet hairpin vortices (Sharma & McKeon 2013). Luhar *et al.* (2014) also use the resolvent framework to predict the high-amplitude wall-pressure previously observed in experiments and simulations. Moarref *et al.* (2013) combine this framework with term balancing arguments to reproduce the inner, outer, and geometrically self-similar scalings of the streamwise energy density in turbulent channel flows. Input-output analysis of the NS equations linearized about a base profile with an eddy viscosity term (Reynolds & Hussain 1972) leads to accurate predictions of the spanwise spacing of near-wall streaks (del Álamo & Jiménez 2006; Cossu *et al.* 2009; Pujals *et al.* 2009; Hwang & Cossu 2010*a,b*) and large-scale structures (Illingworth *et al.* 2018; Madhusudan *et al.* 2019; Morra *et al.* 2019). Related work employing the impulse response (Vadarevu *et al.* 2019) of the LNS transfer function has led to self-similar vortex structures.

The transfer function of the LNS has previously been used to compute quan-

tities associated with the convective velocity of a fluctuating quantity in wall-bounded turbulent flows. For example, the resolvent framework was used to show that the phase speed of streamwise velocity fluctuations with peak contribution to the energy density deviates from the mean velocity in the near-wall region (Moarref *et al.* 2013). Luhar *et al.* (2014) also used the resolvent framework to investigate the scale dependence of wall-pressure propagation speed, which showed agreement with the empirically determined convective velocity (Panton & Linebarger 1974). Zare *et al.* (2017) also computed the convective velocity of streamwise velocity fluctuations for one specific flow scale based on the LNS equation with temporally correlated (colored) stochastic forcing. Their results show qualitatively similar behavior to convective velocities obtained by del Álamo & Jiménez (2009). These works demonstrate the utility of transfer function based approaches in computing the convective velocity. However, none of these works employed input-output analysis to investigate the underlying mechanisms that lead to the deviation of convective velocity from the local mean velocity.

This work takes steps in that direction by using an input-output framework to systematically investigate the scale-dependent convective velocity of velocity fluctuations in turbulent channels. We begin by demonstrating that the proposed approach provides good agreement with the mean convective velocity predictions for fluctuations of the three velocity components previously published in the literature (Kim & Hussain 1993; Geng *et al.* 2015). We then direct our attention to the streamwise velocity fluctuations and exploit the analytical framework to compute the convective velocity for each streamwise–spanwise wavenumber pair at different wall normal locations and to examine interactions between different scales. The results of our scale-dependent analysis are consistent with those obtained using DNS data (del Álamo & Jiménez 2009). In particular, the convective velocities predicted using the input-output based approach employed here show more variation with scale closer to the wall, with the largest variation occurring in the viscous sublayer. Our analysis suggests that this viscous sublayer behavior arises due to structures that are self-similar in the spanwise and wall-normal plane and scale with wall-normal height, which supports Townsend’s attached eddy hypothesis regarding the dominant dynamical structures in wall-bounded flows.

Finally, in the spirit of Lin’s (1953) term-by-term analysis of the momentum equation, we examine how each linear term in the momentum equation contributes to the deviation of the convective velocity from the mean. A linear analysis is expected to yield insight in this regard because both the mean shear term and the viscous term, which play critical roles in the dynamics of the near-wall region, are linear. Moreover, it was recently shown that resolvent analysis retains the fast pressure component arising from the linear interaction between the mean shear and turbulent wall-normal velocity (Luhar *et al.* 2014), and therefore our computations also include these phenomena. Our analysis employs an expression for scale-dependent convective velocities derived by del Álamo & Jiménez (2009), who did not further analyze the various terms. This work also builds upon that of Geng *et al.* (2015) who quantify each term’s contribution to the convective velocity at different wall-normal locations. Our results indicate that the viscous

term provides the largest contribution to the deviation of the convective velocity from the mean in the near-wall region. Based on these observations, we propose a viscous correction to Taylor’s hypothesis, and demonstrate that the revised model accurately reproduces the behavior of the near-wall convective velocity for large-scale structures.

The remainder of the paper is organized as follows. Section 2 describes the problem setup. We detail our transfer function based approach and numerical scheme for calculating the convective velocities in sections 3 and 4, respectively. In section 5, we apply the input-output based approach using mean velocity profiles from turbulent channel flows obtained from Lee & Moser (2015) at three different Reynolds numbers ($Re_\tau = 550, 1000, \text{ and } 5200$). We then discuss the physical origin of the near-wall convective velocities. Section 6 explores the wall-normal coherence of the structures contributing to the convective velocities at a particular wall-normal location. Section 7 analyzes the contribution of each of the linear terms in the momentum equation to the total convective velocity for each streamwise–spanwise wavenumber pair (flow scale). Based on this term-by-term analysis, a viscous correction to Taylor’s hypothesis is proposed and discussed. Section 8 concludes the paper.

2. Problem setup

We consider incompressible flow between two infinite parallel plates driven by a streamwise pressure gradient as shown in figure 1(a), where x, y, z are the streamwise, wall-normal, and spanwise directions, respectively. We decompose the velocity field, $\mathbf{u} = [u \ v \ w]^T$, with T indicating the transpose, and the pressure field, p , into mean and fluctuating quantities; i.e., $\mathbf{u} = \bar{\mathbf{u}}(y)\mathbf{i} + \mathbf{u}'$ with $\mathbf{i}$ denoting the streamwise unit vector and $p = \bar{p} + p'$, where the overbars indicate time averaged quantities, $\bar{\phi} = \lim_{T \rightarrow \infty} \frac{1}{T} \int_0^T \phi(t) dt$, and primes indicate fluctuating quantities. The dynamics of the fluctuations $\mathbf{u}'$ and p' are governed by

$$\partial_t \mathbf{u}' + \bar{u} \partial_x \mathbf{u}' + \nabla p' + \nu' \frac{d\bar{\mathbf{u}}}{dy} \mathbf{i} - \frac{1}{Re_\tau} \Delta \mathbf{u}' = -\mathbf{u}' \cdot \nabla \mathbf{u}' + \overline{\mathbf{u}' \cdot \nabla \mathbf{u}'}, \quad (2.1a)$$

$$\nabla \cdot \mathbf{u}' = 0. \quad (2.1b)$$

The spatial variables are normalized by the half channel height δ ; e.g., $y = y_*/\delta \in [-1, 1]$, where the subscript $*$ indicates dimensional quantities. The velocity is normalized by the friction velocity $U_\tau = \sqrt{\tau_w/\rho}$, where τ_w is the time-averaged mean shear stress at the wall, and ρ is the density of the fluid, which leads to $u := u^+ = u_*/U_\tau$.[†] Time and pressure are normalized by δ/U_τ , and ρU_τ^2 , respectively. We define the inner unit length scale as $\delta_v = \nu/U_\tau$ and use the superscript $+$ to denote the distances measured in inner units; i.e., $y^+ = y_*/\delta_v$. The friction Reynolds number is defined as $Re_\tau = \delta U_\tau/\nu = \delta^+$.

[†] Note, in (2.1), we omit the $+$ superscripts for the velocity fluctuations for notational convenience.

(a)

(b)

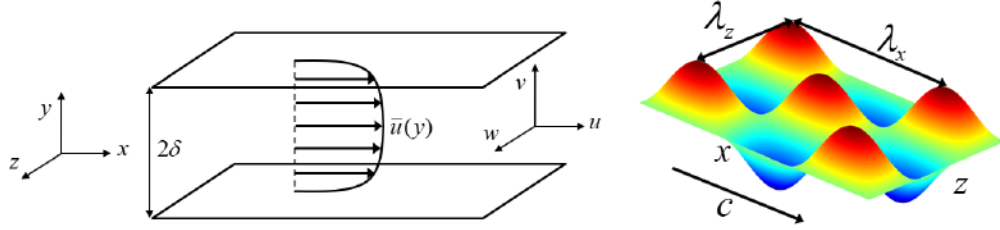


Figure 1: (a) Turbulent flow between two infinite parallel plates with mean profile $\bar{u}(y)$. (b) The fluctuations $\mathbf{u}'$ are decomposed into traveling waves with wavelengths λ_x, λ_z in the x, z directions and downstream phase speed $c = -\lambda_x \omega / 2\pi$ using equation (2.2).

Invariance to shifts in (x, z, t) of equations (2.1) allows us to employ the (x, z, t) spatio-temporal Fourier transform,

$$\hat{\psi}(y; k_x, k_z, c) = \mathcal{F}(\psi) \equiv \int_{-\infty}^{\infty} \int_{-\infty}^{\infty} \int_{-\infty}^{\infty} \psi(x, y, z, t) e^{-i(k_x(x-ct) + k_z z)} dt dx dz, \quad (2.2)$$

where $k_x = 2\pi/\lambda_x$ and $k_z = 2\pi/\lambda_z$ are the respective dimensionless x and z wavenumbers normalized by the channel half-height δ . The transform (2.2) decomposes the flow into traveling waves with wavelengths λ_x, λ_z in the x, z directions and downstream phase speed $c = -\omega/k_x = -\lambda_x \omega / 2\pi$, where ω is the frequency; see figure 1(b). Applying the Fourier transform to (2.1) and denoting the nonlinear term as: $\mathbf{f}'(x, y, z, t) \equiv -\mathbf{u}' \cdot \nabla \mathbf{u}' + \overline{\mathbf{u}' \cdot \nabla \mathbf{u}'}$ yields

$$\underbrace{ik_x(\bar{u} - c)\hat{\mathbf{u}}'}_{\text{I}} + \underbrace{\hat{\nabla} \hat{p}'}_{\text{IIa}} + \underbrace{\hat{v}' \frac{d\bar{u}}{dy} \mathbf{i}}_{\text{IIb}} - \underbrace{\frac{1}{Re_\tau} \hat{\Delta} \hat{\mathbf{u}}'}_{\text{IIc}} = \underbrace{\mathbf{f}'}_{\text{III}}, \quad (2.3a)$$

$$\hat{\nabla} \cdot \hat{\mathbf{u}}' = 0, \quad (2.3b)$$

where $\hat{\nabla} = [ik_x \quad \partial_y \quad ik_z]^T$ and $\hat{\Delta} = \partial_{yy} - (k_x^2 + k_z^2)$. The corresponding no-slip boundary conditions are given by

$$\hat{\mathbf{u}}'(y = \pm 1; k_x, k_z, c) = [0 \quad 0 \quad 0]^T, \quad \forall k_x, k_z, c \in \mathbb{R}. \quad (2.4)$$

The terms in (2.3a) can be grouped into: (I) advection of the fluctuations by the mean velocity in the frame of reference of the traveling wave fluctuations, terms (IIa)-(IIc), which capture the respective effects of pressure, shear, and viscosity, and (III) fluctuation-fluctuation nonlinear interactions.

Taylor's frozen turbulence hypothesis states that for sufficiently low turbulence intensities ($|\mathbf{u}'_{\text{rms}}|_\infty / \bar{u} \rightarrow 0$), the spatio-temporal development of turbulent fluctuations can be described as downstream advection by the mean velocity $\bar{u}(y)$ (Taylor 1938). This statement is equivalent to setting all terms except (I) in (2.3a) to zero, which reduces the equation (2.3a) to the passive advection model:

$$ik_x(\bar{u} - c)\hat{\mathbf{u}}' = 0. \quad (2.5)$$

As previously discussed, the direct application of Taylor's hypothesis tends to

be inaccurate in regions near the wall, where the terms in (IIa)-(IIc) and (III) in (2.3a) provide non-negligible contributions; see e.g., Lin (1953); Geng *et al.* (2015). It is well-known that the model in (2.5) can be improved by replacing the mean velocity, $\bar{u}$, with an empirically determined mean convective velocity ψ_c (Zaman & Hussain 1981; Hutchins *et al.* 2011) for the fluctuating quantity of interest. In the next section, we describe an input-output based approach to computing such a ψ_c .

3. Method for calculating scale-dependent convective velocities

In this section, we describe the employed input-output approach to computing spatio-temporal convective velocities of fluctuating quantities given a mean velocity profile $\bar{u}(y)$. First we rewrite (2.3) in the form

$$\mathcal{L} \begin{bmatrix} \hat{u}' \\ \hat{p}' \end{bmatrix} = \mathcal{B} \hat{f}', \quad (3.1)$$

where

$$\mathcal{L} := \begin{bmatrix} (\mathrm{i}k_x(\bar{u} - c) - \frac{1}{Re_\tau} \hat{\Delta}) \mathbf{I}_{3 \times 3} + \frac{\mathrm{d}\bar{u}}{\mathrm{d}y} \mathbf{S} & \hat{\nabla} \\ \hat{\nabla}^\top & 0 \end{bmatrix}, \quad \mathcal{B} := \begin{bmatrix} \mathbf{I}_{3 \times 3} \\ \mathbf{0}_{1 \times 3} \end{bmatrix}, \quad (3.2)$$

and $\hat{f}'$ is parameterized as input forcing. In equation (3.2), $\mathbf{I}_{n \times n}$ and $\mathbf{0}_{m \times n}$ are respective identity and zero matrices with their sizes indicated by their subscripts, and $\mathbf{S} := \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix}$. A non-bold symbol represents a scalar; e.g., the 0 appearing in (3.2) is a scalar quantity. We then define an output variable

$$\hat{\psi}' = \mathcal{C}_{\hat{\psi}'} \begin{bmatrix} \hat{u}' \\ \hat{p}' \end{bmatrix}, \quad (3.3)$$

where $\mathcal{C}_{\hat{\psi}'} = \mathcal{C}_{\hat{\psi}'}(y; k_x, k_z)$ is a linear operator that maps the state variables to the output of interest. Here, we use the primitive variables $\hat{u}'$ and $\hat{p}'$ as states rather than the wall-normal velocity and vorticity coordinates of the commonly studied Orr-Sommerfeld and Squire equations because this form of the equations provides direct information about the pressure (Luhar *et al.* 2014), which we will later exploit in the term-by-term analysis in section 7.

We define the input-output map $\mathcal{G}_{\hat{\psi}'}$ between the input $\hat{f}'$ and the output $\hat{\psi}'$, in the manner of McKeon & Sharma (2010); Luhar *et al.* (2014); McKeon (2017) as

$$\hat{\psi}' = \mathcal{C}_{\hat{\psi}'} \mathcal{L}^{-1} \mathcal{B} \hat{f}' = \mathcal{G}_{\hat{\psi}'}(y; k_x, k_z, c) \hat{f}'. \quad (3.4)$$

The convective velocity of a fluctuating variable ψ' can be computed using the following two-point space-time correlation for statistically stationary turbulence:

$$R_{\hat{\psi}'}(\xi, \tau; \mathbf{x}) = \langle \psi'(\mathbf{x}, t) \psi'(\mathbf{x} + \xi \mathbf{i}, t + \tau) \rangle, \quad \psi = u, v, w, \omega_x, \omega_y, \omega_z, p, \quad (3.5)$$

where ξ and τ are the respective streamwise and temporal separation between two points. Convective velocities for fluctuations ψ' at some $\mathbf{x}$ are computed from

(3.5) by fixing either ξ or τ and varying the other separation variable to maximize $R_{\hat{\psi}'}(\xi, \tau; \mathbf{x})$ (Wills 1964; Fisher & Davies 1964; Kim & Hussain 1993; Zaman & Hussain 1981; Krogstad *et al.* 1998). We adapt this idea to our approach by computing the Power Spectral Density (PSD) (Wills 1964; del Álamo & Jiménez 2009) for the input-output map (3.4) as

$$\Phi_{\hat{\psi}'}(y; k_x, k_z, c) = \langle \hat{\psi}' \hat{\psi}'^* \rangle = \mathcal{G}_{\hat{\psi}'} \langle \hat{\mathbf{f}}' \hat{\mathbf{f}}'^* \rangle \mathcal{G}_{\hat{\psi}'}^* = \mathcal{G}_{\hat{\psi}'} \mathcal{G}_{\hat{\psi}'}^* \quad (3.6)$$

with $\mathbf{f}'(x, y, z, t)$ parametrized as spatio-temporal delta-correlated Gaussian noise with unit variance; i.e., noise that is white in space and time (Jovanović & Bamieh 2005). The superscript $*$ in (3.6) denotes the complex conjugate, and the angle brackets $\langle \rangle$ indicate an ensemble averaging operation.

