

# Friction between a plane strain circular indenter and a thick poroelastic substrate

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## Abstract

This paper presents a computational study on the role of poroelasticity in gel friction. Motivated by recent experimental studies in the literature, we develop a plane strain finite element model to elucidate the contact mechanics between a circular indenter and a thick poroelastic substrate under both normal and shear loadings. Two cases are considered: i) steady state sliding under fixed normal displacements, and ii) relaxation under fixed normal and shear displacements. In steady state sliding, we find that a net friction force can arise even if no intrinsic adhesive or frictional interaction is implemented at the indenter/substrate interface. Such friction force exhibits a non-monotonic dependence on the sliding velocity and peaks at an intermediate velocity. Our model reveals that this friction force is induced by poroelastic diffusion in the gel substrate which can lead to considerable asymmetry in both the contact profile and contact pressure. In terms of relaxation, if the indenter/substrate interface is set to be frictionless, we find that the friction force induced by poroelasticity relaxes to zero with a characteristic time much faster than that of the normal force. When a finite friction coefficient is introduced at the interface, the normalized relaxation curve for the friction force approaches that for the normal force as the friction coefficient increases. These modeling results suggest that poroelasticity can be an important contributing mechanism for gel friction.

**Keywords:** Friction, poroelasticity, finite element simulations, steady state sliding, relaxation.

## 1. Introduction

The friction of gels is a fundamental problem ubiquitously found in biological systems (e.g., cartilage) and engineering applications (e.g., contact lens). For example, the friction of biological interfaces involving cartilage (Ateshian, 2009; Krishnan et al., 2004; Moore and Burris, 2014) and mucous surfaces (Pult et al., 2015) is directly related to the lubrication and wear of tissues, and thus is of critical importance for applications including artificial joints (Greene et al., 2011, 2008; Jin et al., 1992; McCutchen, 1962) and regenerative medicine (Sterner et al., 2017). More recently, the need to develop medical robotic devices with *in vivo* mobility (Sliker et al., 2012) demands a fundamental understanding of the friction between synthetic materials and biological tissues (Sliker et al., 2010). Unlike macroscopic stiff surfaces that follow Amontons' laws of friction, many biological interfaces are soft, aqueous and dynamic (Gong, 2006; Pitenis et al., 2017), thereby exhibiting complex frictional behaviors far beyond Amontons' laws (Gong et al., 1997). In this regard, hydrogels provide an ideal model system to enable experimental and theoretical investigations on the friction of soft and aqueous interfaces (Gong, 2006). Extensive experimental studies in the literature have revealed that gel friction depends on a wide range of parameters which can be largely categorized into three types: i) loading parameters such as sliding velocity (Delavoipière et al., 2018; Gong et al., 1997; Reale and Dunn, 2017) and normal pressure (Gong et al., 1997), ii) surface conditions such as roughness (Persson and Scaraggi, 2018; Yashima et al., 2014) and interface fluid domain (Yamamoto et al., 2014), and iii) gel properties such as network mesh size (Shoaib et al., 2018), electric charges (Gong et al., 2002), and water content (Dunn et al., 2013).

The dependence on the sliding velocity, referred to as the rate-dependence hereafter, is a key feature of gel friction. Several physical mechanisms can contribute to the rate-dependence of gel friction. The theory established by Gong and Osada (1998) described the interplay of two mechanisms: adhesion and hydrodynamic lubrication. Specifically, this model considers two cases of surface interaction between a hydrogel and a substrate when immersed in water: repulsive or attractive. For the repulsive case, friction is governed by the lubrication of the hydrated water layer of the gel network at the interface. For the attractive case, the adsorption and detachment of gel polymer chains on the substrate surface results in rate-dependent interface adhesion, and thus can also contribute to friction in addition to the lubrication layer. Such adhesive friction was modeled by tracking the stretch of adsorbed chains and the kinetics of chain adsorption and detachment,

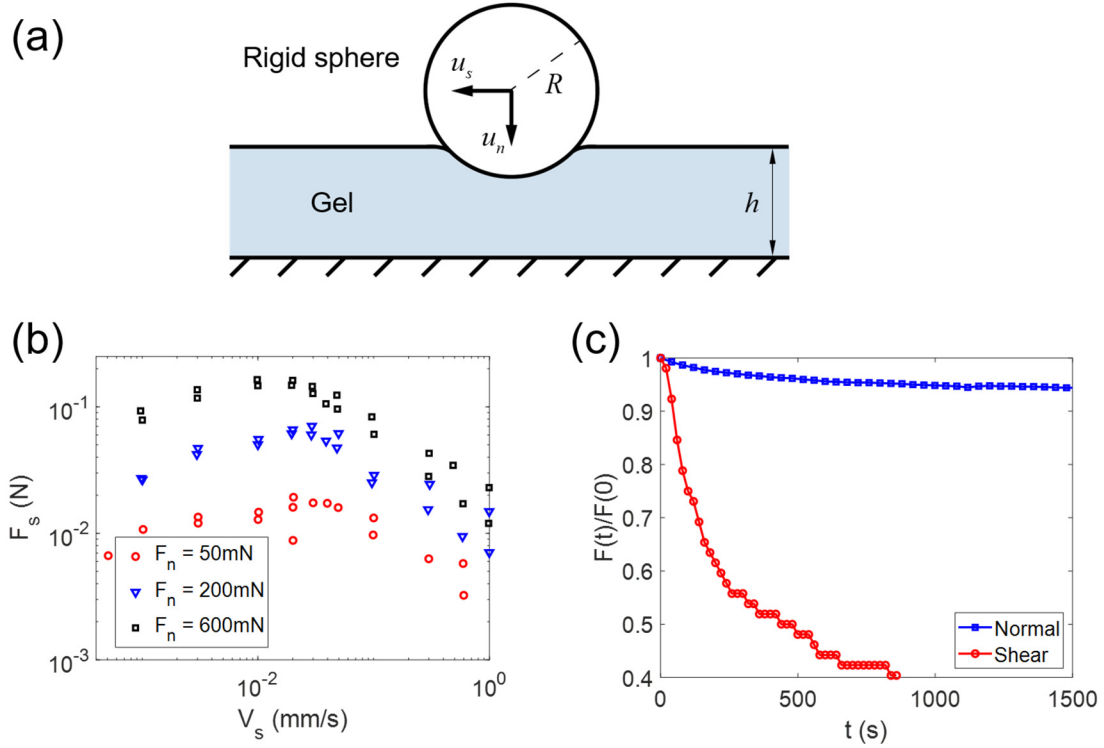
similar to Schallamach's theory of rubber friction (Schallamach, 1963). As the sliding velocity increases, the dominant mechanism transitions from adhesive interaction to hydrodynamic lubrication (Kurokawa et al., 2005). For either adhesion or lubrication, the mechanism described above only accounts for what occurs at the interface. Most gels exhibit rate-dependent, inelastic bulk properties, e.g., viscoelasticity and poroelasticity, which can also lead to rate-dependent friction by inducing energy dissipation upon contact deformation (Delavoipière et al., 2018; Kagata et al., 2002; Rennie et al., 2005; Shoaib and Espinosa-Marzal, 2018). Several recent works were devoted to the coupling between bulk dissipative behavior and interface adhesion (Shoaib and Espinosa-Marzal, 2018) or hydrodynamic lubrication (Smyth and Green, 2017), and how such coupling governs gel friction.

This work focuses on a particular aspect of gel friction: the role of poroelasticity. In recent experimental studies on gel friction, it was found that poroelastic diffusion in the gel substrate can lead to an asymmetric contact area (Delavoipière et al., 2018) and rate-dependent kinetic friction (Delavoipière et al., 2018; Reale and Dunn, 2017). Prior to these works, the poroelastic relaxation in gel substrates upon normal contact with a cylindrical indenter (Hui et al., 2006) or a spherical indenter (Hu et al., 2011, 2010) was leveraged as a method to characterize the elasticity and permeability of gels. Theoretical or computational analyses of the poroelastic contact mechanics involving lateral displacements of the indenter are important for understanding the effect of poroelasticity on gel friction. Delavoipière et al. (2018) developed an analytical model to quantify how poroelasticity affects the contact pressure between a rigid spherical indenter and a thin gel layer under steady state sliding. However, in comparison to normal contact, the analyses of poroelastic contact mechanics with lateral displacements are still limited. Here we present a two-dimensional (2D) finite element analysis on the poroelastic contact mechanics between a rigid circular indenter and a thick gel substrate under the plane strain condition. In Section 2, we highlight relevant experimental results in the literature and define two problems for our finite element model: steady state sliding under fixed normal displacements and relaxation under fixed normal and shear displacements. Modeling results for these two problems are presented and discussed in Sections 3 and 4, respectively, and conclusions are provided in Section 5.

## 2. Problem definition

## 2.1 Experimental results in the literature

In this section, we briefly review relevant experimental works in the literature (Delavoipière et al., 2018; Johannes et al., 2019; Reale and Dunn, 2017) to put our work into perspective. These studies adopted similar experimental configurations, i.e., shearing a rigid spherical probe indented on a gel substrate as shown in Fig.1a. Two types of loading conditions are discussed here: steady state sliding and relaxation.



**Figure 1** (a) Schematic showing the experimental configuration of a rigid sphere indenting and shearing on a gel substrate. (b) Experimental data from literature showing kinetic friction force  $F_s$  under steady state sliding with fixed normal forces  $F_n$  versus sliding velocity  $V_s$ . Data points are extracted from Figure 2 of Delavoipière et al. (2018). (c) Experimental data from literature showing the relaxation of normal or shear force under fixed normal or shear displacements. Data points for the normal force are extracted from the last step loading in Figure 3C (normal) of Johannes et al. (2019) and then normalized. Data points for the shear force are extracted from the first step loading in Figure 4A (shear) of Johannes et al. (2019) and then normalized.

Regarding steady state sliding, Delavoipière et al. (2018) used a glass sphere ( $R = 25.9$  mm) on a thin poly(dimethylacrylamide) gel layer ( $h = 3.1$   $\mu$ m) bonded to a glass surface, while Reale and Dunn (2017) used a smaller glass sphere ( $R = 1.7$  mm) on a thick polyacrylamide gel substrate ( $h =$

9mm). Despite the differences in dimension and material, both works showed that the kinetic friction force  $F_s$  under steady state sliding with a fixed normal force depends on the sliding velocity  $V_s$  (see Fig.1b). This rate dependence was attributed to the competition between poroelastic diffusion in the bulk gel and the advection caused by indenter sliding, which was characterized by the Péclet number  $Pe$ . It is important to note that the definitions of Péclet number in Delavoipière et al. (2018) and Reale and Dunn (2017) differ by a factor of order 1 but are consistent in terms of scaling analysis (see Appendix 1 for a detailed discussion). The experimental results in Delavoipière et al. (2018) show that the kinetic friction force  $F_s$  is a non-monotonic function of the sliding velocity  $V_s$  as illustrated in Fig.1b. The range of sliding velocity corresponds to  $0.04 < Pe < 400$ , and the friction force peaks at  $Pe \sim 1$ . On the other hand, Reale and Dunn (2017) showed that the friction coefficient (or friction force under a fixed normal force) decreases with  $Pe$ , but their experiments cover the range of  $3.5 \leq Pe \leq 5500$  which presumably falls onto the decaying branch of friction coefficient versus  $Pe$ .

Two different approaches were developed to model the dependence of kinetic friction on  $Pe$ : Delavoipière et al. (2018) considered the energy dissipation due to poroelastic diffusion, whereas Reale and Dunn (2017) factored the effect of poroelastic relaxation into an effective, rate-dependent adhesion energy. In principle, these two approaches may be reconciled by recognizing that energy dissipation can lead to an increase in the effective adhesion energy as in the adhesion of viscoelastic solids (Barthel and Frétiigny, 2009). Nevertheless, both approaches are based on energetics. Delavoipière et al. (2018) also developed an analytical model to address the underlying mechanics for the steady state sliding of a rigid spherical indenter on a thin gel substrate. Because of the thin substrate, they first used the elastic foundation approximation to determine the normal strain in the substrate. Using this approximate strain solution, they further calculated the pore pressure field and the interface contact pressure. The model showed that the contact area was circular for  $Pe < 1$ , but became asymmetric when  $Pe > 1$ . Interestingly, in both regimes the contact pressure was found to be asymmetric, i.e., the contact pressure near the leading edge of the indenter was larger than that near the trailing edge, which was induced by the uneven pore pressure in the gel substrate. This result implies that even if the interface is intrinsically frictionless, the poroelasticity could still induce a net friction force through the asymmetric contact pressure. However, since the model by Delavoipière et al. (2018) was approximate and only applicable in the thin substrate limit, whether this implication holds for other geometry (e.g., thick substrate) is

unclear. If it holds, what governs the rate-dependence of poroelasticity induced friction? These questions will be addressed in Section 3.

The previous works reviewed above focus on the rate-dependent kinetic friction force under fixed normal forces. In a recent work (Johannes et al., 2019), we studied the three-dimensional (3D) contact area of a spherical indenter (steel core coated by an elastomer layer with total radius  $R = 265\mu\text{m}$ ) on a polyacrylamide gel substrate ( $h = 420\mu\text{m}$ ). Significant relaxation of normal or shear force was observed under step loadings of normal or shear displacement. For example, the time history of normal and shear forces, normalized by their values at the beginning of relaxation, is shown in Fig.1c. Note that the time in Fig.1c is defined relative to the beginning of relaxation, i.e., just after a step loading of normal or shear displacement is applied. These results show that the shear force relaxes faster and to a much larger extent than the normal force. Although poroelastic relaxation in hydrogel substrates under normal indentation has been extensively modeled (Hu et al., 2011, 2010; Hui et al., 2006), studies on the relaxation under shear displacements are lacking, which will be explored in Section 4.

## 2.2 Finite element model

We build a finite element (FE) model using the commercial software ABAQUS (version 2017, Simulia, Dassault Systèmes, Providence, RI) to study the effect of poroelasticity on gel friction. Since gel friction involves multiple contributing mechanisms, we make the following assumptions to capture the key effects of poroelasticity while keeping the model tractable.

- **Plane strain condition:** the 3D deformation induced by shear loadings of a spherical indenter on a gel substrate is computationally expensive to simulate. Therefore, here we consider a 2D plane strain model consisting of a rigid circular indenter (radius =  $R$ ) on a gel substrate (thickness =  $h$ ).
- **Thick substrate** ( $h \gg R$ ): we assume the limit of thick gel substrate (i.e.,  $h \gg R$ ) to reduce the number of independent parameters, although in experiments the ratio  $h/R$  can cover a large range, e.g.,  $\sim 10^{-4}$  in Delavoipière et al. (2018) and 5 in Reale and Dunn (2017).
- **Linear poroelasticity model:** we assume the gel substrate undergoes infinitesimal displacement and strain during indentation and shear, which allows us to use the linear poroelastic model

(Cheng, 2016) and adopt the linear kinematics by turning off the “NLGEOM” option in ABAQUS.

- **No adhesion at the interface:** interfacial adhesion can be a potentially significant contributing mechanism for gel friction (Gong and Osada, 1998). Here we assume zero adhesion at the interface between the indenter and the gel substrate to isolate the effect of poroelasticity.
- **Friction on the interface:** we first assume the interface to be frictionless and check whether a net friction force can arise solely due to poroelastic diffusion. Next, the model will be extended to include a finite friction coefficient  $\mu$  at the interface.

In reality, a fluid layer may be present between the indenter and the gel substrate, which can influence the friction and adhesion on the interface. For example, the fluid layer can reduce friction through hydrodynamic lubrication. As described in Section 1, this effect has been considered in the gel friction theory by Gong and Osada (1998). As for adhesion, usually it is weak, if not zero, when the contact interface is submerged in fluids (Delavoipière et al., 2018; Johannes et al., 2019; Reale and Dunn, 2017). In certain cases, an interfacial fluid layer may enhance adhesion through capillary interaction (Bhushan, 2003). In this work, we assume zero adhesion and adopt simple interface friction models (i.e., frictionless or constant friction coefficient) in order to focus on the effect of poroelasticity.