The convective velocity ψ_c is then obtained as

$$\psi_c(y; k_x, k_z) \equiv \arg \max_c \Phi_{\hat{\psi}'}(y; k_x, k_z, c), \quad (3.7)$$

which represents convective velocities of the coherent structures with (x, z) spatial extents of $\lambda_x = 2\pi/k_x$ and $\lambda_z = 2\pi/k_z$ as a function of wall-normal location. This definition of convective velocity based on (3.7) neglects the distribution of the PSD for a given (λ_x, λ_z) pair, which is expected to contain energy at a range of temporal frequencies. The distribution of the spectrum could be partially accounted for by instead defining the convective velocity in terms of the center of gravity of the PSD. That quantity is commonly used to compute convective velocity in simulation (DNS and LES) studies as it requires time-averaging instead of Fourier transforming in the time domain; see e.g., del Álamo & Jiménez (2009); Chung & McKeon (2010); Renard & Deck (2015). Our approach can be adapted to accommodate such a definition (and others) through a suitable modification of equation (3.7). In the current work, we recomputed a subset of the results using the center of gravity method to ensure that the main conclusions of our study are not altered by our choice of definition.

Assuming that $\mathbf{f}'(x, y, z, t)$ is spatio-temporal delta-correlated Gaussian noise with unit variance implies that the velocity itself is Gaussian. This is clearly not true as velocity probability density functions are known to have heavy tails and odd-order moments that do not vanish (Frisch & Kolmogorov 1995). Colored-in-time forcing has been shown to produce more accurate statistics (Zare *et al.* 2017). However, the Gaussian white-noise parametrization is appealing because it is a simple, analytically tractable forcing that has been widely used to provide important insights into the dynamics; e.g., Farrell & Ioannou (1993); Bamieh & Dahleh (2001); Jovanović & Bamieh (2005). Therefore, this type of forcing provides a good starting point for understanding the role of linear mechanisms in determining the convective velocity and simplifies analysis because it does not introduce a preferential forcing in any of the spatial or temporal directions.

We focus on streamwise, wall-normal, and spanwise velocity fluctuations which are computed based on the respective output operators

$$\mathcal{C}_{\hat{u}'} = [1 \ 0 \ 0 \ 0], \quad \mathcal{C}_{\hat{v}'} = [0 \ 1 \ 0 \ 0], \quad \text{and} \quad \mathcal{C}_{\hat{w}'} = [0 \ 0 \ 1 \ 0] \quad (3.8)$$

in (3.4). However, we note that the approach can be generalized to the calculation of the convective velocity for any fluctuating quantity, ψ' , whose Fourier transform can be written in the form (3.3) with an appropriate choice of $\mathcal{C}_{\hat{\psi}'}$. For example, the output operators corresponding to the vorticity fluctuations $\hat{\omega}'_x$, $\hat{\omega}'_y$, $\hat{\omega}'_z$, and the pressure $\hat{p}'$ are given by:

$$\begin{aligned} \mathcal{C}_{\hat{\omega}'_x} &= \begin{bmatrix} 0 & -ik_z & \partial_y & 0 \end{bmatrix}, & \mathcal{C}_{\hat{\omega}'_y} &= \begin{bmatrix} ik_z & 0 & -ik_x & 0 \end{bmatrix}, \\ \mathcal{C}_{\hat{\omega}'_z} &= \begin{bmatrix} -\partial_y & ik_x & 0 & 0 \end{bmatrix}, & \mathcal{C}_{\hat{p}'} &= \begin{bmatrix} 0 & 0 & 0 & 1 \end{bmatrix}, \end{aligned} \quad (3.9)$$

respectively. An analysis of the convective velocity of vorticity fluctuations is carried out in Liu & Gayme (2019).

In the next section, we describe the numerical implementation of the input-output based approach for channel flow at three different Reynolds numbers. The resulting convective velocities are analyzed in subsequent sections.

4. Numerical approach

The operators in (3.6) are discretized using the Chebyshev differentiation matrices generated by the MATLAB routines of Weideman & Reddy (2000). We denote the discretization of $\mathcal{G}_{\hat{\psi}'}$ as $\tilde{\mathcal{G}}_{\hat{\psi}'}$. The resulting discretized expression for the PSD at wall-normal location $y = y_i$ is given by

$$\tilde{\Phi}_{\hat{\psi}'}(y_i; k_x, k_z, c) = (\tilde{\Phi}_{\hat{\psi}'}(\mathbf{y}; k_x, k_z, c))_i = \left(\tilde{\mathcal{G}}_{\hat{\psi}'}(\mathbf{y}; k_x, k_z, c) \tilde{\mathcal{G}}_{\hat{\psi}'}^*(\mathbf{y}; k_x, k_z, c) \right)_{ii}, \quad (4.1)$$

where $\mathbf{y} = \{y_1, y_2, \dots\}$ are the discrete grid points in the wall-normal direction, and $(A)_{ij}$ indicates the (i, j) element of the matrix A . The convective velocity at a fixed $(y; k_x, k_z)$ can then be approximated through the discretized analog of (3.7). In computing this quantity, we employ the Clenshaw–Curtis quadrature (Trefethen 2000) to obtain the L_2 inner product for the Chebyshev spaced wall-normal grid. We implement the no slip boundary condition $\hat{\mathbf{u}}'(y = \pm 1) = \mathbf{0}$ explicitly following the approach of Luhar *et al.* (2014). This implementation allows us to use primitive variables $\hat{\mathbf{u}}'$ and $\hat{p}'$, which as previously discussed offers us direct information regarding the fast pressure. We performed the same analysis using the Orr–Sommerfeld and Squire form described in Jovanović & Bamieh (2005) and verified that results do not change.

The turbulent mean velocities in (2.1a) are obtained from the DNS of Lee & Moser (2015) at $Re_\tau = 550, 1000$, and 5200, which all use simulation domains with $L_{x*}/\delta = 8\pi$ and $L_{z*}/\delta = 3\pi$. For the $Re_\tau = 550$ and $Re_\tau = 1000$ cases, our calculations use 122 collocation points in the wall-normal direction. We employ 192 collocation points for the $Re_\tau = 5200$ calculations. We compute the optimal value of equation (3.7) by computing the PSD for 201 uniformly spaced points over the phase speed range $c^+ \in [0, 30]$ for each wavenumber pair (k_x, k_z) . We then select the single c^+ that maximizes the PSD. This phase speed range and 90×90 logarithmically spaced points in the spectral range $k_x \in [10^{-2}, 10^3]$ and $k_z \in [10^{-2}, 10^3]$ are employed for all three Reynolds numbers. We verified that doubling either the number of collocation points in the wall-normal direction

or the number of Fourier modes in the horizontal directions does not alter the results, indicating grid convergence.

5. Convective velocity in turbulent channels

In this section, we use the method described in sections 3 and 4 to compute the convective velocity of the velocity fluctuations for turbulent channel flow at $Re_\tau = 550$, $Re_\tau = 1000$, and $Re_\tau = 5200$. We first validate the approach by computing the mean convective velocities and comparing our results to those computed from DNS data (del Álamo & Jiménez 2009; Geng *et al.* 2015). We then take advantage of the analytical framework to further analyze the contribution of different length scales to the local convective velocity.

5.1. Validation of the input-output based approach

The weighted average convective velocity $[\psi_c]_h$ of a fluctuating quantity ψ' can be computed as:

$$[\psi_c]_h(y) = \frac{\int_{\Omega} \psi_c(y; k_x, k_z) h(y; k_x, k_z) dk_x dk_z}{\int_{\Omega} h(y; k_x, k_z) dk_x dk_z} \quad (5.1)$$

with an averaging domain Ω over (k_x, k_z) and a weighting function $h(y; k_x, k_z) = \langle |\mathcal{F}_{xz}(\psi')|^2 \rangle k_x^2$, where $\mathcal{F}_{xz}$ is the x - z Fourier transform:

$$\mathcal{F}_{xz}(\psi')(y; k_x, k_z, t) \equiv \int_{-\infty}^{\infty} \int_{-\infty}^{\infty} \psi'(x, y, z, t) e^{-i(k_x x + k_z z)} dx dz. \quad (5.2)$$

We compute this average quantity for each of the three fluctuating velocity components by first computing the convective velocity of each component using (3.6) and (3.7) with the corresponding output operators given in (3.8). These quantities are then filled into (5.1) with an averaging domain $\Omega : (\lambda_x^+, \lambda_z^+) > (500, 80)$. The weighting function $h(y; k_x, k_z)$ in equation (5.1) is selected to provide the least-squares fit to the passive advection model: $\partial_t \psi' + [\psi_c]_h \partial_x \psi' = 0$, $\psi' = u', v', w'$ as discussed by del Álamo & Jiménez (2009). This choice allows a direct comparison with Geng *et al.* (2015), who explicitly employed this fit in their computations.

The averaging domain $\Omega : (\lambda_x^+, \lambda_z^+) > (500, 80)$ was chosen to include the sub-layer streaks proposed as the source of the elevated near-wall convective velocity (Kim & Hussain 1993) but to avoid the nonlinear effects that dominate at smaller scales, where our linear analysis is not expected to be valid. The limitations of our input-output based approach at these smaller scales can be understood by examining the energy spectrum that is compared to that of DNS data in figure 2. Here it is clear that the spectrum for DNS falls off much faster with decreasing wavelength than the spectrum obtained using the input-output based approach in this work. The relatively heavier weighting at the small scales (wavelength) structures imposed through the present approach results in a lower overall convective velocity in these regions. Our choice of averaging domain eliminates the effect of this heavier weighting and implicitly assumes that smaller wavelengths

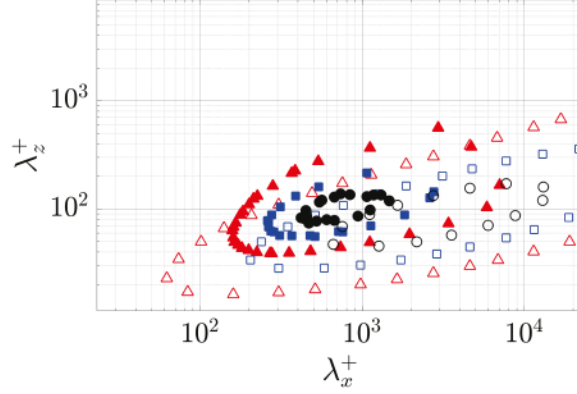


Figure 2: Premultiplied two-dimensional spectral density of streamwise velocity fluctuations $k_x k_z \int \Phi_{\hat{u}'}(y; k_x, k_z, c) dc$ at $y^+ \approx 5$ for Reynolds number $Re_\tau = 1000$. Contours are shown for 0.2 ($\triangle$); 0.5 ($\square$); 0.8 ($\circ$) times the maximum value computed using the present approach. Results are plotted with the premultiplied spectral density of streamwise velocity fluctuations computed from DNS data. Contours are plotted at 0.2 ($\blacktriangle$); 0.5 ($\blacksquare$); 0.8 ($\bullet$) times the maximum value from DNS data at $Re_\tau = 934$ (del Álamo *et al.* 2004) (<https://torroja.dmt.upm.es/channels/data/spectra/>).

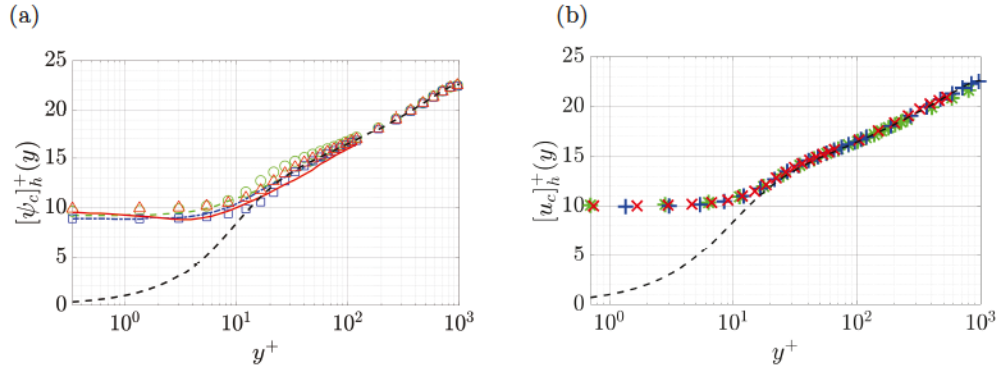


Figure 3: (a) The average convective velocity of velocity fluctuations, $[\psi_c]_h^+(y)$: $\psi' = u'$ ($\triangle$); $\psi' = v'$ ($\square$); $\psi' = w'$ ($\circ$) computed using the present approach and (5.1) with their corresponding weighting functions $h = \langle |\mathcal{F}_{xz}(\psi')|^2 \rangle k_x^2$ and an averaging domain of $(\lambda_x^+, \lambda_z^+) > (500, 80)$ at $Re_\tau = 1000$. Results are plotted with convective velocities computed from DNS data (Geng *et al.* 2015) at $Re_\tau = 932$: $\psi' = u'$ ($-$); $\psi' = v'$ ($- \cdot -$); $\psi' = w'$ ($- -$). (b) The model-based average convective velocity for streamwise fluctuations $\psi' = u'$ computed from (5.1) with the weighting function $h = \langle |\mathcal{F}_{xz}(u')|^2 \rangle k_x^2$ over averaging domain $(\lambda_x^+, \lambda_z^+) > (500, 80)$ at $Re_\tau = 550$ ($\times$); $Re_\tau = 1000$ ($+$); $Re_\tau = 5200$ ($*$). The black dashed lines in both (a) and (b) are the turbulent mean velocity profile at $Re_\tau \approx 1000$ from Lee & Moser (2015).

are energetically negligible. Therefore, these small wavelength components do not contribute to the average convective velocity computed using our input-output approach.

The performance of the input-output based model at small scales may be improved by integrating known correlations from DNS or experimental data, e.g., shaping the forcing based on spatial or temporal correlations obtained via simulation data (Moarref *et al.* 2014; Zare *et al.* 2017). Improvements could also potentially be realized by using an eddy viscosity based enhancement of the LNS equations (Reynolds & Hussain 1972), which Zare *et al.* (2017) have shown can provide similar improvements to the input-output response as the introduction of colored-in-time forcing. Understanding the relative benefits of each of these approaches over the current model is a topic of ongoing work.

Figure 3(a) compares the resulting mean convective velocities to those obtained from the DNS data based computations of Geng *et al.* (2015). The plot demonstrates that the model-based average convective velocities of the streamwise, wall-normal, and spanwise velocity fluctuations show good agreement with those computed from DNS data (Geng *et al.* 2015).