The linear poroelastic solid model implemented in our FE simulations is briefly described as follows. We treat the gel substrate as an isotropic, fully saturated poroelastic medium, and assume both the solid and fluid constituents are incompressible. In this case, the stress-strain relation is given by (Cheng, 2016):

$$\sigma_{mn} = 2G\varepsilon_{mn} + \frac{2G\nu}{1-2\nu}\varepsilon_{ss}\delta_{mn} - p\delta_{mn}, \quad (1)$$

where  $\sigma_{mn}$  is the stress component,  $\varepsilon_{mn}$  is the strain component,  $G$  is the shear modulus,  $\nu$  is the Poisson’s ratio in the fully drained state,  $p$  is the pore pressure, and  $\delta_{mn}$  is the Kronecker delta. Note that the subscripts (e.g.,  $m, n$  or  $s$ ) ranges from 1 to 3 and the Einstein’s summation convention of summing over repeated indices is adopted. The flux vector  $\mathbf{q}$ , i.e., the volume of fluid passing through a unit area per unit time, is governed by the Darcy’s law:

$$\mathbf{q} = -\frac{k}{\eta}\nabla p, \quad (2)$$

where  $k$  is the intrinsic permeability (unit:  $\text{m}^2$ ) and  $\eta$  is the dynamic viscosity of the fluid constituent (unit:  $\text{Pa}\cdot\text{s}$ ). Since both the solid and fluid constituents are incompressible, the bulk strain  $\varepsilon_{ss}$  can only be due to fluid transport, which leads to the following continuity equation:

$$\frac{\partial \varepsilon_{ss}}{\partial t} + \nabla \cdot \mathbf{q} = 0, \quad (3)$$

where  $t$  is the time. Combining the strain compatibility equations and eq.(1), we can obtain the following harmonic equation (Cheng, 2016):

$$\nabla^2 \left( p - \frac{2(1-\nu)G}{(1-2\nu)} \varepsilon_{ss} \right) = 0. \quad (4)$$

Substituting eq.(2) and eq.(4) into eq.(3), we obtain a diffusion equation in terms of the bulk strain  $\varepsilon_{ss}$ :

$$\frac{\partial \varepsilon_{ss}}{\partial t} - D \nabla^2 \varepsilon_{ss} = 0, \quad (5)$$

where the diffusivity  $D$  is defined as

$$D = \frac{k}{\eta} \frac{2(1-\nu)G}{(1-2\nu)}. \quad (6)$$

To facilitate the FE simulations, we set the shear modulus  $G = 27.9 \text{ kPa}$  and drained Poisson's ratio  $\nu = 0.28$  according to Hu et al. (2010). Without loss of generality, the permeability is set to be  $k = 9.9 \times 10^{-9} \text{ m}^2$ . Note that the permeability  $k^*$  in ABAQUS (unit:  $\text{m/s}$ ) has a different definition from  $k$  in eq.(2). They are related by

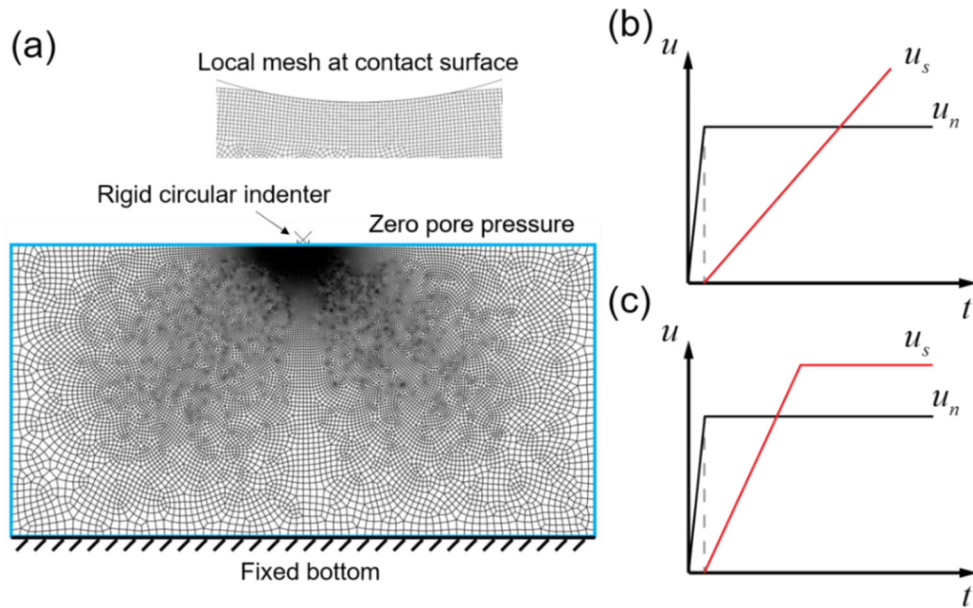
$$k^* = \frac{\gamma_w}{\eta} k, \quad (7)$$

where  $\gamma_w$  is the specific weight (i.e., weight per unit volume) of the fluid constituent. We use default values for both  $\gamma_w$  and  $\eta$  which correspond to  $\gamma_w = 10^3 \text{ N/m}^3$  and  $\eta = 10^3 \text{ Pa}\cdot\text{s}$ . As a result, the diffusivity is  $D = 9.04 \times 10^{-7} \text{ m}^2/\text{s}$ .

The geometry, boundary conditions and loading history of our FE model are illustrated in Fig.2. Figure 2a shows the 2D plane strain mesh. The thickness  $h$  and width  $w$  of the gel substrate are far larger than the radius ( $R=30\text{mm}$ ) of the rigid circular indenter, i.e.,  $h = 50R$  and  $w = 100R$ . The



bottom boundary is fixed and impermeable, while the other three boundaries are free and subjected to zero pore pressure. The contact between the rigid indenter and the gel substrate is assumed to be hard contact. In addition, the contacting interface is assumed to be frictionless or frictional with a coefficient  $\mu$ . The gel substrate is meshed by 73766 CPE4P four-point quadrilateral elements with the smallest element size  $l_{\min} = R/120$  within the contact region such that there are at least 55 elements at the contact interface (see Fig.2a). The “Soil” analysis step in ABAQUS Standard is applied to simulate the poroelastic response of the gel substrate. Additional details on the effect of damping factor and substrate width  $w$  are given in Appendix 2. Although friction experiments (Delavoipière et al., 2018; Reale and Dunn, 2017) are typically conducted under fixed normal forces, here we apply displacement boundary conditions in both the normal and shear directions to ensure convergence of the simulations. Specifically, two cases of loading histories are considered as shown in Fig.2b and Fig.2c. In both cases, a normal indentation displacement  $u_n$  is first applied with a velocity of 10mm/s and then held fixed for the rest of the simulation. The first case corresponds to steady state sliding, where the shear displacement  $u_s$  is continuously increased at a fixed velocity  $V_s$  (see Fig.2b). The second case is for shear force relaxation, where  $u_s$  is first increased at a fixed rate and then held fixed. The directions of  $u_n$  and  $u_s$  are shown in Fig.1a. The normal indentation depth in all simulations is kept small ( $u_n = 1$  mm, 2 mm or 3 mm) to satisfy the infinitesimal deformation assumption.

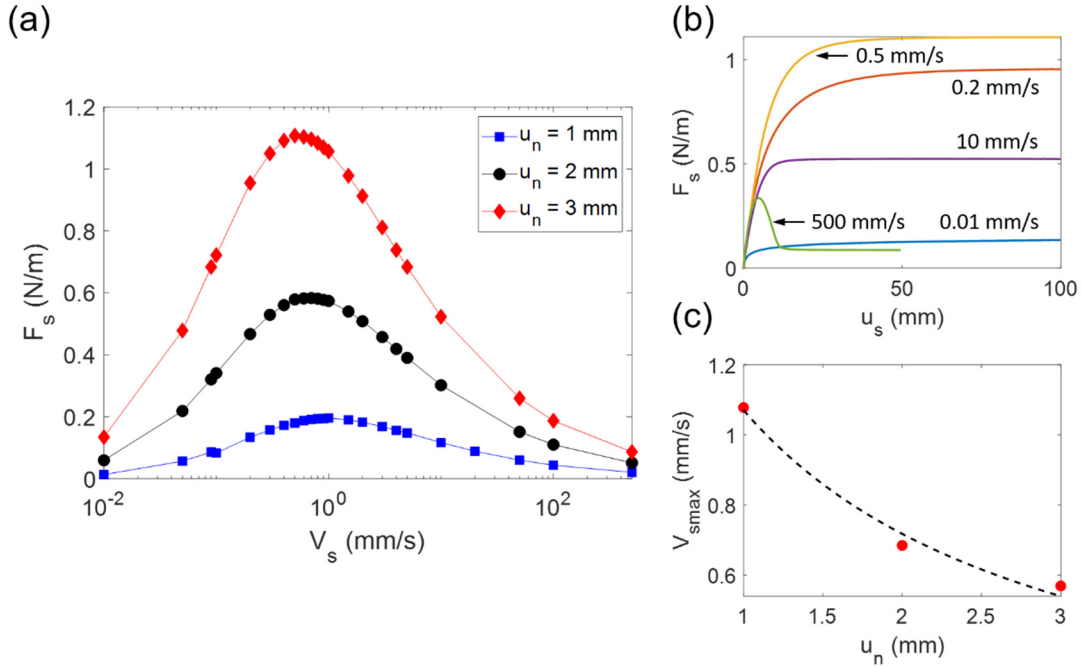


**Figure 2** (a) Mesh geometry and boundary condition of the FE model. (b) Loading history of the normal displacement  $u_n$  and shear displacement  $u_s$  for steady state sliding. (c) Loading history of  $u_n$  and  $u_s$  for shear force relaxation.

### 3. Steady state sliding

#### 3.1 Frictionless interface

We first focus on the case of steady state sliding with a frictionless interface. Under continuous sliding with a fixed normal displacement (see Fig.2b), we find that the indenter experiences a net friction force (see Fig.3b) even though the interface is frictionless with zero shear traction. Figure 3b also shows that the net friction force reaches a steady state value after a transient stage, and the steady state friction exhibits a non-monotonic dependence on the sliding velocity  $V_s$ . In Fig.3a, we plot the steady state friction  $F_s$  as a function of  $V_s$  for three normal indentation depths  $u_n$  ( $= 1, 2$ , or  $3$  mm). These results show that  $F_s$  increases with the indentation depth  $u_n$  at a fixed  $V_s$ , but the velocity  $V_{smax}$  at which the peak friction occurs slightly decreases with  $u_n$  (see Fig.3c).



**Figure 3** (a) Steady state friction force  $F_s$  versus the sliding velocity  $V_s$  for three different normal displacements ( $u_n = 1, 2$ , and  $3$  mm). (b) The friction force  $F_s$  versus shear displacement  $u_s$  at given sliding velocities (normal displacement  $u_n = 3$  mm). (c) The sliding velocity  $V_{smax}$  where peak friction force occurs decays slightly with the prescribed normal displacement  $u_n$ . The symbols are extracted from part (a) and the dashed line is given by an empirical fitting formula,  $V_{smax} = 2.18/(u_n + 1.03)$  to guide the view.

To understand the underlying mechanics of the steady state friction response, we show in Fig.4 the FE results for a representative case with  $u_n = 3$  mm and  $V_s = 0.7$  mm/s. For comparison, the same loading condition is applied to a control case where the substrate is elastic with the same  $G$  and  $\nu (= 0.28)$  as the drained poroelastic solid. Results from the elastic control case are also shown in Fig.4. Since the interface is frictionless, the net friction force, if any, should come from the normal pressure  $p_c$  within the contact region. In Fig.4a where the arc angle  $\theta$  is used to describe the location within the contact region, we see that the contact pressure  $p_c$  is asymmetric about the center of the indenter:  $p_c$  along the leading edge of the contact region (i.e., on the left of the indenter center) is considerably higher than the trailing edge. This is associated with an asymmetric contact region: the leading edge is wider than the trailing edge. As a result, integral of the horizontal component of  $p_c$  in the contact region leads to a non-zero net force resisting the sliding motion, i.e., the friction force. In contrast, Fig.4b shows that for the elastic control, the contact region and  $p_c$  are symmetric about the center of the indenter, which implies a zero net friction force. The zero net friction force for the elastic control is expected since sliding does not induce any additional deformation due to the frictionless interface, and the distribution of  $p_c$  should be the same as that for pure normal indentation. To illustrate this point, we note that analytical solution for  $p_c$  under normal indentation is available based on the plane strain Hertz solution (Johnson, 1987):

$$p_c = \frac{2F_n}{\pi a} \left( 1 - \frac{x^2}{a^2} \right)^{1/2}, \quad (8)$$

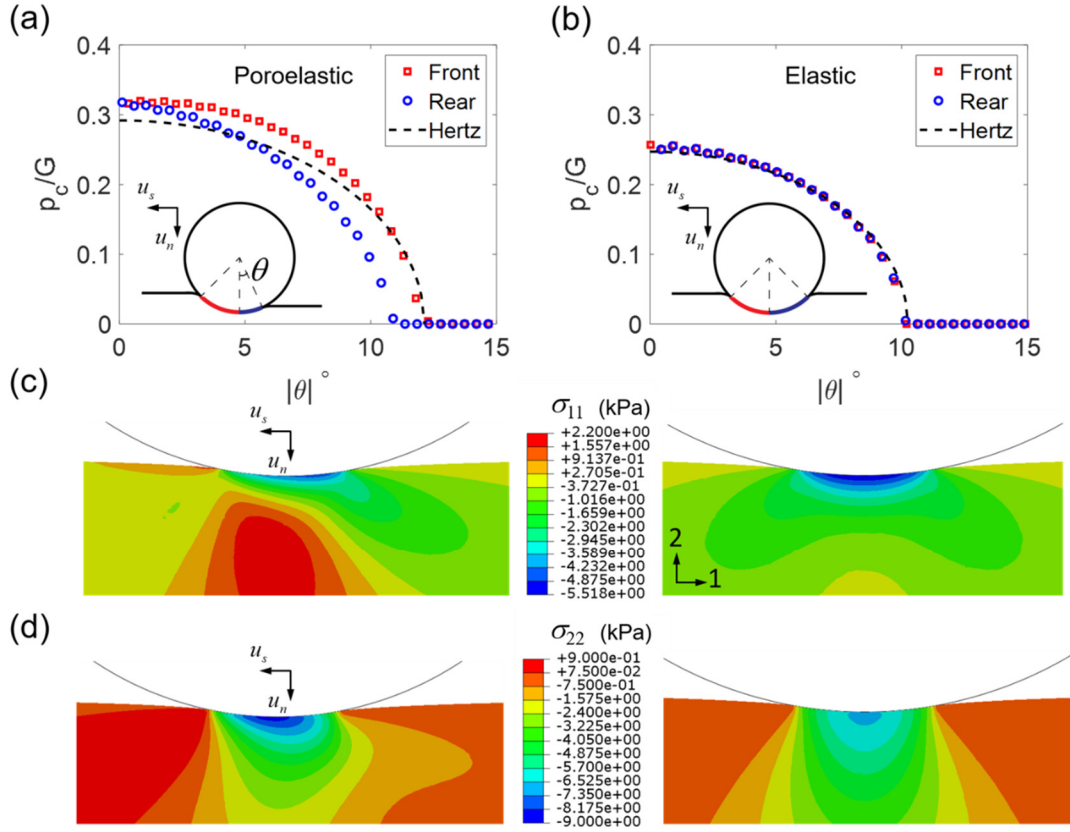
where  $F_n$  is the normal compressive force,  $x$  is the horizontal coordinate measured from the center of contact region, and  $a$  is the half contact width. The contact width  $a$  is related to the normal force by:

$$a^2 = \frac{2(1-\nu)F_n R}{\pi G}, \quad (9)$$

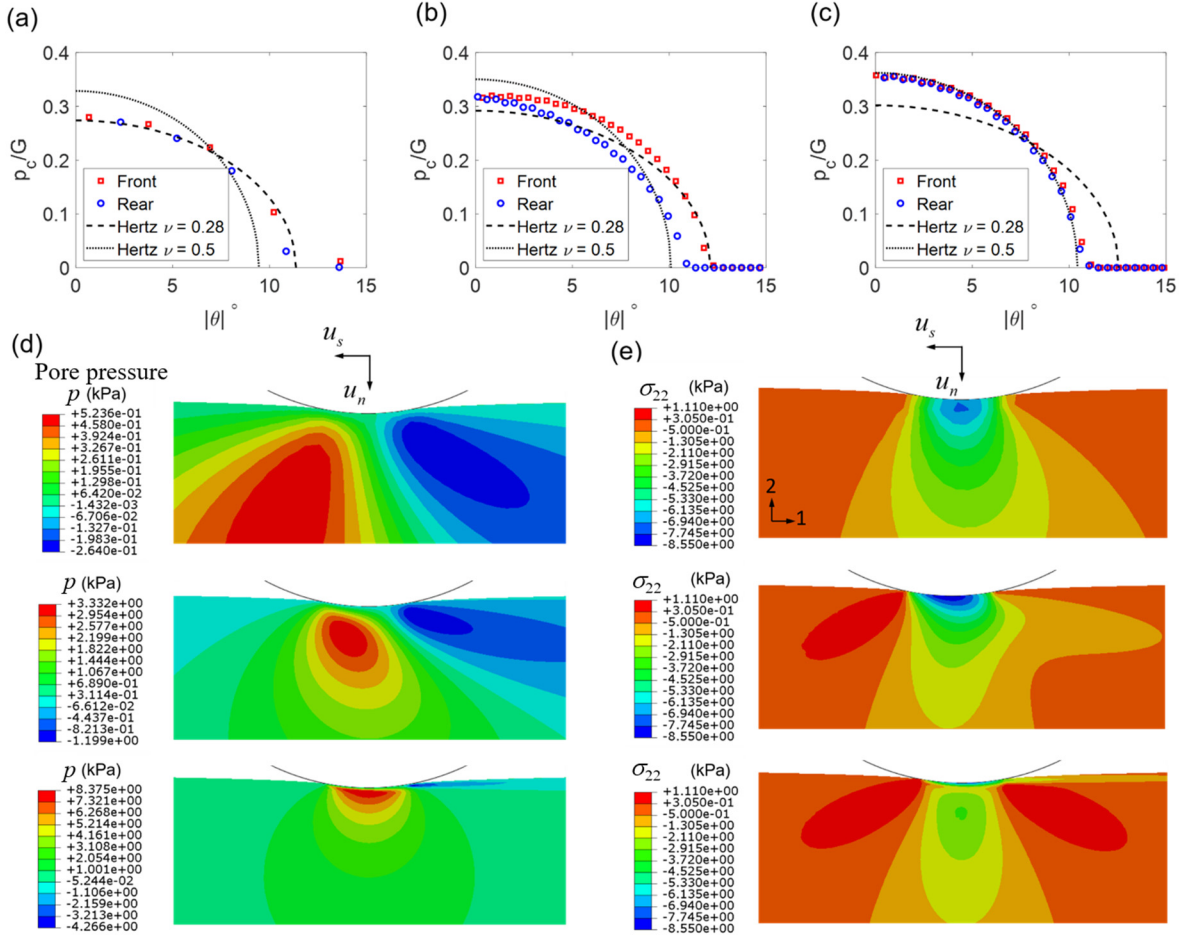
where  $R$  is the indenter radius. Since  $x = R \sin \theta$ , eq.(8) can be rewritten as

$$\frac{p_c}{G} = \frac{1}{(1-\nu)} \left( \frac{a^2}{R^2} - \sin^2 \theta \right)^{1/2}, \text{ where } |\theta| \leq \sin^{-1} \left( \frac{a}{R} \right). \quad (10)$$

Using the value of  $F_n$  from the FE result, we first calculate  $a$  using eq.(9) and then plot eq.(10) in Figs.4a and 4b as the dashed line. It can be seen that eq.(10) agrees well with the FE result of elastic control, but not the poroelastic case.



**Figure 4** (a-b) Normalized contact pressure  $p_c/G$  along the interface for the poroelastic case and the elastic control where  $u_n=3\text{mm}$  and  $V_s=0.7\text{mm/s}$ . The symbols represent FE results and the dashed line represents the plane strain Hertz solution in eq.(10). (c-d) Contours of the normal stress components (c)  $\sigma_{11}$  and (d)  $\sigma_{22}$  in the substrate for the poroelastic case and the elastic control. The subscripts “1” and “2” denote the horizontal and vertical directions, respectively.



**Figure 5** (a-c) Normalized contact pressure  $p_c/G$  along the interface for the poroelastic case with  $u_n = 3$  mm and three different  $V_s$ : (a) 0.05 mm/s, (b) 0.7 mm/s, and (c) 100 mm/s. Also, plotted is the plane strain Hertz solution in eq.(10) with two different Poisson's ratios (0.28 and 0.5) for the drained and undrained states of the poroelastic substrate. (d-e) Contours of (d) pore pressure  $p$  and (e) normal stress  $\sigma_{22}$  in the substrate under steady state sliding. In (d-e), the top, middle and bottom rows correspond the cases with  $V_s = 0.05, 0.7$ , and 100 mm/s, respectively.

The result in Fig.4a leads to the following question: what gives rise to the asymmetric contact pressure in the poroelastic case? To gain insight, we plot the stress fields of  $\sigma_{11}$  and  $\sigma_{22}$  within the substrate in Figs.4c and 4d. Asymmetry in both stress fields is clearly observed for the poroelastic case, in contrast to the symmetric stress fields in the elastic control. Since the pore pressure  $p$  is zero at the interface due to the prescribed boundary condition, it does not contribute directly to the contact pressure  $p_c$ . Instead, the asymmetric profile of  $p_c$  must be related to asymmetric deformation field induced by poroelastic diffusion in the substrate, as manifested by the asymmetric contact width in Fig.4a. As shown in Fig.5, steady state sliding of the indenter can

cause a high pore pressure region in front of the indenter and a low pore pressure region behind it, implying that fluid is driven from the leading edge of the indenter to the trailing edge. This uneven distribution of pore pressure breaks the symmetry of the substrate deformation field, thereby leading to asymmetric contact pressure  $p_c$  and hence a net friction force  $F_s$ . This mechanism also explains why  $F_s$  increases with the normal indentation depth  $u_n$  observed in Fig.3b, because a larger  $u_n$  gives rise to a wider contact region.

Next, we discuss the rate-dependence of poroelasticity induced friction. Figure 5 shows the results for three cases with  $u_n = 3$  mm and different values of  $V_s$  ( $= 0.05, 0.7$  and  $100$  mm/s) that represent the regimes of low, intermediate and high sliding velocities. Specifically, we first plot the contact pressure distribution for  $V_s = 0.05, 0.7$  and  $100$  mm/s in Fig.5a, 5b, and 5c, respectively. Interestingly, the contact pressure is nearly symmetric under both low and high velocities (i.e.,  $0.05$  and  $100$  mm/s), which is consistent with the low friction force in these two regimes as shown in Fig.3a. Figures 5d and 5e show the fields of pore pressure  $p$  and normal stress  $\sigma_{22}$  in the substrate for these three cases. At low velocity ( $V_s = 0.05$  mm/s), the pore pressure field is approximately antisymmetric about the center of the indenter, which consists of a positive  $p$  region ahead of the indenter and a negative  $p$  region behind it (note that  $V_s$  points to the left). However, because the substrate is almost in the drained state due to the low velocity, the magnitude of  $p$  is too small to influence  $\sigma_{22}$ . Therefore,  $\sigma_{22}$  is still approximately symmetric about the indenter center. As  $V_s$  increases to  $0.7$  mm/s, the magnitude of  $p$  is sufficiently large which results in an asymmetric  $\sigma_{22}$  field leading to the asymmetric contact pressure in Fig.5b. At high velocity ( $V_s = 100$  mm/s), the positive  $p$  region dominates and is roughly symmetric about the indenter center, while the negative  $p$  region is suppressed. This corresponds to the undrained state of the substrate. Therefore, even though the magnitude of  $p$  is high,  $\sigma_{22}$  is again nearly symmetric about the indenter center and so is the contact pressure in Fig.5c. It should be noted that these results were based on the assumption of a zero pore pressure condition on the top surface of the gel substrate, including the contact region. In reality, the impermeable indenter may resist poroelastic diffusion across the contact interface. This condition implies that the flux within the contact region should be finite or even zero (i.e., completely impermeable), while the substrate surface outside the contact region is still subjected to zero pore pressure. However, during steady state sliding the contact region keeps moving, which makes it challenging to prescribe different boundary conditions within and outside the contact region. In Appendix 3, we show that the results in Figs.5a-5c do not change

qualitatively if the zero pore pressure condition on the entire top surface of the gel substrate is replaced by a zero flux condition. Therefore, the assumed zero pore pressure condition at the contact interface does not qualitatively affect our conclusion regarding the asymmetry of contact pressure.