Figure 3(b) replots the results for the streamwise velocity fluctuations in inner units for $Re_\tau = 550$, $Re_\tau = 1000$, and $Re_\tau = 5200$. The results collapse with the average convective velocities computed from the input-output based approach at different Reynolds numbers all tending to a constant value $\approx 10u_\tau$ near the wall. This Reynolds number invariance of convective velocities is consistent with the results reported in figure 3 of Geng *et al.* (2015).

The Reynolds number dependence can be analyzed by rewriting equations (2.3a) and (2.3b) using the following change of variables $k_x = k_x^+ Re_\tau$, $\hat{\nabla} = \hat{\nabla}^+ Re_\tau$, $\frac{d\bar{u}}{dy} = Re_\tau \frac{d\bar{u}}{dy^+}$ and $\hat{\Delta} = \hat{\Delta}^+ Re_\tau^2$, which gives:

$$ik_x^+(\bar{u} - c)\hat{u}' - \hat{\Delta}^+\hat{u}' + \hat{v}'\frac{d\bar{u}}{dy^+}i + \hat{\nabla}^+\hat{p}' = \frac{\hat{f}'}{Re_\tau}, \quad (5.3a)$$

$$\hat{\nabla}^+ \cdot \hat{u}' = 0. \quad (5.3b)$$

As neither the mean velocity profile $\bar{u}$ nor the mean shear $d\bar{u}/dy^+$ at a specific y^+ vary over the Reynolds number in the near-wall region (see e.g., Chapter 7.1.4 of Pope (2000)), the left hand side of equation (5.3a) does not significantly vary over Reynolds number. The right hand side of equation (5.3a) is related to Re_τ , but the Reynolds number only influences the magnitude of stochastic forcing $\hat{f}'$. According to equation (3.6) and (3.7), the phase speed, c , at which $\Phi_{\hat{u}'}$ peaks does not change, and thus, the convective velocity of the streamwise fluctuations, u_c remains unaffected. This leads to the Reynolds number independence observed in the right panel of figure 3.

This inner units scaling was also previously observed by Moarref *et al.* (2013) for streamwise energy density and further generalized by Sharma *et al.* (2017) in the framework of resolvent analysis. They end up with the same Re_τ^{-1} scaling for the spatio-temporal transfer function as shown in equation (A4) of Moarref *et al.* (2013), and they also pointed out that Re_τ independence of turbulent mean profile $\bar{u}(y)$ for $y^+ \lesssim 100$ is necessary for this inner units scaling.

Having validated the ability of the input-output based computations to reproduce the mean trends, we next investigate the scale dependence of the convective velocity.

5.2. Scale-by-scale analysis of convective velocity

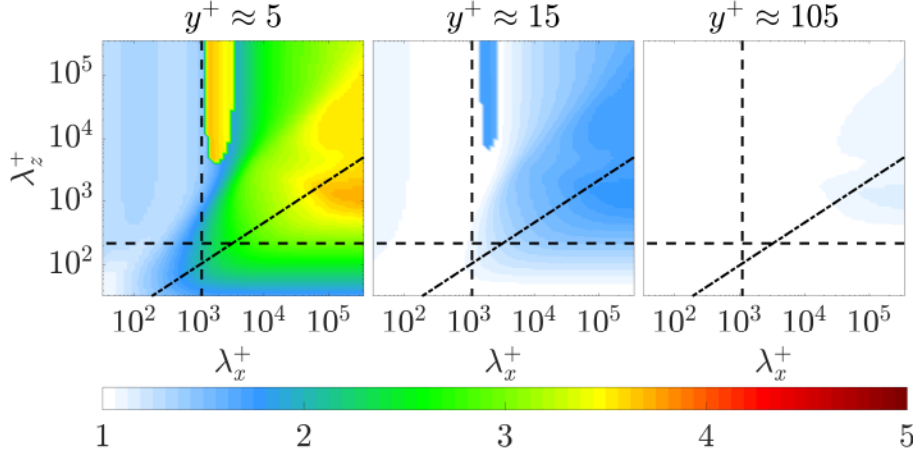
The model-based approach employed herein allows one to calculate the contribution of each individual (λ_x, λ_z) wavelength pair to the local convective velocity at each wall-normal location; i.e., $u_c(y; k_x, k_z)$. We now take advantage of this feature to investigate the scale-dependence of the convective velocity and examine which scales contribute to its deviation from the local mean velocity in various regions of the flow. Figure 4 shows the convective velocity of the streamwise velocity fluctuations normalized by the local mean velocity: $u_c(y; k_x, k_z)/\bar{u}(y)$ for (a) $Re_\tau = 550$, (b) $Re_\tau = 1000$, and (c) $Re_\tau = 5200$, as a function of the streamwise-spanwise wavelengths $(\lambda_x^+, \lambda_z^+)$ at wall-normal locations associated with the viscous sublayer ($y^+ \approx 5$), the buffer layer ($y^+ \approx 15$), and the log-law region ($y^+ \approx 100$). As expected, the results in the viscous sublayer show the most significant deviations from the mean velocity, whereas there is little difference in the log-law region, which is consistent with the data in figure 3.

In figure 4, the convective velocities of structures in the intermediate scale range $\lambda_x \approx 2$ show a discontinuity as the streamwise wavelength λ_x varies. This phenomenon was also observed for the scale-dependent convective velocity of wall-pressure in pipe flow predicted using resolvent analysis with broadband forcing; see figure 12(a) in Luhar *et al.* (2014). In the term-by-term analysis in section 7, we will further confirm that the convective velocity of structures associated with these scales is highly influenced by the pressure.

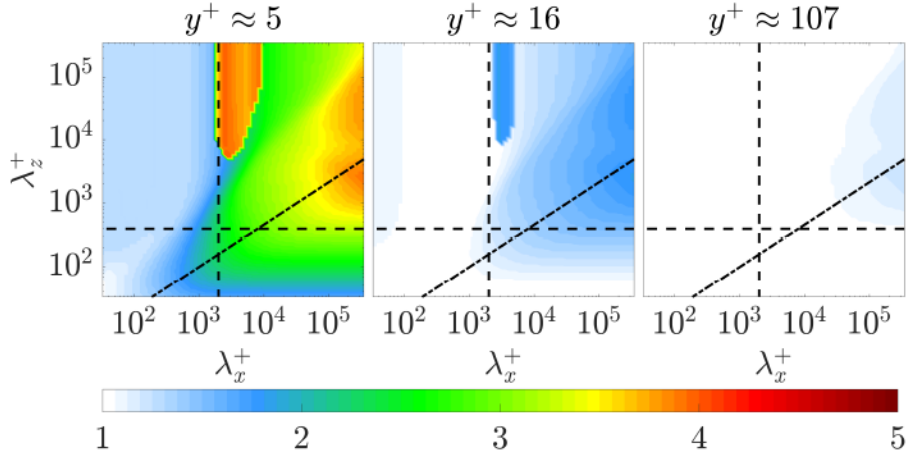
Figure 5 shows the PSD, $\Phi_{\hat{u}'}(y; k_x, k_z, c)$ computed from equation (3.6), of the streamwise velocity fluctuations for $Re_\tau = 1000$ as a function of wall-normal location y^+ and phase speed c^+ at six different streamwise-spanwise wavelength pairs. Figures 5(a) and (b) show that the energy of the large wavelength structures at example points $\triangleleft (\lambda_x^+, \lambda_z^+) = (133052, 857)$ and $\triangleright (\lambda_x^+, \lambda_z^+) = (133052, 14756)$, are concentrated near $(y^+, c^+) \approx (200, 18.4)$, and that structures traveling at $c^+ \approx 18.4$ provide the largest contribution to the spectral density in the near-wall region. Figure 5(c) indicates that structures traveling at a higher velocity than the local mean also contribute most to the PSD in the near-wall region for the intermediate-scale structures. In contrast, the PSD distributions over (c^+, y^+) for structures with small streamwise wavelengths are more concentrated near the mean velocity profile $\bar{u}$ as shown in figures 5(d), and (e) for example points $\square (\lambda_x^+, \lambda_z^+) = (11, 14756)$, and $\diamond (\lambda_x^+, \lambda_z^+) = (11, 11)$.

Figure 5 indicates that the PSD distribution over phase speed is nearly symmetric about its peak in figures 5(a), (b), (d), and (e), which indicates close correspondence between the center of gravity and peak of the PSD definitions of convective velocity. For the representative intermediate flow scale plotted in figure 5(c), the PSD distribution over the phase speed shows skewness, which is expected to lead to differences in the convective velocity obtained by considering the distribution. In order to evaluate the differences, we recomputed the results in figure 4(b) using a center of gravity definition $\check{\psi}_c(y; k_x, k_z) \equiv \frac{\int c \Phi_{\check{\psi}'}(y; k_x, k_z, c) dc}{\int \Phi_{\check{\psi}'}(y; k_x, k_z, c) dc}$ and found that there are indeed differences for these scales. Specifically, the discontinuity in the convective velocity as streamwise wavelength λ_x varies over larger λ_z is smoothed. Differences also occur at the flow scales that are very small in

(a)



(b)



(c)

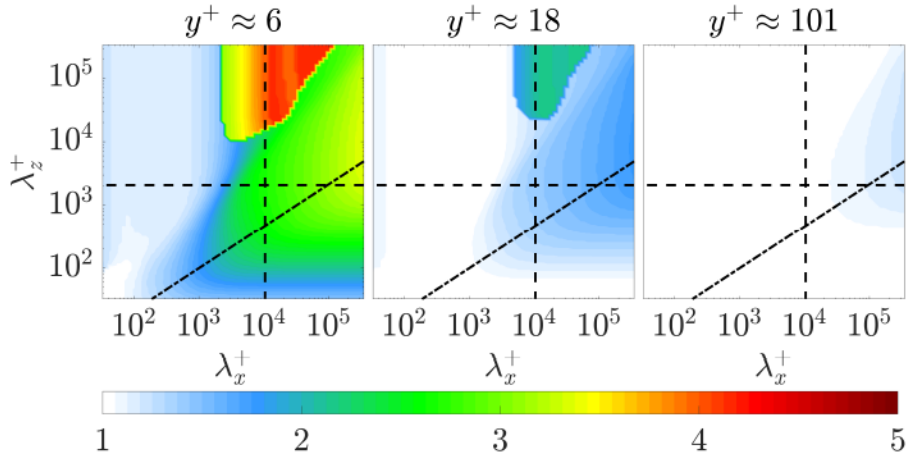


Figure 4: Scale-dependent convective velocity normalized by the local mean velocity $u_c(y; \lambda_x, \lambda_z)/\bar{u}(y)$ at (a) $Re_\tau = 550$, (b) $Re_\tau = 1000$, and (c) $Re_\tau = 5200$. The black dashed lines are given by $(\lambda_x, \lambda_z) = (2, 0.4)$, which are identified by del Álamo & Jiménez (2009) as the lower-bound of the large-scale convective velocity. The black dash-dot lines are $\lambda_z^+ = \lambda_x^{+2/3}$, which fit through the knee of these contours.

the spanwise direction but very long in the streamwise direction; e.g., the structures indicated by $\circ$ (λ_x^+, λ_z^+) = (133052, 11) in figure 5(f). These differences are not surprising because figure 5(f) demonstrates that the PSD is quite flat and therefore advection does not dominate at these flow scales. Here, neither definition of convective velocity is physically meaningful, as the maximum is not associated with a clear peak and the center of gravity definition merely selects the center point of the computational domain. The comparison verified that the overall trends that are highlighted in this manuscript, such as the influence of the large-scale structures in the near-wall and buffer regions as well as the slope of the knee through the contours indicated by dash-dot lines in the panels of figure 4 were unchanged when we used the center of gravity in our computations. We therefore proceed with the definition in terms of the peak of the PSD in equation (3.7) in the remainder of the manuscript.

Both figures 4 and 5 indicate that large channel spanning structures have an influence on the convective velocity in both the viscous sublayer and the buffer layer. This phenomenon was investigated by del Álamo & Jiménez (2009), who identified large scales as structures of size $(\lambda_x, \lambda_z) > (2, 0.4)$. This large-scale cut-off is identified by horizontal and vertical dashed lines in each panel of figure 4.

Figure 6(a) plots the mean convective velocities obtained for the averaging domain $(\lambda_x, \lambda_z) > (2, 0.4)$ for $Re_\tau = 550$ and $Re_\tau = 1000$. A comparison to DNS data indicates that the model produces qualitatively and quantitatively similar behavior to results computed from DNS data (figure 5(a) in del Álamo & Jiménez (2009)). Note the results in figure 6(a) are scaled by the bulk velocity; i.e., $U_b = \frac{1}{2} \int_{-1}^1 \bar{u}(y) dy$ for direct comparison with del Álamo & Jiménez (2009).

In figure 6(b), we further analyze the influence of the large scales by comparing the scale-dependent convective velocity at $y^+ \approx 16$ with $Re_\tau = 1000$ from our approach to figure 3(a) in del Álamo & Jiménez (2009). The darker blue region of the largest λ_x^+ and moderate to largest λ_z^+ in figure 6(b) indicates the influence of large and very large-scale motion on the convective velocity in this region, which supports previous studies (Kim & Hussain 1993; Krogstad *et al.* 1998) indicating that fast moving structures centered further away from the wall (where the local mean velocity is larger) have an influence very near the wall due to their large size (Dinkelacker *et al.* 1977; Kreplin & Eckelmann 1979; Farabee & Casarella 1991; Kim & Hussain 1993; Hutchins *et al.* 2011).