To provide theoretical support for the FE results in Fig.5d, we note that an analytical solution for the plane strain problem of steady state translation of a vertical concentrated force acting on a poroelastic half space is available in the literature (Cheng, 2016). Although our problem involves a circular indenter with a finite radius  $R$ , the concentrated force solution can provide a good approximation for material points whose distance to the contact region center is much larger than  $R$ . We set up a moving coordinate system centered at the concentrated force with the  $x$ -axis pointing horizontally left and  $y$ -axis pointing vertically down. The pore pressure field  $p(x, y)$  in the substrate is given by Chapter 7.16 of Cheng (2016)

$$p(x, y) = \frac{1}{\pi} \int_0^\infty \text{Im} \{ i \tilde{p}(y) e^{ilx} \} dl, \quad (11)$$

$$\tilde{p}(y) = \frac{i F_n (1-\nu) \tilde{V}}{\left[ (1-2\nu) \left( \sqrt{1-i\tilde{V}} - 1 \right) + i(1-\nu) \tilde{V} \right]} \left( e^{-ly} - e^{-ly\sqrt{1-i\tilde{V}}} \right), \quad y > 0, \quad (12)$$

and

$$\tilde{V} = \frac{V}{Dl}, \quad (13)$$

where “Im” in eq.(11) means the imaginary part of a complex number,  $l$  is a dummy variable for integration,  $F_n$  is the concentrated force (pointing vertically downward),  $V$  is the translational velocity of the concentrated force,  $D$  is the diffusivity of the poroelastic substrate, and  $\nu$  is the Poisson’s ratio of the drained state. In our problem, the origin of  $x$ - $y$  system is located at the center of contact region and the sliding velocity  $V_s$  is equivalent to  $V$ . Next we consider two limiting cases.

First, in the limit of  $V \rightarrow \infty$ , the first order approximation of eq.(12) is

$$\tilde{p}(y) \approx F_n e^{-ly}, \quad (14)$$

which leads to

$$p(x, y) \approx \frac{F_n}{\pi} \int_0^\infty e^{-ly} \cos(lx) dl = \frac{F_n}{\pi} \frac{y}{x^2 + y^2}. \quad (15)$$

This result implies that the pore pressure field is symmetric about the  $y$ -axis as  $V \rightarrow \infty$ , which is consistent with the high velocity regime ( $V_s = 100\text{mm/s}$ ) shown in Fig.5d. Moreover, eq.(15) coincides with the hydrostatic pressure field given by the Flamant solution for a vertical downward point force acting on an incompressible elastic half space (Johnson, 1987):

$$-\frac{\sigma_{ss}}{3} = \frac{F_n}{\pi} \frac{y}{x^2 + y^2}. \quad (16)$$

Therefore, as the limit of  $V \rightarrow \infty$ , the poroelastic substrate remains in its undrained state and behaves as an incompressible elastic solid.

Second, in the limit of  $V \rightarrow 0$ , the first order approximation of eq.(12) is

$$\tilde{p}(y) \approx 2F_n(1-\nu)e^{-ly} \left( 1 - e^{\frac{i\tilde{V}ly}{2}} \right), \quad (17)$$

which leads to

$$p(x, y) \approx \frac{F_n(1-\nu)}{\pi} \frac{Vy}{D} \int_0^\infty e^{-ly} \sin(lx') dl = \frac{F_n(1-\nu)V}{\pi D} \frac{xy}{x^2 + y^2}. \quad (18)$$

This result has two implications: i) the pore pressure field is antisymmetric about the  $y$ -axis as  $V \rightarrow 0$ ; ii) the magnitude of pore pressure  $p$  scales linearly with  $V$  and thus is small, both of which are consistent with the low velocity regime ( $V_s = 0.05\text{ mm/s}$ ) shown in Fig.5d. In this limit, the poroelastic substrate is in the drained state and behaves as a compressible elastic solid.

The drained and undrained limits are further illustrated by plotting the plane strain Hertz solution of contact pressure  $p_c$  in Figs.5a-5c. Specifically, we first use the normal force  $F_n$  from the FE result to determine the contact width in eq.(9) and then calculate  $p_c/G$  using eq.(10). Unlike Figs.4a-4b, here we plot two cases with different Poisson's ratios:  $\nu = 0.28$  or  $0.5$ . The former corresponds to the drained limit (compressible) of the gel, and the latter corresponds to the undrained limit (incompressible) of the gel. The contact pressure profiles in the low velocity (see Fig.5a) and high velocity (see Fig.5c) regimes (see Fig.5a) agree well with the drained elastic solution ( $\nu = 0.28$ ) and undrained elastic solution ( $\nu = 0.5$ ), respectively. In the intermediate

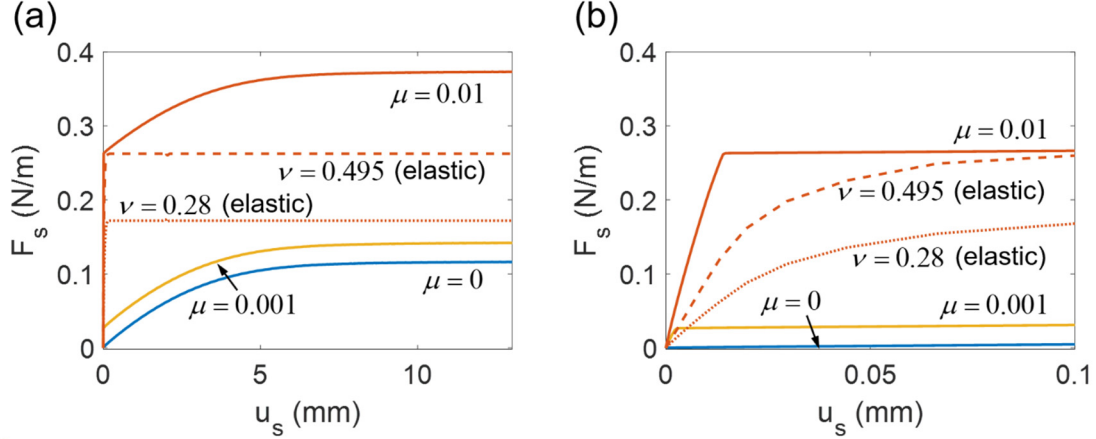


velocity regime (see Fig.5b), the contact pressure profile deviates from both elastic solutions. The FE result of  $p_c$  near the center of contact region ( $|\theta| \sim 0^\circ$ ) is bounded between the two elastic solutions, indicating that this is a region of active poroelastic diffusion. Interestingly,  $p_c$  approaches the drained elastic solution near the leading edge of the contact region and the undrained elastic solution near the trailing edge of the contact region. This suggests that the substrate material near the leading edge of the contact region is more compliant than that near the trailing edge, which leads to a larger contact width at the leading edge.

Finally, we note that the non-monotonic rate-dependence of friction force and the asymmetric contact region have been experimentally observed by Delavoipière et al. (2018) for a spherical indenter on a thin hydrogel substrate. The friction experiments in Delavoipière et al. (2018) were conducted under fixed normal forces. Therefore, the decrease in friction force at high sliding velocities is accompanied by a reduction in the size of contact area. In contrast, our simulations were performed under fixed normal displacements, where the contact size is approximately the same for different sliding velocities. The rate-dependence of friction force observed in our simulations is mainly caused by the rate-dependent pore pressure field and the asymmetry in contact pressure, as explained above.

### 3.2 Frictional interface

When a non-zero friction coefficient  $\mu$  is implemented at the interface, poroelasticity can still contribute to the steady state friction force by inducing an extra net shear force through the mechanism of asymmetric contact pressure. This phenomenon is illustrated in Fig.6 where the friction force  $F_s$  versus shear displacement  $u_s$  curves are plotted for  $u_n = 1$  mm,  $V_s = 10$  mm/s and three different friction coefficients  $\mu$ . For cases with non-zero  $\mu$ , the friction  $F_s$  first increases abruptly when  $u_s$  is small (see Fig.6b), followed by a more gradual increase of  $F_s$  until a plateau is achieved. The first stage (i.e., abrupt increase of  $F_s$ ) is due to static friction, where the substrate deforms with a fixed contact area and  $F_s$  results from frictional shear tractions on the interface. As a result, this stage is absent for  $\mu = 0$ . The second stage (i.e., gradual increase of  $F_s$ ) is attributed to poroelastic diffusion associated with indenter sliding, and follows a similar trend for all three cases of  $\mu$ .