A linear fit through the knee of the contour plot in figure 6(b) shows self-similar structures with a ratio $\lambda_z^+ \sim \lambda_x^{+\frac{2}{3}}$ in both our results and those reported in del Álamo & Jiménez (2009). This type of x - z similarity in energy spectra density has been previously observed in the context of geometric self-similarity (del Álamo *et al.* 2004; Chandran *et al.* 2017). For example, del Álamo and co-authors (del Álamo & Jiménez 2003; del Álamo *et al.* 2004) found that the isocontours of the pre-multiplied energy spectrum of u' form a corner centered along $\lambda_z \sim \lambda_x^n$ with $\frac{1}{3} < n < 1$. In that work, the value of n changed over the wall-normal location with lower bound on n near the buffer layer ($y^+ \approx 15$), increasing to $n = \frac{1}{2}$ in the log-law region and reaching $n = 1$ in the outer region of the flow. Recent

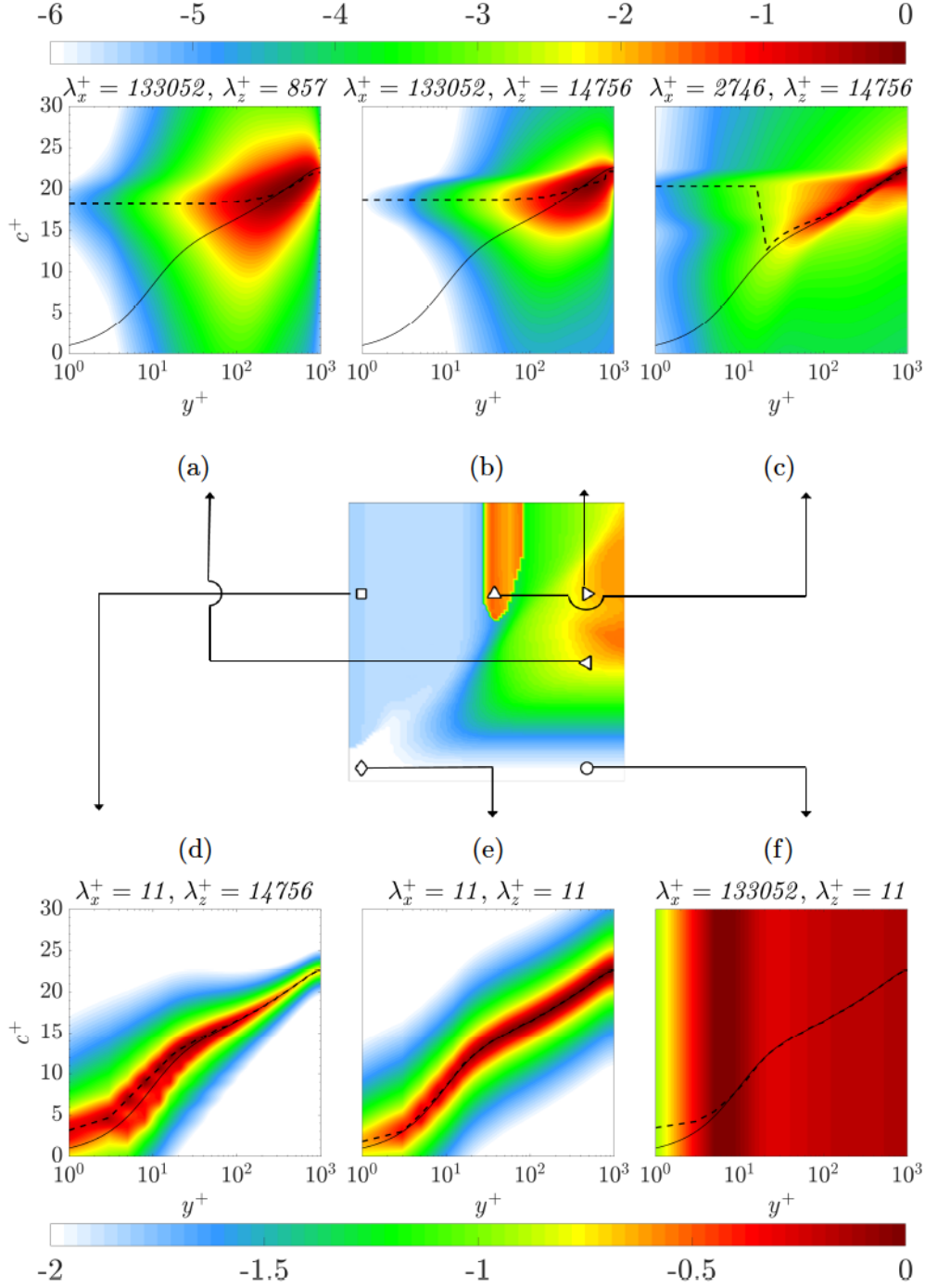


Figure 5: Normalized power spectral density of streamwise velocity fluctuations $\frac{\Phi_{\hat{u}'}(y; k_x, k_z, c)}{\max_{c, y}(\Phi_{\hat{u}'}(y; k_x, k_z, c))}$ over wall-normal location y^+ and phase speed c^+ . The symbols represent locations associated with large-scale structures at (a) $\triangleleft (\lambda_x^+, \lambda_z^+) = (133052, 857)$ and (b) $\triangleright (\lambda_x^+, \lambda_z^+) = (133052, 14756)$; (c) intermediate-scale structures $\triangle (\lambda_x^+, \lambda_z^+) = (2746, 14756)$, and structures with small streamwise or spanwise wavelengths, respectively: (d) $\square (\lambda_x^+, \lambda_z^+) = (11, 14756)$, (e) $\diamond (\lambda_x^+, \lambda_z^+) = (11, 11)$, and (f) $\circ (\lambda_x^+, \lambda_z^+) = (133052, 11)$. The color is in base 10 logarithmic scale. The black solid lines represent the mean streamwise velocity profile and the black dashed lines are convective velocities computed using the method in section 3. The middle panel, which is reproduced from figure 4(b) at $y^+ \approx 5$ for $Re_\tau = 1000$, indicates the locations corresponding to each

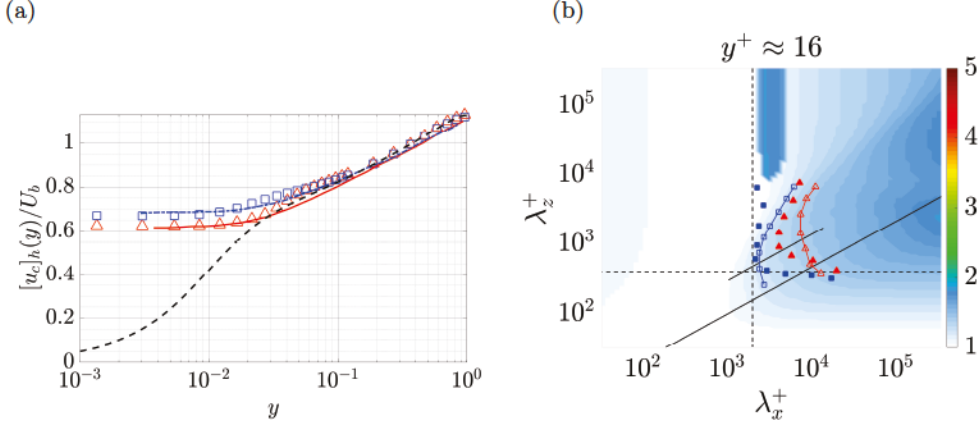


Figure 6: (a) The average convective velocity of streamwise fluctuations $[u_c]_h/U_b$ over $(\lambda_x, \lambda_z) > (2, 0.4)$. Model based results at $Re_\tau = 550$ ($\triangle$) and $Re_\tau = 1000$ ($\square$) are compared to convective velocities computed from DNS data (del Álamo & Jiménez 2009) at $Re_\tau = 547$ ($-$) and $Re_\tau = 934$ ($- -$). For direct comparison with del Álamo & Jiménez (2009), the results in (a) are scaled by the bulk velocity; i.e., $U_b = \frac{1}{2} \int_{-1}^1 \bar{u}(y) dy$. The black dashed line is the mean velocity profile at $Re_\tau \approx 1000$ from Lee & Moser (2015). (b) Model based scale-dependent convective velocity at $y^+ \approx 16$ for $Re_\tau = 1000$: contour lines $u_c(y; k_x, k_z)/\bar{u}(y) = 1.40$ ($-\triangle-$); $u_c(y; k_x, k_z)/\bar{u}(y) = 1.21$ ($-\square-$); are compared to convective velocities computed from DNS data at $y^+ = 15$ and $Re_\tau = 934$ (del Álamo & Jiménez 2009) $u_c(y; k_x, k_z)/\bar{u}(y) = 1.40$ ($\blacktriangle$); $u_c(y; k_x, k_z)/\bar{u}(y) = 1.21$ ($\blacksquare$). The black dashed lines are given by $(\lambda_x, \lambda_z) = (2, 0.4)$. The black solid lines are $\lambda_z^+ \sim \lambda_x^{+2/3}$, which fit through the knee of model based convective velocity and DNS data contours (del Álamo & Jiménez 2009), respectively.

experimental measurements of two-dimensional spectra in zero-pressure gradient boundary layers indicate a $\lambda_z \sim \sqrt{\lambda_x}$ ($Re_\tau = 2430$) and $\lambda_z \sim \lambda_x$ ($Re_\tau = 26090$) relationship in the start of the log-law region (Chandran *et al.* 2016, 2017).

The scaling law of convective velocity explored here is closely related to the temporal self-similarity previously observed in the literature. More specifically, Lozano-Durán & Jiménez (2014) showed that tall attached structures are both geometrically and temporally self-similar with lifetimes proportional to their distance from the wall. They also attribute the lifetime and deformation of these structures to the vertical gradient of their convective velocity. Long lifetimes, which require low dispersion, have been associated with coherent structures (Adrian 2007); e.g., hairpin vortices that are observed to propagate downstream with small velocity dispersion (Adrian *et al.* 2000). Non-dispersive coherent structures are implied by the isocontour lines of the scale dependent convective velocities in figures 4 and 6(b), which forms a $\lambda_z^+ \sim \lambda_x^{+2/3}$ knee. Based on this observation, we conjecture that the x - z similarity observed here is closely related to the scaling laws of energy spectra.

To explain the $\lambda_z^+ \sim (\lambda_x^+)^{2/3}$ scaling, we employ a simple model involving the bending of streamlines in the cross-plane due to the presence of a streamwise vortex. This simple model was originally proposed by Jiménez *et al.* (2004) to

explain the contribution of high-momentum streaks to the energy spectrum. Consider convection of $\hat{u}'$ due to a point vortex with circulation γ in the cross-plane $\partial_t u' + \frac{2\pi\gamma}{r^2} \frac{\partial u'}{\partial \theta} = 0$ where (r, θ) is the polar representation of the (y, z) plane, centered on the vortex. For a homogeneous shear initial condition, $u'(t=0, r, \theta) \sim Sy = Sr \sin \theta$, with shear rate S , we have $u'(t, r, \theta) \sim Sr \sin(\theta - \gamma t/2\pi r^2)$. At a given time, t , setting $\theta = \pi$ yields the ‘size’ of the vortex-distorted region as:

$$R_z = \sqrt{\gamma t/2\pi^2}. \quad (5.4)$$

Moreover, the length of streak is determined by the velocity difference between its top and bottom with shear rate S , which can be roughly estimated as:

$$\lambda_x = SR_z t, \quad (5.5)$$

if we assume that the streak height is roughly equal to its width. We estimate the meandering magnitude of streaks by approximating the spanwise drift of the vortices under the induction of their reflected images across the wall, which leads to

$$\lambda_z = 2\sqrt{2}w't, \quad (5.6)$$

where w' denotes a spanwise velocity fluctuation of the order of $w'^+ \approx 1$ (Kim *et al.* 1987). Combining equations (5.4), (5.5), and (5.6) gives the scaling:

$$\lambda_z^+ \sim (\lambda_x^+)^{\frac{2}{3}}. \quad (5.7)$$

Although this is an idealized analysis, it leads to the trends observed both here and in DNS based convective velocity analysis. The assumptions underlying this scaling are also consistent with the existence of structures at a wide variety of scales extending into the channel; i.e., structures reminiscent of Townsend’s attached eddies (Townsend 1976; Perry *et al.* 1986; Marusic & Monty 2019). We next calculate the wall-normal coherence of these structures to further examine this connection.

6. Wall-normal coherence of viscous sublayer structures

The convective velocity at each (λ_x, λ_z) wavelength pair, $u_c(y; k_x, k_z)$, is obtained via the maximization in (3.7), so we refer to the spectral component of u' defined by $(\lambda_x, \lambda_z, u_c)$ as the ‘characteristic structure’. We hypothesize that the characteristic structure at a given wall-normal location, y_0 , is responsible for the dominant convection at that location, and that it also contributes to the energetics elsewhere in the channel due to its wall-normal extent. In this section, we investigate this wall-normal extent using the spectral coherence between signals at two different wall-normal locations. We focus on the characteristic structures that provide the dominant convection in the viscous sublayer and on wavelength pairs along the knee of the isocontours of u_c in figure 4; i.e., along $\lambda_z^+ = (\lambda_x^+)^{\frac{2}{3}}$.

The u' frequency–wavenumber spectrum, $\Phi_{\hat{u}'} = \langle |\hat{u}'|^2 \rangle$ of streamwise fluctuations in the viscous sublayer ($y^+ \approx 5$) for $Re_\tau = 1000$, is shown in figure 7 in terms of phase speed and wavelengths along $\lambda_z^+ = (\lambda_x^+)^{\frac{2}{3}}$. The autocorrelation maxima defining the convective velocity are plotted as a dashed line. The ridge

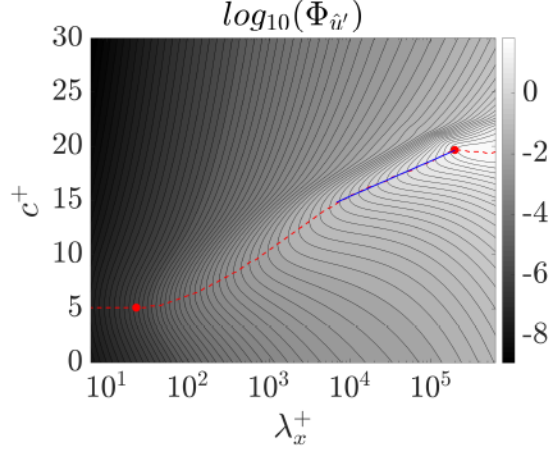


Figure 7: Isocontours of $\Phi_{\hat{u}'}$ in the viscous sublayer ($y^+ \approx 5$) along $\lambda_z^+ = (\lambda_x^+)^{\frac{2}{3}}$ at $Re_\tau = 1000$, calculated using (3.6). The red dashed line indicates the c^+ at which $\Phi_{\hat{u}'}$ peaks for each λ_x^+ , and the blue solid line indicates the region of logarithmic increase. It therefore defines the convective velocity u_c as in (3.7). The markers ($\bullet$) indicate the locations where we evaluate the two-point wall-normal coherence in figure 8.

corresponding to these maximum values asymptote to constants at both the large and small wavelength limits, but show a region of linear growth followed by a region of logarithmic increase (blue solid line) between two red circle markers. The logarithmic behavior is similar to the variation of the mean velocity profile $\bar{u}(y)$ with y and is consistent with the assumption that the dominant viscous sublayer convection at streamwise wavelength λ_x arises due to a structure advecting at the local mean velocity at $y \sim \lambda_x^n$ for some $n > 0$.

The spectral coherence between two wall-normal locations y' and y , defined for a fluctuating variable ψ is defined as:

$$0 \leq \chi_{\psi'; y' y}(k_x, k_z, c) \equiv \frac{|\Phi_{\hat{\psi}', \text{cross}}(y', y; k_x, k_z, c)|^2}{\Phi_{\hat{\psi}'}(y'; k_x, k_z, c) \Phi_{\hat{\psi}'}(y; k_x, k_z, c)} \leq 1, \quad (6.1)$$

where $\Phi_{\hat{\psi}', \text{cross}}$ is the cross-spectral density of ψ' between locations y' and y ; i.e.,

$$\Phi_{\hat{\psi}', \text{cross}}(y', y; k_x, k_z, c) = \langle \hat{\psi}'^*(y'; k_x, k_z, c) \hat{\psi}'(y; k_x, k_z, c) \rangle. \quad (6.2)$$

The cross-spectral densities are the off-diagonal components of the matrix obtained from the finite-dimensional representation (4.1) of $\Phi_{\hat{\psi}'}$. Figure 8 shows the two-point spectral coherence for streamwise velocity fluctuations $\chi_{u'; y' y}$ for two characteristic structures along $\lambda_z^+ = (\lambda_x^+)^{\frac{2}{3}}$ in the near-wall region: (a) a short wavelength component, $(\lambda_x^+, \lambda_z^+) \approx (22, 8)$, and (b) a long wavelength component, $(\lambda_x^+, \lambda_z^+) \approx (2 \times 10^5, 3 \times 10^3)$. The phase speeds associated with these characteristic structures are indicated in figure 7 by circle markers. The shorter wavelength component is associated with the smaller convective velocity and the longer wavelength component with the larger convective velocity.

In figure 8, we see a wall-normal coherence that extends *from* the wall; i.e., it

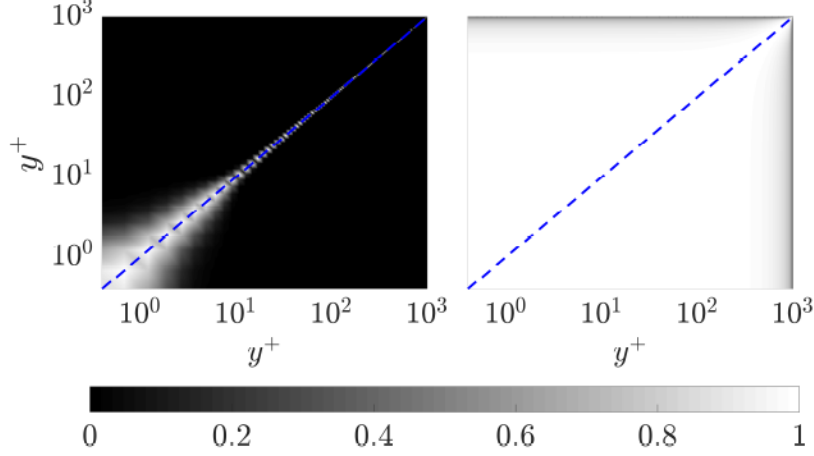


Figure 8: Two-point spectral coherence of streamwise velocity fluctuations $\chi_{u';y'y}^+$ for data at $Re_\tau = 1000$, as defined in (6.1) at $(\lambda_x^+, \lambda_z^+) \approx (22, 8)$ (left) and $(\lambda_x^+, \lambda_z^+) \approx (2 \times 10^5, 3 \times 10^3)$ (right) indicated by circle markers in figure 7. Both points are along $\lambda_z^+ = (\lambda_x^+)^{\frac{2}{3}}$, and their phase speeds in friction units are approximately $c^+ \approx 5$ for the small-scale structure and $c^+ \approx 20$ for the large one. Perfectly coherent signals have a spectral coherence of 1, and incoherent signals have a spectral coherence of 0.

does not involve any wall-detached patches, consistent with Townsend’s attached-eddy hypothesis (Townsend 1976; Perry *et al.* 1986; Marusic & Monty 2019). As predicted by del Álamo & Jiménez (2009), the long wavelength component is more coherent further into the channel towards the core than the short wavelength component with its coherence falling to 0 in the core. This growth of coherence away from the wall with increasing wavelength suggests that the structures contributing to the convective velocity in the viscous sublayer extend from the wall deep into the log-law region, but only weakly into the wake region, reminiscent of the long meandering structures in the log-law region whose footprint extends to the near-wall region (Jiménez *et al.* 2004; Hutchins & Marusic 2007; Monty *et al.* 2007; Guala *et al.* 2006; Balakumar & Adrian 2007).

Calculations (not presented here for brevity) indicate that components with identical convective velocity as determined by figure 4 also have nearly identical wall-normal coherence. This behavior, also suggested by del Álamo & Jiménez (2009), agrees with the hypothesis that a random arrangement of similar basic structures with dimensions given by $\lambda_z^+ = (\lambda_x^+)^{\frac{2}{3}}$, leads to the long-tailed behavior of the contours in figure 4.

Figure 9 shows the spectral coherence with respect to the viscous sublayer location ($y_{\text{sublayer}}^+ \equiv 5$) $\chi_{u';y_{\text{sublayer}}y}(k_x, k_z, c)$ along $\lambda_z^+ = (\lambda_x^+)^{\frac{2}{3}}$. The monotonic behavior of the spectral coherence in figure 9 implies that structures larger in (x, z) are also larger in y . The wall-normal coherence for $\lambda_x^+ \gtrsim 200$ indicated by the dashed lines overlain on the (red) solid contours representing spectral coherences of 0.1 and 0.5 shows an aspect ratio $y^+ \sim (\lambda_x^+)^{\frac{2}{3}}$. The minimum wall-normal coherence length associated with these larger wavelengths is ~ 15 wall

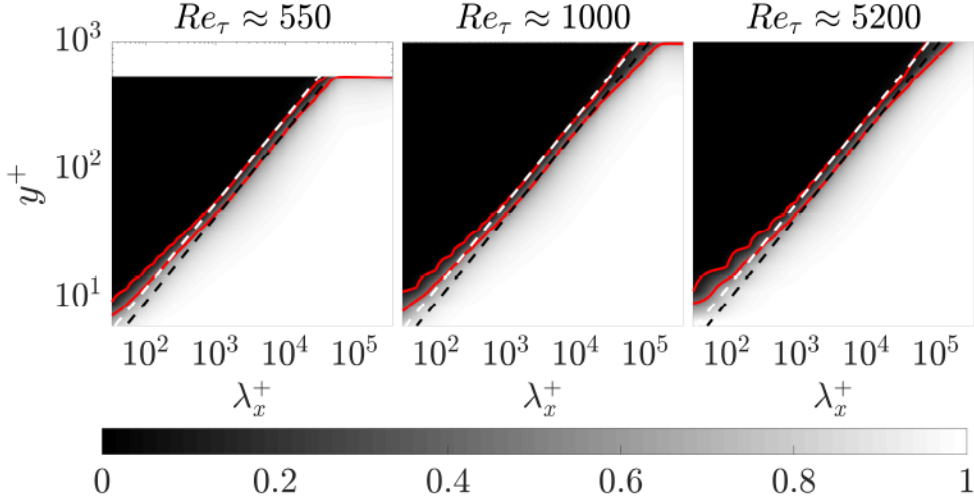


Figure 9: Two-point spectral coherence of streamwise velocity fluctuations $\chi_{u';y'y}$ at $Re_\tau \approx 550$, $Re_\tau \approx 1000$, and $Re_\tau \approx 5200$, as defined in (6.1), between $y'^+ \approx y_{\text{sublayer}}^+ = 5$ and wall-normal locations above it for wavelengths defined by $\lambda_z^+ = (\lambda_x^+)^{2/3}$ and phase speeds indicated by the dashed maxima line in figure 7. The solid red lines serve to indicate the boundaries of regions of high/low coherence and are isocontours of spectral coherence with values 0.1 and 0.5. The white dashed lines are $y^+ = 0.55(\lambda_x^+)^{2/3}$ and the black dashed lines are $y^+ = 0.43(\lambda_x^+)^{2/3}$ and they serve as fits to the red lines. Perfectly coherent signals have a spectral coherence of 1, and incoherent signals have a spectral coherence of 0.

units, which is the approximate location of the buffer layer and also the location of the well-known peak in the root-mean-square (RMS) streamwise velocity fluctuations; see for example, Lee & Moser (2015). This self-similarity represented by a power-law relationship at larger wavelengths is also suggestive of the attached-eddy structures proposed by Townsend (1976); Perry *et al.* (1986); Marusic & Monty (2019).

From the power-law behavior $y^+ \sim (\lambda_x^+)^{2/3}$ for $\lambda_x^+ \gtrsim 200$ shown in figure 9, we can also extract the structure inclination angle contributing to this self-similar behavior. The 0.1 and 0.5 spectral coherence contour is fitted by $y^+ = \alpha(\lambda_x^+)^{2/3}$ with $\alpha = 0.55$ (white dash lines) and $\alpha = 0.43$ (black dash lines), respectively. We select the spectral coherence contours as 0.1 and 0.5 to fit the scaling laws because we observe significant variation of coherence between this range in figure 9, while outside of this range, the coherence show saturation. Such a saturation phenomenon is also observed in the coherence computed from the experimental data; see e.g., figure 4 of Baars *et al.* (2016) and figure 5(b) of Baars *et al.* (2017). Furthermore, the contours of two-dimensional spectral coherence of 0.1, 0.3, and 0.5 are shown to collapse when scaled with the wall-normal height of the structures; indicating the presence of self-similar structures, see figure 4 of Madhusudan *et al.* (2019).

Thus, $y^+ \sim \lambda_z^+$ with a constant of proportionality α between 0.55 and 0.43,

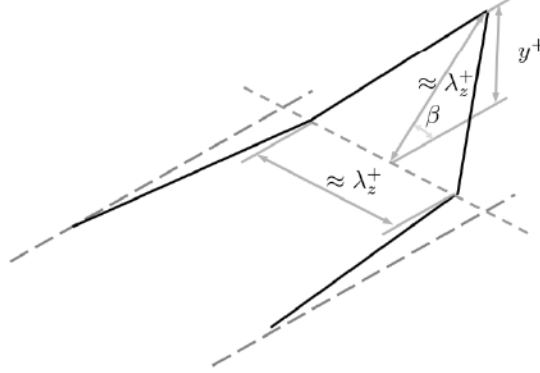


Figure 10: The structures of height/width aspect ratio 1 with an inclination angle β in analogy with Perry & Chong (1982).

respectively, imply that the projection of the structures onto the cross-stream plane has a smaller height than width. If we assume the structures contributing to the spectral coherence have a height/width aspect ratio of 1 as depicted in the cartoon in figure 10, then the dimensions of the cross-plane projection of the structure represented by spectral coherence between 0.1 and 0.5 implies a tilt angle $\beta = \arcsin(\alpha)$ between approximately 25° and 33° .

Townsend (1976) suggests an inclination angle of $\sim 30^\circ$ for attached double roller eddies (a pair of counter-rotating, inclined, approximately streamwise vortex structures) to explain the experimental observations of Grant (1958). Experimental observations in turbulent boundary layers, on the other hand, have yielded inclination angles between 15° and 20° (Brown & Thomas 1977; Marusic & Heuer 2007; Carper & Porté-Agel 2004). The latter inclination angles were calculated using two-point temporal correlations and Taylor's hypothesis. In turbulent channel flows, hairpin vortices have been the focus of considerable interest. Although a single well-defined inclination angle cannot be associated with hairpin vortices, the inclinations of hairpin like structures vary from 12° (elongated legs) to 45° (hairpin heads) (Adrian 2007). The present work does not restrict the structures contributing to the convective velocities to any one of the structures discussed above, but does provide an inclination angle, assuming structures are roughly of aspect ratio 1, which is within the range of previous observations.

In figure 9, the structure inclination angle predicted by this model also shows Reynolds number invariance, which is consistent with experimental observations (Marusic & Heuer 2007). For different Reynolds numbers, $y^+ = \alpha(\lambda_x^+)^{\frac{2}{3}}$ with $\alpha = 0.55$ and $\alpha = 0.43$, corresponding to tilt angles 25° and 33° , always give good approximations for 0.1 and 0.5 spectral coherence, respectively. Structure inclination angles inferred from the cross correlation of x in the experimental studies of Marusic & Heuer (2007) are found to be invariant over 3 orders of magnitude change in Reynolds number.

Our results reveal that the contributions from the relatively larger scale structures lead to the elevated velocities in the viscous sublayer seen in figure 3 and

that these structures have dimensions given by $y^+ \sim \lambda_z^+ \sim (\lambda_x^+)^{\frac{2}{3}}$ with a minimum size ≈ 15 friction units, which is the approximate location of the buffer layer. The inclination angles of these structures do not vary with Reynolds number, which is consistent with experimental observations. These observations are consistent with the attached-eddy hypothesis in that they predict wall-attached structures that are self-similar in the cross-plane and contribute to the dominant convection. However, the attached-eddy hypothesis predicts that these structures are also self-similar in the streamwise direction, which our approach does not show.

7. Term-by-term analysis of scale-dependent convective velocities

We next use the input-output framework to analyze the contribution of different linear mechanisms to the scale-dependent convective velocity of the streamwise velocity fluctuations.

We follow the method shown in equation (2.11) of del Álamo & Jiménez (2009) to obtain the normalized deviation of the convective velocity from the mean velocity contributed from various terms. In particular, we multiply the x -momentum in equation (2.3a) by $\hat{u}'^*$ and take the imaginary part of the result to obtain:

$$\frac{u_c - \bar{u}(y)}{\bar{u}(y)} = \frac{\overbrace{k_x \text{Re}\{\langle \hat{p}' \hat{u}'^* \rangle\}}^{\text{IIa}} + \text{Im}\left\{ \overbrace{\frac{d\bar{u}}{dy} \langle \hat{v}' \hat{u}'^* \rangle}^{\text{IIb}} - \overbrace{\frac{1}{Re_\tau} \langle \hat{u}'^* \partial_{yy}^2 \hat{u}' \rangle}^{\text{IIc}} - \overbrace{\langle \hat{f}_u' \hat{u}'^* \rangle}^{\text{III}} \right\}}{k_x \bar{u}(y) \langle \hat{u}' \hat{u}'^* \rangle}. \quad (7.1)$$

Here $\text{Re}\{\cdot\}$ and $\text{Im}\{\cdot\}$ represent the respective real part and imaginary part of the argument. The terms in equation (7.1) represent the relative contributions of the pressure term (IIa), the mean shear term (IIb), and the viscous term (IIc), each normalized by $k_x \bar{u}(y) \langle \hat{u}' \hat{u}'^* \rangle$.

We compute each term in (7.1) by modifying the output operator in (3.4) and then computing the cross-spectra through an appropriate modification of (3.6). For example, we can use the output operator corresponding to the fluctuating pressure in (3.9) to obtain $\mathcal{G}_{\hat{p}'}$ and then compute the cross-spectra as

$$\langle \hat{p}' \hat{u}'^* \rangle = \mathcal{G}_{\hat{p}'} \langle \hat{f}' \hat{f}'^* \rangle \mathcal{G}_{\hat{u}'}^* = \mathcal{G}_{\hat{p}'} \mathcal{G}_{\hat{u}'}^*. \quad (7.2)$$

The other terms in (7.1) can be computed in a similar manner.

Figure 11(a), (b), and (c) show the respective contributions from the pressure term (IIa), the mean shear term (IIb), and the viscous term (IIc) to the scale-dependent convective velocity of the streamwise velocity fluctuations ($Re_\tau \approx 1000$) at the same three wall-normal locations as in figure 4. As shown in figure 11(a), the pressure plays an important role for the intermediate scale structures ($\lambda_x \approx 2$ and $\lambda_z > \lambda_x$), which supports our conjecture that the discontinuity in these scales shown in figure 4 is related to the pressure. Luhar *et al.*'s (2014) figure 12(a) also showed a discontinuity of the scale-dependent convective velocity of wall-pressure computed using resolvent analysis and the maximum of the PSD to define the convective velocity. As discussed in Section 5, using the center of gravity of the PSD to define the convective velocity eliminates the discontinuity. A similar smoothing effect resulting from the use of the center of gravity definition versus

the maximum value of the PSD was also observed in figures 12(a) and (b) of Luhar *et al.* (2014), where the authors compared these two convective velocity definitions for pressure fluctuations. The overall convective velocity of these intermediate-scale structures also includes contributions from both the mean shear (figure 11(b)) and the viscous terms (figure 11(c)), which indicates that multiple physical mechanisms are at play.

For the large-scale structures with $(\lambda_x, \lambda_z) > (2, 0.4)$, the deviation of convective velocity from the mean is primarily due to the viscous and the mean shear terms. In the viscous sublayer ($y^+ \approx 5$), the viscous term provides a relatively larger contribution to the deviation of the convective velocity from the mean than the mean shear term, whereas these two terms provide approximately equal contribution to the convective velocity in the buffer layer ($y^+ \approx 15$).