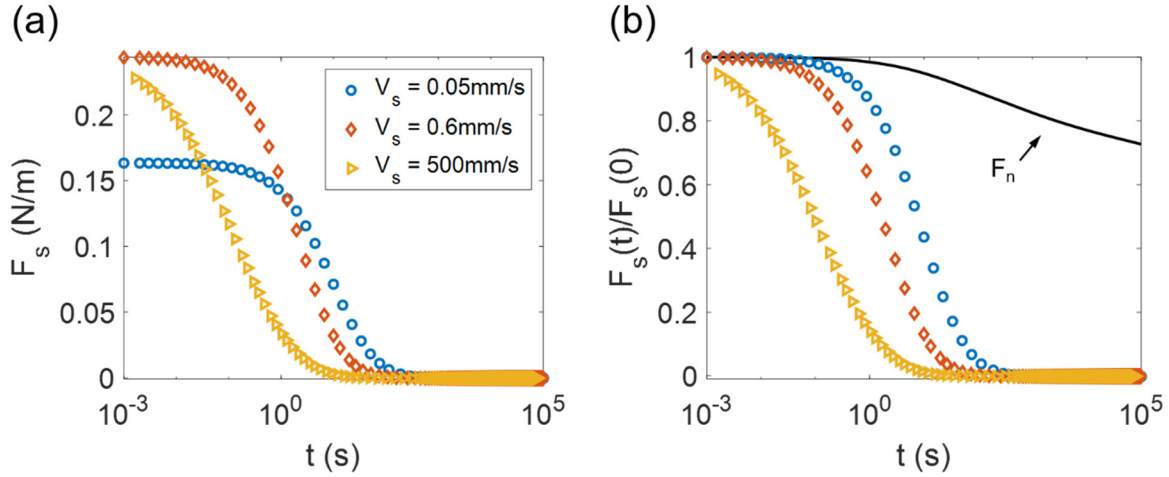
**Figure 6** (a) Friction force  $F_s$  versus shear displacement  $u_s$  for three different interface friction coefficients ( $\mu = 0, 0.001$ , and  $0.01$ ). Also plotted are the results for elastic substrate with the same shear modulus  $G$  as the poroelastic substrate and Poisson's ratio  $\nu = 0.28$  or  $0.495$ . Note that the normal displacement is fixed at  $u_n = 1$  mm and the sliding velocity is  $V_s = 10$  mm/s. (b) A zoom-in view at small  $u_s$  to show the abrupt increase of  $F_s$  at small  $u_s$ .

To highlight the effect of poroelasticity, we take the case of  $\mu = 0.01$  as an example and replace the poroelastic substrate by an elastic one with the same  $G$  and  $\nu (= 0.28)$  as the drained limit. Result for the elastic case ( $\nu = 0.28$ ) is shown in Fig.6 as a dotted line. As expected,  $F_s$  settles at a constant value once sliding starts. Interestingly, the elastic case ( $\nu = 0.28$ ) exhibits a higher shear compliance than the poroelastic case during the static friction stage (see Fig.6b), and a lower shear force  $F_s$  than the poroelastic case at the onset of sliding. We attribute the difference to the fact that the elastic case with  $\nu = 0.28$  represents the drained limit and thus is more compliant than the poroelastic substrate. This is evidenced by the dashed line in Fig.6 where we change the Poisson ratio  $\nu$  of the elastic case to  $0.495$ , i.e., closer to the undrained limit of the poroelastic substrate. The elastic case with  $\nu = 0.495$  approaches the poroelastic case during the static friction stage, and the onset of sliding occurs at the same  $F_s$ . After the onset of sliding, the friction force in the poroelastic case continues to increase, which corresponds to the transient build-up of asymmetric contact pressure before a steady state is achieved.

## 4. Relaxation under fixed shear displacement

### 4.1 Frictionless interface

Poroelastic diffusion in the substrate can also lead to the relaxation of normal force  $F_n$  and shear force  $F_s$  if the displacements  $u_n$  and  $u_s$  are fixed. Here we focus on relaxation of the shear force  $F_s$  which will be compared with that of the normal force  $F_n$ . We start with the frictionless case and apply the loading history in Fig.2c. Specifically, the normal displacement  $u_n$  is first increased to 3mm at 10mm/s and then held fixed. After that, the shear displacement  $u_s$  is increased to 2mm at a velocity  $V_s$  and then held fixed. The relaxation histories of  $F_s$  for three cases with different  $V_s$  are shown in Fig.7a. Note that the time  $t$  is relative to the onset of fixing  $u_s$ , which is different for these two cases. The initial value of  $F_s$  at the beginning of relaxation ( $t = 0$ ) is different in the three cases due to the rate dependence of  $F_s$  under sliding. The long-term limit of  $F_s$  is practically zero, which is expected because of the frictionless interface.



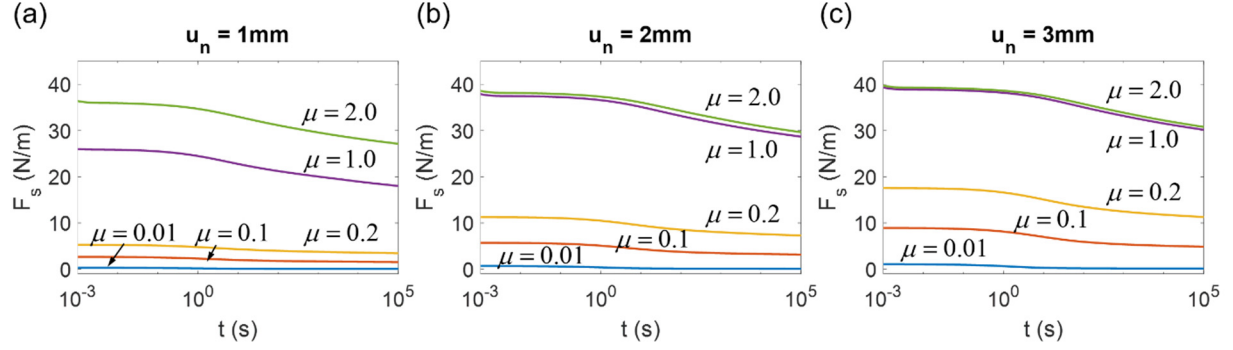
**Figure 7** (a) Time history of shear force  $F_s$  after  $u_s$  is increased to 2mm at  $V_s = 0.05, 0.6$ , or 500mm/s and then held fixed. (b) Normalized time history for the normal force  $F_n$  (solid line) and shear force  $F_s$  (dashed lines).

To facilitate comparison between the relaxation history of  $F_n$  and  $F_s$ , we plot the force normalized by its initial value at the beginning of relaxation ( $t = 0$ ) in Fig.7b. Again, the time  $t$  is not defined in the absolute sense, but rather relative to the onset of fixing  $u_n$  (for  $F_n$ ) or  $u_s$  (for  $F_s$ ). Two important differences between the relaxation of normal and shear forces are observed: i) the shear force  $F_s$  relaxes much faster than the normal force  $F_n$ ; (ii) the shear force  $F_s$  relaxes to zero while the normal force  $F_n$  relaxes to a plateau value related to the drained state. Both features are consistent with experimental data from literature as shown in Fig.1c. The difference in relaxation time between  $F_n$  and  $F_s$  is intriguing. In principle, the characteristic relaxation time  $\tau$  due to

poroelastic diffusion can be estimated by  $\tau \sim L^2/D$ , where  $L$  is the size of the diffusion region involved in relaxation and  $D$  is the poroelastic diffusivity (see eq. (6)). The relaxation processes of  $F_n$  and  $F_s$  should involve the same characteristic length  $L$ , i.e., the contact width, and yet exhibit substantially different relaxation times. To understand this behavior, we recall that the shear force  $F_s$  on a frictionless substrate relies on asymmetry of the contact region. Poroelastic diffusion not only reduces the contact pressure over time, but also drives interfacial slippage which decreases the extent of contact region asymmetry. While the relaxation of normal force  $F_n$  is dominated by the reduction in contact pressure, the relaxation of shear force  $F_s$  is governed by the combined effect of contact pressure reduction and interfacial slippage. This mechanism explains why  $F_s$  relaxes much faster than  $F_n$ . It also implies that the relaxation of  $F_s$  is dependent on the loading history, e.g., the sliding velocity  $V_s$  before the onset of relaxation can affect the asymmetry of contact region and hence the subsequent time history of relaxation, as shown in Fig. 7b.