For structures with small streamwise and spanwise wavelengths; i.e., $\lambda_x^+ \lesssim 10$ and $\lambda_z^+ \lesssim 10$, all of the terms in (7.1) are negligible (as indicated by the white region of the colormap). This suggests that they convect at the local mean velocity or that their convective velocity is not captured through the linear terms retained in our approach. However, as previously noted, the nonlinear fluctuation-fluctuation interactions likely dominate at these scales, so linear analysis is unlikely to fully explain the mechanisms at play. Understanding the effects of nonlinearity is beyond the scope of the current paper, so we leave this as a topic of future work.

To gain more insight into the effect of each term, we next compute the convective velocities by neglecting the contribution of different terms in the linear dynamics that form the spatio-temporal transfer function in (3.4). In each case, we first describe how neglecting the term(s) of interest alters these operators and then evaluate the effect on the convective velocity. Setting the mean shear term to zero reduces the linear operator in (3.2) to

$$\mathcal{L} := \begin{bmatrix} (ik_x(\bar{u} - c) - \frac{1}{Re_\tau} \hat{\Delta}) \mathbf{I}_{3 \times 3} & \hat{\nabla} \\ \hat{\nabla}^\top & 0 \end{bmatrix}. \quad (7.3)$$

In this case, the operators $\mathcal{B}$ and $\mathcal{C}$ in (3.8) remain the same. We note that although the mean shear term is zero, $\bar{u}(y)$ is still a function of wall-normal location; therefore there is still shear imposed by the mean flow. Figure 12(a) shows that the convective velocities of the large scales continue to deviate from the mean velocity even when we eliminate the linear term associated with the mean shear. However, the knee occurring at $\lambda_z^+ = \frac{5}{2} \sqrt{\lambda_x^+}$ shown in figure 12(a) is different from that at $\lambda_z^+ = \lambda_x^{+\frac{2}{3}}$ in figure 4 based on the full linear approach. This is consistent with figure 11(b), which indicates that the mean shear term plays a role in the self-similarity predicted in this approach.

We next isolate the role of the pressure. For this analysis we group the effect of the pressure gradient and the mass conservation terms because they both contribute to the nonlocality of the turbulent flow. This relationship can be understood by viewing the pressure in the momentum equation as the Lagrange multiplier that enforces the divergence-free velocity field; see e.g., section 5.6.2 in Schmid & Henningson (2001). Neglecting both the pressure term and

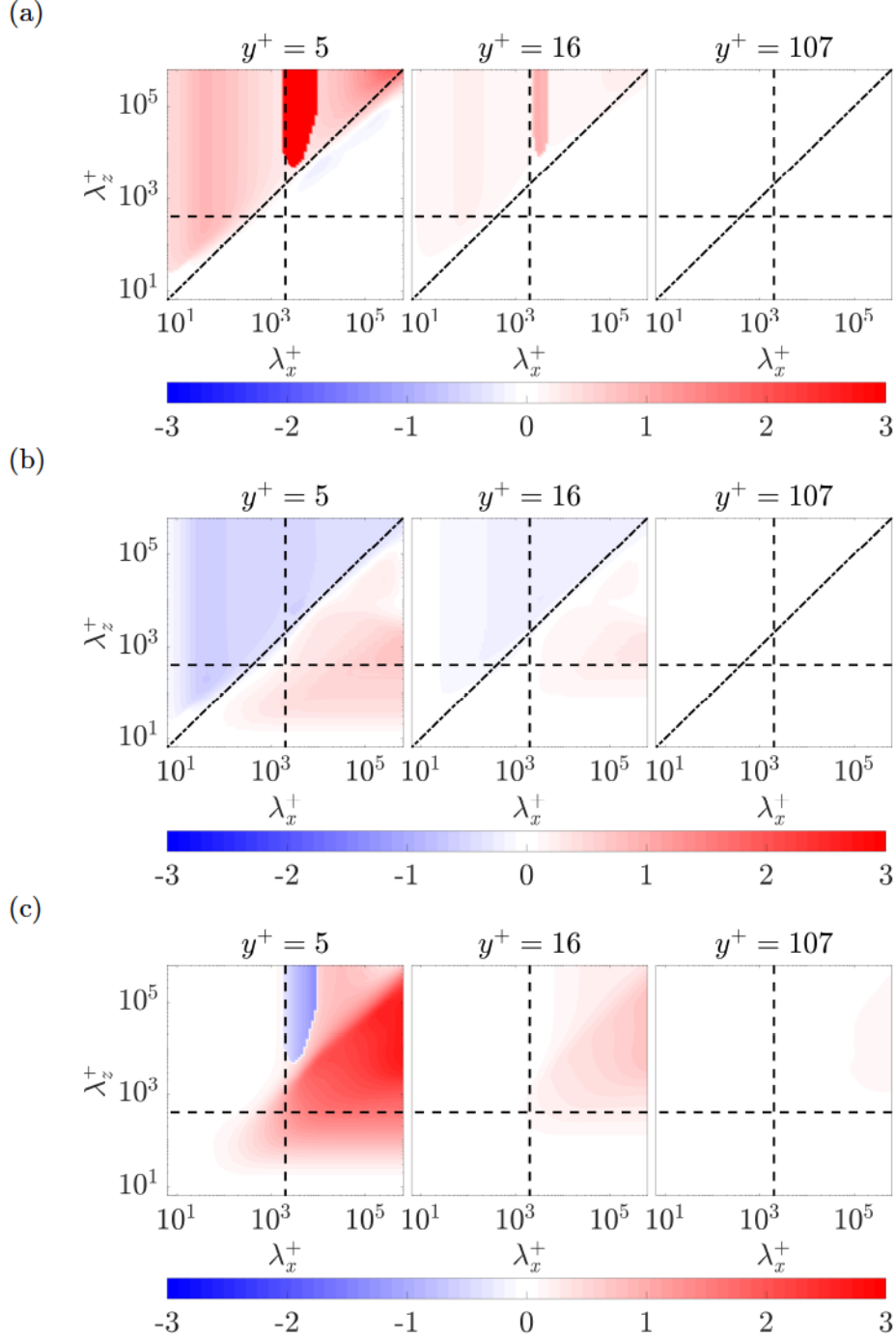
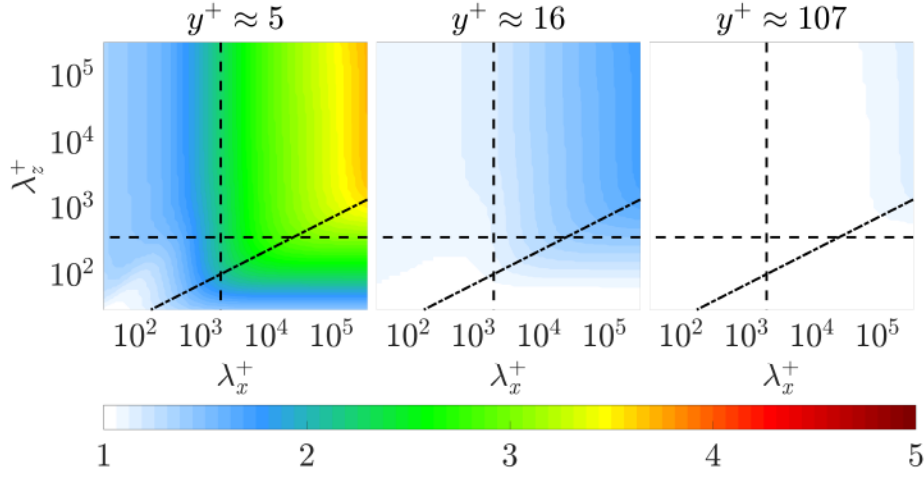
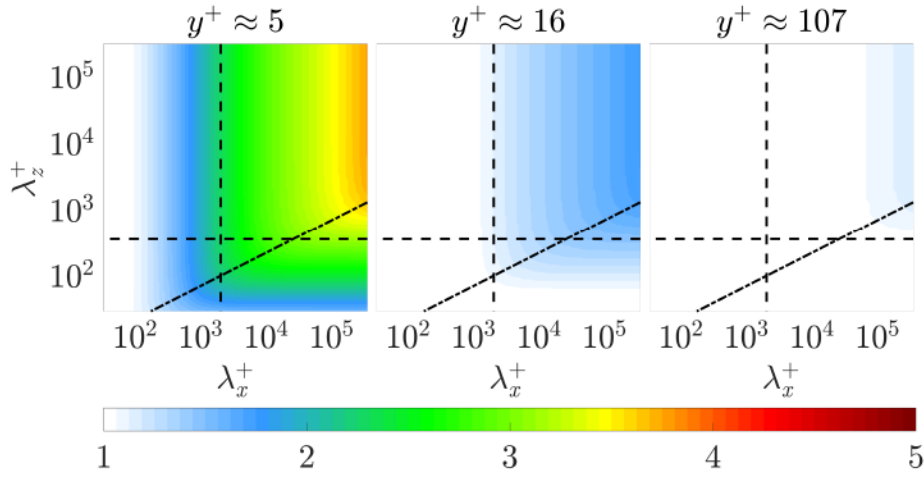


Figure 11: Different linear terms' contributions to scale-dependent convective velocities $(u_c(y; k_x, k_z) - \bar{u}(y))/\bar{u}(y)$ quantified using equation (7.1): (a) the pressure term (IIa), (b) the mean shear term (IIb), and (c) the viscous term (IIc). All terms are normalized by $k_x \bar{u}(y) \langle \hat{u}' \hat{u}'^* \rangle$. The Reynolds number is $Re_\tau = 1000$. The black dashed lines are given by $(\lambda_x, \lambda_z) = (2, 0.4)$, and the black dash-dot lines in (a) and (b) are $\lambda_z^+ = \lambda_x^+$. Note: the white region of the color map represents a value close to zero.

(a)



(b)



(c)

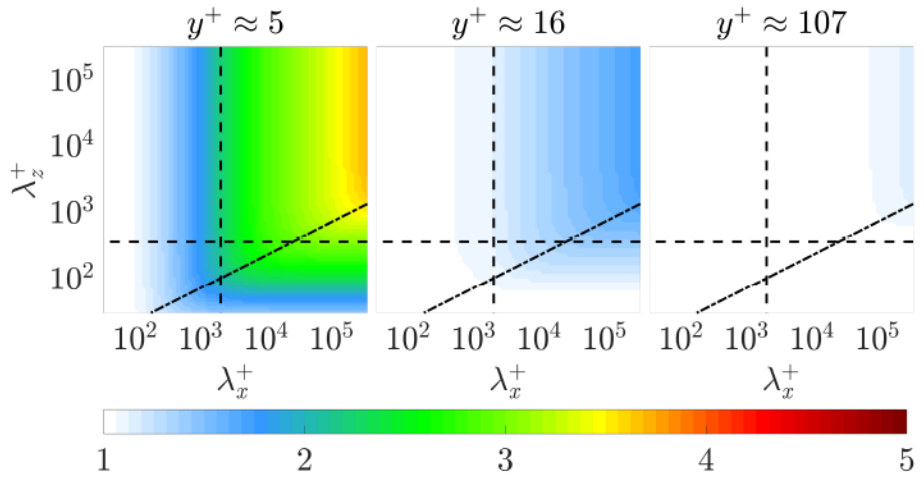


Figure 12: $u_c(y, \lambda_x, \lambda_z)/\bar{u}(y)$ at $Re_\tau = 1000$ (a) neglecting the mean shear term as (7.3), (b) neglecting the coupling from the pressure and mass conservation as (7.4), and (c) neglecting the mean shear term, the pressure term and mass conservation together as (7.5). The black dashed lines are given by $(\lambda_x, \lambda_z) = (2, 0.4)$. The black dash-dot lines are $\lambda_z^+ = \frac{5}{2}\sqrt{\lambda_x^+}$, which fit through the knee of these contours.

the divergence free constraint reduces the operators in the input-output map $\mathcal{G}_{\hat{u}'} = \mathcal{C}_{\hat{u}'} \mathcal{L}^{-1} \mathcal{B}$ to:

$$\mathcal{C}_{\hat{u}'} := \begin{bmatrix} 1 & \mathbf{0}_{1 \times 2} \end{bmatrix}, \mathcal{L} := \left[(ik_x(\bar{u} - c) - \frac{1}{Re_\tau} \hat{\Delta}) \mathbf{I}_{3 \times 3} + \frac{d\bar{u}}{dy} \mathbf{S} \right], \mathcal{B} := \mathbf{I}_{3 \times 3}. \quad (7.4)$$

The resulting convective velocities in figure 12(b) are similar to those in figure 12(a) with the mean shear term set to zero. Neither of these terms appear to be responsible for the influence of the large-scale structures that leads to the observed behavior of the convective velocity in the near-wall region. They also do not reproduce the $\lambda_z^+ = \lambda_x^{+\frac{2}{3}}$ scaling, but they do emit self-similar structures with a different scaling exponent, $\lambda_z^+ = \frac{5}{2} \lambda_x^{+\frac{1}{2}}$.

In order to evaluate their combined effect, we next neglect the contributions of both the mean shear and pressure terms, leaving only the advective and viscous terms. The resulting input-output based approach for the streamwise velocity fluctuations is given by $\hat{u}' = \mathcal{C}_{\hat{u}'} \mathcal{L}^{-1} \mathcal{B} \hat{f}_x$ with

$$\mathcal{C}_{\hat{u}'} := [1], \quad \mathcal{L} := \left[ik_x(\bar{u} - c) - \frac{1}{Re_\tau} \hat{\Delta} \right], \quad \mathcal{B} := [1]. \quad (7.5)$$

Figure 12(c) shows the resulting convective velocity contours, which are similar to the results in panels (a) and (b). In particular, they reproduce the influence of the large-scale structures in the near-wall and buffer regions seen in the full LNS based approach. Figure 13 plots the power spectral density of the streamwise velocity fluctuations at different phase speeds c^+ and wall-normal locations y^+ computed using the model in equation (7.5). Although there are some differences from the results obtained using the full LNS system shown in figure 5, the phase speed that maximizes the energy spectrum; i.e., the convective velocity, still asymptotes to a constant value near the wall for large wavelength structures.

The main difference between these results and the full LNS based approach is that they show the same $\lambda_z^+ = \frac{5}{2} \lambda_x^{+\frac{1}{2}}$ scaling as the previous model in (7.5) with the influence of the pressure and mean shear removed. The inability to reproduce the correct aspect ratio for the self-similar structures suggest that their morphology is due to interactions between viscous mechanisms and other inviscid mechanism arising due to the interaction of the fluctuations with the mean shear $d\bar{u}/dy$ and the pressure, such as the lift-up effect (Brandt 2014) and the Orr mechanism (Farrell 1987; Jiménez 2013). However, the prediction of the main trends and scale interactions suggest that this type of model may provide a good balance between accuracy and simplicity. We next explore its potential as a viscous correction to Taylor's hypothesis.

We obtain this correction by rewriting equation (7.5) as

$$ik_x(\bar{u}(y) - c) \hat{u}' - \frac{\hat{\Delta}}{Re_\tau} \hat{u}' = \hat{f}_x. \quad (7.6)$$

Figure 14 compares the average convective velocity of streamwise velocity fluctuations computed using the viscous correction (7.6) with its corresponding weighting functions $h = \langle |\mathcal{F}_{xz}(u')|^2 \rangle k_x^2$ and an averaging domain of $(\lambda_x^+, \lambda_z^+) > (500, 80)$ at $Re_\tau = 1000$ to the results from the full LNS based approach and convective velocities obtained from DNS data at $Re_\tau = 932$ from Geng *et al.* (2015).

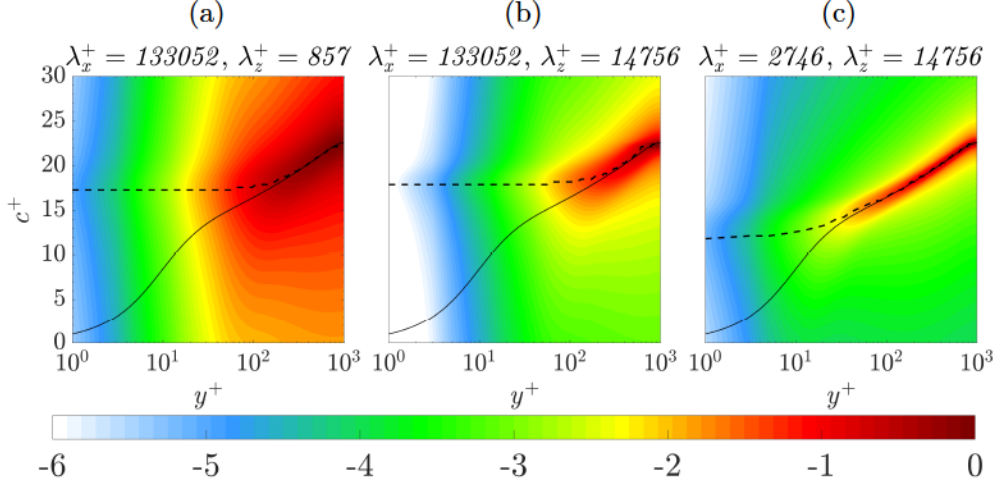


Figure 13: Power spectral density of streamwise velocity fluctuations over wall-normal location y^+ and phase speed $c \frac{\Phi_{\hat{u}'}(y; k_x, k_z, c)}{\max_{c, y}(\Phi_{\hat{u}'}(y; k_x, k_z, c))}$ at $Re_\tau = 1000$ from model (7.5) for representative large-scale structures $\triangleleft (\lambda_x^+, \lambda_z^+) = (133052, 857)$, $\triangleright (\lambda_x^+, \lambda_z^+) \approx (133052, 14756)$, and intermediate-scale structures $\triangle (\lambda_x^+, \lambda_z^+) \approx (2746, 14756)$. The color is in base 10 logarithmic scale. The black solid lines represent the mean streamwise velocity profile, and the black dashed lines are convective velocities, which are defined in (3.7) as the phase speed that maximizes the PSD of the streamwise fluctuations $\Phi_{\hat{u}'}(y; k_x, k_z, c)$.

This figure shows that the average convective velocity predicted from the viscous correction shows excellent agreement with results obtained from DNS data for $y^+ \in [5, 15]$, but begins to deviate for $y^+ \lesssim 3$. We therefore conclude that this viscous correction provides a potential dynamical modification on Taylor's hypothesis to improve the convective velocity estimates for use with experimental data.

This viscous correction introduced in equation (7.6) could be augmented using an eddy viscosity, in the spirit of the eddy viscosity enhanced LNS equations introduced in Reynolds & Hussain (1972). Such a dynamical correction was previously shown to provide similar improvements in model-fidelity for certain structures as the inclusion of colored-in-time forcing (Zare *et al.* 2017). This type model enhancement may be particularly relevant in this context because the pertinent terms would all be retained in the associated modification of the viscous correction proposed in equation (7.6). Assessing the potential benefits of such an approach is a topic of future work.

The convective velocities computed with this viscous correction to Taylor's hypothesis for a range of Reynolds numbers are compared in figure 15. The results indicate that the regions in $(\lambda_x^+, \lambda_z^+)$ where the convective velocities deviate from the local mean velocity are very similar across these Reynolds numbers, which is consistent with the observations in figure 4 indicating that the viscous correction preserves the previously observed Reynolds number invariance.

The viscous correction proposed here may also be applicable to input-output

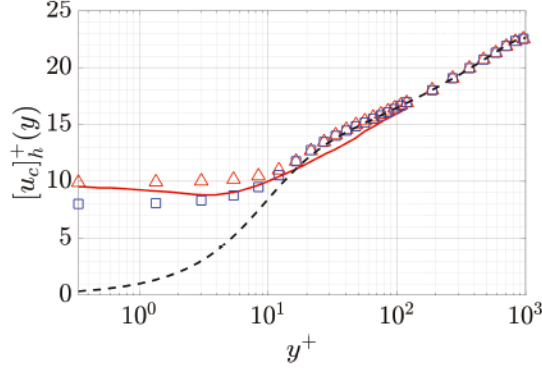


Figure 14: The average convective velocity of streamwise velocity fluctuations, $[u_c]_h^+(y)$: ($\square$); computed using the viscous correction (7.6) with their corresponding weighting functions $h = \langle |\mathcal{F}_{xz}(u')|^2 \rangle k_x^2$ and an averaging domain of $(\lambda_x^+, \lambda_z^+) > (500, 80)$ at $Re_\tau = 1000$. Results are plotted with convective velocities of streamwise velocity fluctuations computed from both the LNS based approach described herein for $Re_\tau = 1000$: ($\triangle$) and DNS data (Geng *et al.* 2015) at $Re_\tau = 932$: (—). The black dashed line is the turbulent mean velocity profile at $Re_\tau \approx 1000$ from Lee & Moser (2015).

based computations of convective velocities for other fluctuating quantities due to the similarity in the behavior of the near-wall convective velocities of velocity and vorticity components previously reported in the literature; see e.g., figures 3 and 5 of Geng *et al.* (2015); figures 1 and 2 of Kim & Hussain (1993) and the results in Liu & Gayme (2019). Exploring this notion is a topic of ongoing work.

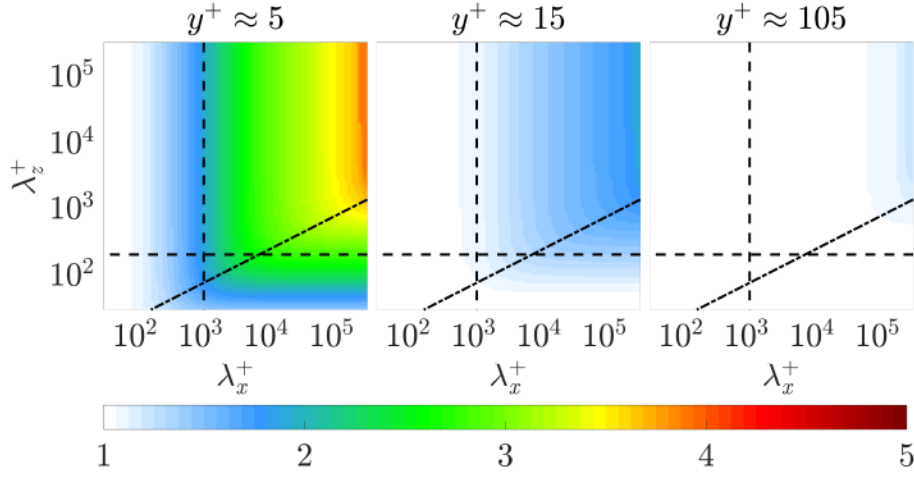
8. Conclusion

In this work, we analyze convective velocities of fluctuating quantities based on the stochastically-forced linearized Navier–Stokes equations with a given turbulent mean velocity profile. This approach allows for a detailed investigation of the scale-dependent convective velocities at all wall-normal locations, which enables a comprehensive examination of the mechanisms at play in the generation of convective velocities.

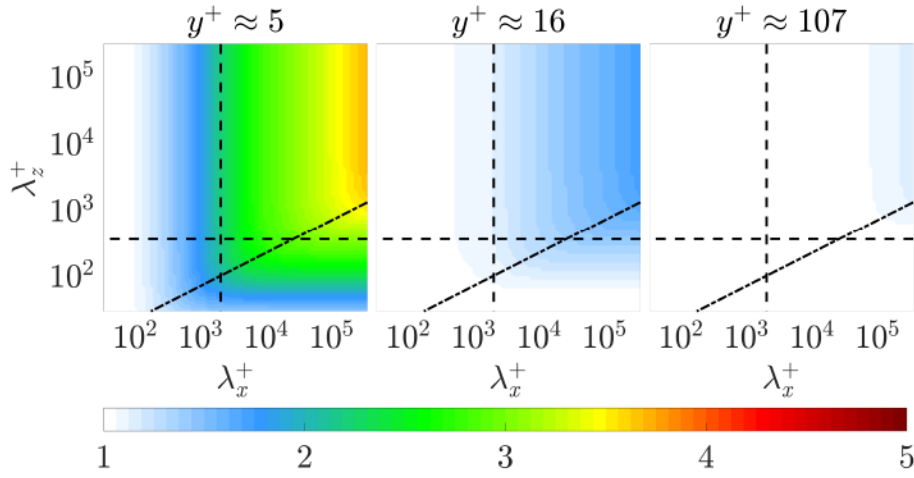
The convective velocities of velocity fluctuations obtained using the input-output based model reproduce trends previously observed in the literature, such as the deviation of the average convective velocity from the mean velocity and its tendency toward a constant value in the near-wall region. The model-based results indicate that the convective velocity of the streamwise velocity fluctuations closer to the wall show a stronger dependence on wavelength. The model predicted convective velocities show Reynolds number invariance when normalized in inner units, which is connected to the inner unit scaling of the resolvent operator (Moarref *et al.* 2013) and consistent with observations from DNS data (Geng *et al.* 2015) and experimental measurements (Marusic & Heuer 2007).

Our analysis also indicates that a wide range of structures contribute to the convective velocity especially in the viscous sublayer, where the convective velocity has been shown to be strongly scale-dependent.

(a)



(b)



(c)

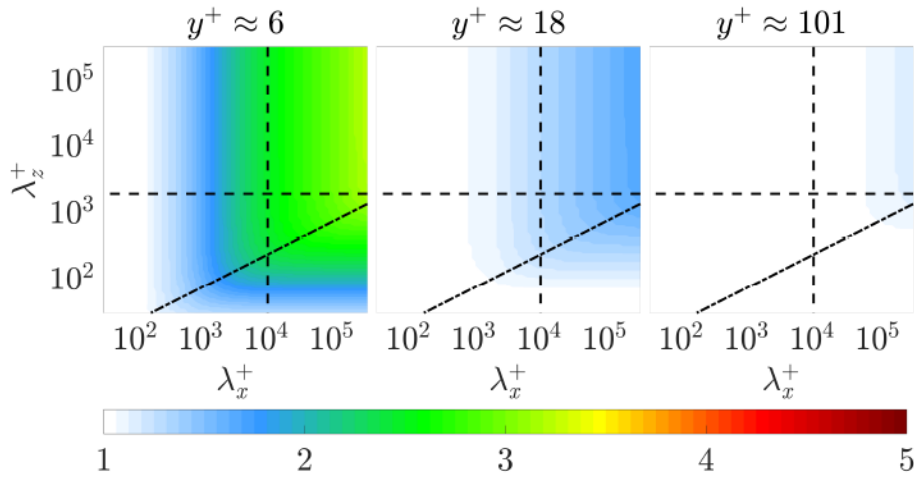


Figure 15: $u_c(y, \lambda_x, \lambda_z)/\bar{u}(y)$ predicted using the viscous correction to Taylor's hypothesis in equation (7.6) at (a) $Re_\tau = 550$, (b) $Re_\tau = 1000$, and (c) $Re_\tau = 5200$. The black dashed lines are given by $(\lambda_x, \lambda_z) = (2, 0.4)$. The black dash-dot lines are $\lambda_z^+ = \frac{5}{2}\sqrt{\lambda_x^+}$, which fits through the knee of these contours.

The primary structures contributing to the near-wall convective velocity based on the model are larger than the height of the buffer layer and are inclined at an angle between 25° and 33° . These predictions confirm the findings of Kim & Hussain (1993), who suggested that buffer layer structures are responsible for elevated convective velocities near the wall. However, our analysis suggests that a range of larger structures also contribute to this near-wall convective velocity. We demonstrate that these structures are self-similar in the cross-plane, similar to Townsend’s attached-eddies, yet scale as the $\frac{2}{3}$ power of a cross-plane dimension in the streamwise direction. Our model suggests that there is a connection between the convective velocity and structures whose signatures in measurements of power spectra scale as $\lambda_z^+ \sim \lambda_x^{+\frac{2}{3}}$.

We isolate and quantify the contributions from the pressure, mean shear, and viscous terms to the deviation of convective velocity from the local mean velocity. Based on this term- by-term analysis, a viscous correction to Taylor’s hypothesis is proposed. The proposed correction leads to a simplified model that accurately reproduces the behavior of near-wall convective velocity of the streamwise velocity fluctuations of large-scale structures.

The results presented here could be extended in a number of ways. For example, the representation of the forcing could be more closely tied to the nonlinearity observed in experimental or numerical simulation results by e.g., using simulation data to generate correlations for colored forcing (Moarref *et al.* 2014; Zare *et al.* 2017). Introducing an eddy viscosity based LNS representation (Reynolds & Hussain 1972) is another direction of ongoing work. The present approach has been specifically developed for wall-bounded flows with two homogenous spatial directions, and its efficacy has been demonstrated in the particular case of turbulent channel flow. The applicability of such a model, and other stochastically-forced models based on the linearized Navier–Stokes equations to a broader class of turbulent flows, including turbulent boundary layers, is the subject of ongoing work.

Acknowledgments

The authors thank Ismail Hameduddin for his contributions to the formulation and preliminary version of these results. His insight through discussions are also greatly appreciated. The authors gratefully acknowledge support from US National Science Foundation (NSF) through grant number CBET 1652244, program manager Ronald Joslin. C.L. also greatly appreciates support from the Chinese Scholarship Council.

REFERENCES

- ADRIAN, R. J. 2007 Hairpin vortex organization in wall turbulence. *Phys. Fluids* **19** (4), 041301.
- ADRIAN, R. J., MEINHART, C. D. & TOMKINS, C. D. 2000 Vortex organization in the outer region of the turbulent boundary layer. *J. Fluid Mech.* **422**, 1–54.
- BAARS, W. J., HUTCHINS, N. & MARUSIC, I. 2016 Spectral stochastic estimation of high-Reynolds-number wall-bounded turbulence for a refined inner-outer interaction model. *Phys. Rev. Fluids* **1** (5), 054406.

- BAARS, W. J., HUTCHINS, N. & MARUSIC, I. 2017 Self-similarity of wall-attached turbulence in boundary layers. *J. Fluid Mech.* **823**.
- BALAKUMAR, B. & ADRIAN, R. 2007 Large- and very-large-scale motions in channel and boundary-layer flows. *Philos. Trans. R. Soc. London A Math. Phys. Eng. Sci.* **365** (1852), 665–681.
- BAMIEH, B. & DAHLEH, M. 2001 Energy amplification in channel flows with stochastic excitation. *Phys. Fluids* **13** (11), 3258–3269.
- BRADSHAW, P. 1967 ‘Inactive’ motion and pressure fluctuations in turbulent boundary layers. *J. Fluid Mech.* **30** (2), 241–258.
- BRANDT, L. 2014 The lift-up effect: The linear mechanism behind transition and turbulence in shear flows. *Eur. J. Mech. B-Fluid* **47**, 80–96.
- BROWN, G. L. & THOMAS, A. S. W. 1977 Large structure in a turbulent boundary layer. *Phys. Fluids* **20** (10), S243–S252.
- BULLOCK, K. J., COOPER, R. E. & ABERNATHY, F. H. 1978 Structural similarity in radial correlations and spectra of longitudinal velocity fluctuations in pipe flow. *J. Fluid Mech.* **88** (3), 585–608.
- CARPER, M. & PORTÉ-AGEL, F. 2004 The role of coherent structures in subfilter-scale dissipation of turbulence measured in the atmospheric surface layer. *J. Turbul.* **5**, 1–24.
- CHANDRAN, D., BAIDYA, R., MONTY, J. P. & MARUSIC, I. 2016 Measurement of two-dimensional energy spectra in a turbulent boundary layer. In *Proceedings of the 20th Australasian Fluid Mechanics Conference*. Perth, Australia.
- CHANDRAN, D., BAIDYA, R., MONTY, J. P. & MARUSIC, I. 2017 Two-dimensional energy spectra in high-Reynolds-number turbulent boundary layers. *J. Fluid Mech.* **826**.
- CHUNG, D. & MCKEON, B. J. 2010 Large-eddy simulation of large-scale structures in long channel flow. *J. Fluid Mech.* **661**, 341–364.
- COSSU, C., PUJALS, G. & DEPARDON, S. 2009 Optimal transient growth and very large-scale structures in turbulent boundary layers. *J. Fluid Mech.* **619**, 79–94.
- DEL ÁLAMO, J. C. & JIMÉNEZ, J. 2003 Spectra of the very large anisotropic scales in turbulent channels. *Phys. Fluids* **15** (6), L41.
- DEL ÁLAMO, J. C. & JIMÉNEZ, J. 2006 Linear energy amplification in turbulent channels. *J. Fluid Mech.* **559**, 205–213.
- DEL ÁLAMO, J. C. & JIMÉNEZ, J. 2009 Estimation of turbulent convection velocities and corrections to Taylor’s approximation. *J. Fluid Mech.* **640**, 5–26.
- DEL ÁLAMO, J. C., JIMÉNEZ, J., ZANDONADE, P. & MOSER, R. D. 2004 Scaling of the energy spectra of turbulent channels. *J. Fluid Mech.* **500**, 135–144.
- DENNIS, D. J. C. & NICKELS, T. B. 2008 On the limitations of Taylor’s hypothesis in constructing long structures in a turbulent boundary layer. *J. Fluid Mech.* **614**, 197–206.
- DINKELACKER, A., HESSEL, M., MEIER, G. E. A. & SCHEWE, G. 1977 Investigation of pressure fluctuations beneath a turbulent boundary layer by means of an optical method. *Phys. Fluids* **20** (10), S216–S224.
- FARABEE, T. M. & CASARELLA, M. J. 1991 Spectral features of wall pressure fluctuations beneath turbulent boundary layers. *Phys. Fluids A-Fluid* **3** (10), 2410–2420.
- FARRELL, B. F. 1987 Developing disturbances in shear. *J. Atmos. Sci.* **44** (16), 2191–2199.
- FARRELL, B. F. & IOANNOU, P. J. 1993 Stochastic forcing of the linearized Navier-Stokes equations. *Phys. Fluids A-Fluid* **5** (11), 2600–2609.
- FAVRE, A., GAVIGLIO, J. & DUMAS, R. 1967 Structure of velocity space-time correlations in a boundary layer. *Phys. Fluids* **10** (9), S138–S145.
- FISHER, M. J. & DAVIES, P. O. A. L. 1964 Correlation measurements in a non-frozen pattern of turbulence. *J. Fluid Mech.* **18** (1), 97–116.
- FRISCH, U. & KOLMOGOROV, A. N. 1995 *Turbulence: the legacy of A.N. Kolmogorov*. Cambridge University Press.
- GANAPATHISUBRAMANI, B., HUTCHINS, N., MONTY, J. P., CHUNG, D. & MARUSIC, I. 2012 Amplitude and frequency modulation in wall turbulence. *J. Fluid Mech.* **712**, 61–91.
- GENG, C., HE, G., WANG, Y., XU, C., LOZANO-DURÁN, A. & WALLACE, J. M. 2015 Taylor’s hypothesis in turbulent channel flow considered using a transport equation analysis. *Phys. Fluids* **27** (2), 025111.

- GRANT, H. L. 1958 The large eddies of turbulent motion. *J. Fluid Mech.* **4** (2), 149–190.
- GUALA, M., HOMMEMA, S. E. & ADRIAN, R. J. 2006 Large-scale and very-large-scale motions in turbulent pipe flow. *J. Fluid Mech.* **554**, 521–542.
- HUTCHINS, N. & MARUSIC, I. 2007 Evidence of very long meandering features in the logarithmic region of turbulent boundary layers. *J. Fluid Mech.* **579**, 1–28.
- HUTCHINS, N., MONTY, J. P., GANAPATHISUBRAMANI, B., NG, H. C. H. & MARUSIC, I. 2011 Three-dimensional conditional structure of a high-Reynolds-number turbulent boundary layer. *J. Fluid Mech.* **673**, 255–285.
- HWANG, Y. & COSSU, C. 2010a Amplification of coherent streaks in the turbulent Couette flow: an input–output analysis at low Reynolds number. *J. Fluid Mech.* **643**, 333–348.
- HWANG, Y. & COSSU, C. 2010b Linear non-normal energy amplification of harmonic and stochastic forcing in the turbulent channel flow. *J. Fluid Mech.* **664** (664), 51–73.
- ILLINGWORTH, S. J., MONTY, J. P. & MARUSIC, I. 2018 Estimating large-scale structures in wall turbulence using linear models. *J. Fluid Mech.* **842**, 146–162.
- JIMÉNEZ, J. 2013 How linear is wall-bounded turbulence? *Phys. Fluids* **25** (11), 110814.
- JIMÉNEZ, J. 2018 Coherent structures in wall-bounded turbulence. *J. Fluid Mech.* **842**, P1.
- JIMÉNEZ, J., DEL ÁLAMO, J. C. & FLORES, O. 2004 The large-scale dynamics of near-wall turbulence. *J. Fluid Mech.* **505**, 179–199.
- JOVANOVIĆ, M. R. & BAMIEH, B. 2005 Componentwise energy amplification in channel flows. *J. Fluid Mech.* **534**, 145–183.
- DE KAT, R. & GANAPATHISUBRAMANI, B. 2015 Frequency-wavenumber mapping in turbulent shear flows. *J. Fluid Mech.* **783**, 166–190.
- KIM, J. & HUSSAIN, F. 1993 Propagation velocity of perturbations in turbulent channel flow. *Phys. Fluids A-Fluid* **5** (3), 695–706.
- KIM, J., MOIN, P. & MOSER, R. 1987 Turbulence statistics in fully developed channel flow at low Reynolds number. *J. Fluid Mech.* **177**, 133–166.
- KREPLIN, H. & ECKELMANN, H. 1979 Propagation of perturbations in the viscous sublayer and adjacent wall region. *J. Fluid Mech.* **95** (2), 305–322.
- KROGSTAD, P., KASPERSEN, J. H. & RIMESTAD, S. 1998 Convection velocities in a turbulent boundary layer. *Phys. Fluids* **10** (4), 949–957.
- LEE, M. & MOSER, R. D. 2015 Direct numerical simulation of turbulent channel flow up to $Re_\tau = 5200$. *J. Fluid Mech.* **774**, 395–415.
- LEHEW, J., GUALA, M. & MCKEON, B. J. 2010 A study of convection velocities in a zero pressure gradient turbulent boundary layer. In *Proceedings of the 40th Fluid Dynamics Conference and Exhibit*, p. 4474. Chicago, Illinois.
- LEHEW, J., GUALA, M. & MCKEON, B. J. 2011 A study of the three-dimensional spectral energy distribution in a zero pressure gradient turbulent boundary layer. *Exp. Fluids* **51** (4), 997–1012.
- LIN, C. C. 1953 On Taylor’s hypothesis and the acceleration terms in the Navier-Stokes equations. *Q. Appl. Math.* **10** (4), 295–306.
- LIU, C. & GAYME, D. F. 2019 Convective velocities of vorticity fluctuations in turbulent channel flows: an input-output approach. In *Proceedings of the Eleventh International Symposium on Turbulence and Shear Flow Phenomenon*. Southampton, UK.
- LOZANO-DURÁN, A. & JIMÉNEZ, J. 2014 Time-resolved evolution of coherent structures in turbulent channels: characterization of eddies and cascades. *J. Fluid Mech.* **759**, 432–471.
- LUHAR, M., SHARMA, A. S. & MCKEON, B. J. 2014 On the structure and origin of pressure fluctuations in wall turbulence: predictions based on the resolvent analysis. *J. Fluid Mech.* **751**, 38–70.
- LUMLEY, J. L. 1965 Interpretation of time spectra measured in high-intensity shear flows. *Phys. Fluids* **8** (6), 1056–1062.
- MADHUSUDANAN, A., ILLINGWORTH, S. J. & MARUSIC, I. 2019 Coherent large-scale structures from the linearized Navier-Stokes equations. *J. Fluid Mech.* **873**, 89–109.
- MARUSIC, I. & HEUER, W. D. C. 2007 Reynolds number invariance of the structure inclination angle in wall turbulence. *Phys. Rev. Lett.* **99** (11), 114504.
- MARUSIC, I., MCKEON, B. J., MONKEWITZ, P. A., NAGIB, H. M., SMITS, A. J. & SREENIVASAN, K. R. 2010 Wall-bounded turbulent flows at high Reynolds numbers: Recent advances and key issues. *Phys. Fluids* **22** (6), 065103.

- MARUSIC, I. & MONTY, J. P. 2019 Attached eddy model of wall turbulence. *Annu. Rev. Fluid Mech.* **51**, 49–74.
- MATHIS, R., HUTCHINS, N. & MARUSIC, I. 2009a Large-scale amplitude modulation of the small-scale structures in turbulent boundary layers. *J. Fluid Mech.* **628**, 311–337.
- MATHIS, R., MONTY, J. P., HUTCHINS, N. & MARUSIC, I. 2009b Comparison of large-scale amplitude modulation in turbulent boundary layers, pipes, and channel flows. *Phys. Fluids* **21** (11), 111703.
- McKEON, B. J. 2017 The engine behind (wall) turbulence: perspectives on scale interactions. *J. Fluid Mech.* **817**, P1.
- McKEON, B. J. & SHARMA, A. S. 2010 A critical-layer framework for turbulent pipe flow. *J. Fluid Mech.* **658**, 336–382.
- MOARREF, R., JOVANOVIĆ, M. R., TROPP, J. A., SHARMA, A. S. & McKEON, B. J. 2014 A low-order decomposition of turbulent channel flow via resolvent analysis and convex optimization. *Phys. Fluids* **26** (5), 051701.
- MOARREF, R., SHARMA, A. S., TROPP, J. A. & McKEON, B. J. 2013 Model-based scaling of the streamwise energy density in high-Reynolds-number turbulent channels. *J. Fluid Mech.* **734**, 275–316.
- MONTY, J. P., STEWART, J. A., WILLIAMS, R. C. & CHONG, M. S. 2007 Large-scale features in turbulent pipe and channel flows. *J. Fluid Mech.* **589**, 147–156.
- MORRA, P., SEMERARO, O., HENNINGSON, D. S. & COSSU, C. 2019 On the relevance of Reynolds stresses in resolvent analyses of turbulent wall-bounded flows. *J. Fluid Mech.* **867**, 969–984.
- PANTON, R. L. & LINEBARGER, J. H. 1974 Wall pressure spectra calculations for equilibrium boundary layers. *J. Fluid Mech.* **65** (2), 261–287.
- PERRY, A. E. & CHONG, M. S. 1982 On the mechanism of wall turbulence. *J. Fluid Mech.* **119**, 173–217.
- PERRY, A. E., HENBEST, S. & CHONG, M. S. 1986 A theoretical and experimental study of wall turbulence. *J. Fluid Mech.* **165**, 163–199.
- POPE, S. B. 2000 *Turbulent flows*. Cambridge, England: Cambridge university press.
- PUJALS, G., GARCÍA-VILLALBA, M., COSSU, C. & DEPARDON, S. 2009 A note on optimal transient growth in turbulent channel flows. *Phys. Fluids* **21** (1), 015109.
- RENARD, N. & DECK, S. 2015 On the scale-dependent turbulent convection velocity in a spatially developing flat plate turbulent boundary layer at Reynolds number $Re_\theta = 13000$. *J. Fluid Mech.* **775**, 105–148.
- REYNOLDS, W. C. & HUSSAIN, A. K. M. F. 1972 The mechanics of an organized wave in turbulent shear flow. Part 3. Theoretical models and comparisons with experiments. *J. Fluid Mech.* **54** (2), 263–288.
- SCHMID, P. J. & HENNINGSON, D. S. 2001 *Stability and transition in shear flows*. Berlin, Germany: Springer Science & Business Media.
- SHARMA, A. S. & McKEON, B. J. 2013 On coherent structure in wall turbulence. *J. Fluid Mech.* **728**, 196–238.
- SHARMA, A. S., MOARREF, R. & McKEON, B. J. 2017 Scaling and interaction of self-similar modes in models of high Reynolds number wall turbulence. *Phil. Trans. R. Soc. A* **375** (2089), 20160089.
- SMITS, A. J., McKEON, B. J. & MARUSIC, I. 2011 High-Reynolds number wall turbulence. *Annu. Rev. Fluid Mech.* **43** (1), 353–375.
- SQUIRE, D. T., HUTCHINS, N., MORRILL-WINTER, C., SCHULTZ, M. P., KLEWICKI, J. C. & MARUSIC, I. 2017 Applicability of Taylor’s hypothesis in rough-and smooth-wall boundary layers. *J. Fluid Mech.* **812**, 398–417.
- TAYLOR, G. I. 1938 The spectrum of turbulence. *Proc. R. Soc. London A Math. Phys. Eng. Sci.* **164** (919), 476–490.
- TOWNSEND, A. A. 1961 Equilibrium layers and wall turbulence. *J. Fluid Mech.* **11** (1), 97–120.
- TOWNSEND, A. A. 1976 *The structure of turbulent shear flow*. Cambridge, England: Cambridge University Press.
- TREFETHEN, L. N. 2000 *Spectral methods in MATLAB*. Society for Industrial and Applied Mathematics (SIAM).

- VADAREVU, S. B., SYMON, S., ILLINGWORTH, S. J. & MARUSIC, I. 2019 Coherent structures in the linearized impulse response of turbulent channel flow. *J. Fluid Mech.* **863**, 1190–1203.
- WEIDEMAN, J. A. C. & REDDY, S. C. 2000 A MATLAB differentiation matrix suite. *ACM Trans. Math. Softw.* **26** (4), 465–519.
- WILLMARTH, W. W. & WOOLDRIDGE, C. E. 1962 Measurements of the fluctuating pressure at the wall beneath a thick turbulent boundary layer. *J. Fluid Mech.* **14** (2), 187–210.
- WILLS, J. A. B. 1964 On convection velocities in turbulent shear flows. *J. Fluid Mech.* **20** (3), 417–432.
- YANG, X. I. A. & HOWLAND, M. F. 2018 Implication of Taylors hypothesis on measuring flow modulation. *J. Fluid Mech.* **836**, 222–237.
- ZAMAN, K. B. M. Q. & HUSSAIN, A. K. M. F. 1981 Taylor hypothesis and large-scale coherent structures. *J. Fluid Mech.* **112**, 379–396.
- ZARE, A., JOVANOVIĆ, M. R. & GEORGIU, T. T. 2017 Colour of turbulence. *J. Fluid Mech.* **812**, 636–680.