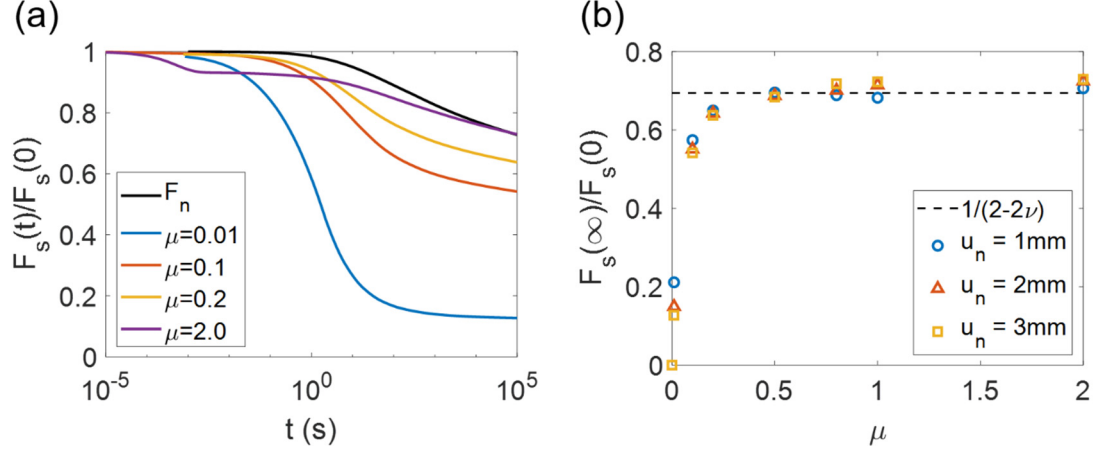
## 4.2 Frictional interface

Based on the discussion in Section 4.1, we hypothesize that if interfacial slippage is suppressed, the relaxation history of shear force  $F_s$  should approach that of the normal force  $F_n$ . To test this hypothesis, we increase the interfacial friction coefficient  $\mu$  in the FE model and rerun the simulations where the indenter is first moved to a normal displacement  $u_n$  (=1, 2, or 3mm) at 10mm/s and then to a shear displacement  $u_s$  =2mm at 500mm/s. Figure 8 shows the relaxation histories of  $F_s$  for five cases with different  $\mu$ . Similar behaviors are observed for all three normal displacements  $u_n$ . As expected, increasing  $\mu$  leads to larger  $F_s$  over the entire time span of relaxation. More interestingly, the relaxation appears to become slower as  $\mu$  increases from 0.01 to 2. This is consistent with our hypothesis that larger  $\mu$  suppresses interfacial slippage and thus slows down the relaxation of  $F_s$ .



**Figure 8** Time history of shear force  $F_s$  when  $u_s$  is increased to 2mm at  $V_s = 500\text{mm/s}$  and then held fixed. Five cases of interfacial friction coefficient  $\mu$  ( $=0.01, 0.1, 0.2, 1, 2$ ) and three fixed normal displacements: (a)  $u_n = 1\text{mm}$ , (b)  $u_n = 2\text{mm}$ , and (c)  $u_n = 3\text{mm}$  are considered.

We take a closer look at the shear force relaxation by plotting the normalized time history for  $u_n = 3\text{mm}$  in Fig.9a. Also included for comparison is the relaxation time history of the corresponding normal force. As the friction coefficient  $\mu$  increases from 0.01 to 0.2, the relaxation curve of shear force  $F_s$  clearly approaches that of the normal force  $F_n$ , both in terms of the relaxation time and the long-term plateau value. However, when  $\mu$  is as large as 2, we observe a peculiar fast relaxation event occurring before  $t \sim 10^{-3}$  s in addition to the main relaxation history. Similar behaviors are observed for  $\mu = 0.8$  and 1. Indeed, the normalized relaxation curves for  $\mu = 0.8$  and 1 overlaps that for  $\mu = 2$  and thus are not shown in Fig.9a. To rule out the possibility of numerical error, we refine the mesh for the case of  $\mu = 2$  so that the minimum element size is reduced by 20 times, but still obtain the same result as that in Fig.9a. Next we consider possible physical explanations. Since relaxation is driven by poroelastic diffusion in the substrate, the fast relaxation event observed for  $\mu = 0.8$  to 2 indicates that there exists a deformation length scale much smaller than the contact width. One possible mechanism is that the compression at the leading edge due to the shear loading can induce local wrinkling instability. This behavior has been observed in friction experiments between a spherical indenter and a rubber substrate, and serves as the precursor for a phenomenon known as the Schallamach wave (Barquins, 1985). The tendency of local instability within the contact region can provide an additional length scale that is much smaller than contact width. The exact coupling between poroelastic diffusion and local elastic instability is an interesting problem that requires a separate study and is not pursued here.



**Figure 9** (a) Normalized time history of shear force  $F_s$  under  $u_n = 3$  mm with different interfacial friction coefficients  $\mu$  and for the corresponding normal force  $F_n$ . (b) The long-term plateau value of shear force relaxation  $F_s(\infty)/F_s(0)$  versus friction coefficient  $\mu$ .

Finally, we plot the long-term plateau value of the normalized shear relaxation curve, referred to as  $F_s(\infty)/F_s(0)$ , but actually evaluated at  $t = 10^5$  s due to the finite time span of FE results. As shown in Fig. 9b, the plateau value increases substantially with  $\mu$ . Specifically,  $F_s(\infty)/F_s(0)$  starts at  $\sim 0.2$  when  $\mu = 0.01$  and settles at  $1/(2-2\nu)$  when  $\mu \geq 0.5$ . Note that  $1/(2-2\nu)$  is the theoretical plateau value for the normal force relaxation  $F_n(\infty)/F_n(0)$ , which describes the transition from the undrained state to the drained state of the substrate.

## 5. Conclusions

We present a computational study on the contact mechanics between a rigid indenter and a poroelastic substrate under plane strain conditions. Our focus is on the resultant shear force on the indenter under two types of loading conditions: i) continuous sliding with a fixed normal displacement and ii) relaxation under fixed normal and shear displacements.

- For continuous sliding, we find that poroelastic diffusion in the substrate can lead to a net shear force even if the contact interface is frictionless. Our FE data reveals that the poroelasticity induced friction originates from the asymmetric pore pressure field in the substrate induced by sliding. This further causes asymmetric contact pressure at the interface and hence the net friction force. The magnitude of such poroelasticity induced friction is sensitive to the sliding

velocity: it diminishes at either low or high sliding velocities, where the substrate behaves as drained or undrained elastic solid, and the peak friction occurs at an intermediate velocity. We find that the rate-dependent pore pressure field within the substrate is responsible for the dependence of friction on sliding velocity. For cases with a non-zero interfacial friction coefficient, poroelasticity can still induce an extra contribution to the steady state friction.

- For relaxation, we find that the poroelasticity induced friction force on a frictionless interface relaxes much faster than the normal force. In addition, the long-term plateau of the relaxation is zero. For cases with a non-zero interfacial friction coefficient, the relaxation curve for friction force approaches that of the normal force as the friction coefficient increases, both in terms of the relaxation time and the long-term plateau.

Although our study is based on idealized plane strain geometry and a linear poroelastic model, the conclusions drawn from the FE results are qualitatively consistent with experimental data in the literature. More importantly, the FE results provide theoretical insights to elucidate the effects of bulk poroelasticity on the friction of gels.

## Acknowledgement

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## Appendix 1 Péclet number

As described in Section 2.1, despite the differences in geometrical dimensions, both Delavoipière et al. (2018) and Reale and Dunn (2017) studied the steady state sliding of a rigid glass sphere on a gel substrate under fixed normal force  $F_n$ , and defined Péclet number to describe the competition between poroelastic diffusion and advection induced by sliding. Their definitions of Péclet number are consistent in scaling but are different in values. In Delavoipière et al. (2018), the Péclet number is defined as

$$Pe_1 = \frac{V_s}{a} \frac{a_0^2}{4 \frac{k}{\eta} \frac{2(1-\nu)G}{1-2\nu}} = \frac{V_s a}{4D(a/a_0)^2}, \quad (19)$$

where  $V_s$  is the sliding velocity,  $D$  is the poroelastic diffusivity of the gel,  $a$  is the contact radius during steady state sliding and  $a_0$  is the long-term equilibrium contact radius under a fixed normal force  $F_n$ . Note that the ratio  $a/a_0$  is smaller than or equal to 1 because of the fixed normal force (rather than fixed normal displacement) during sliding. Indeed, Delavoipière et al. (2018) experimentally found that  $a/a_0$  is roughly 1 when  $Pe_1 < \sim 1$ , and decreases to  $\sim 0.6$  for  $Pe_1$  up to 100. In addition, Delavoipière et al. (2018) used a very thin gel substrate, and therefore  $a_0$  was given by a thin film approximation rather than the Hertz theory. In contrast, the Péclet number in Reale and Dunn (2017) is defined as

$$Pe_2 = \frac{2V_s a}{D}. \quad (20)$$

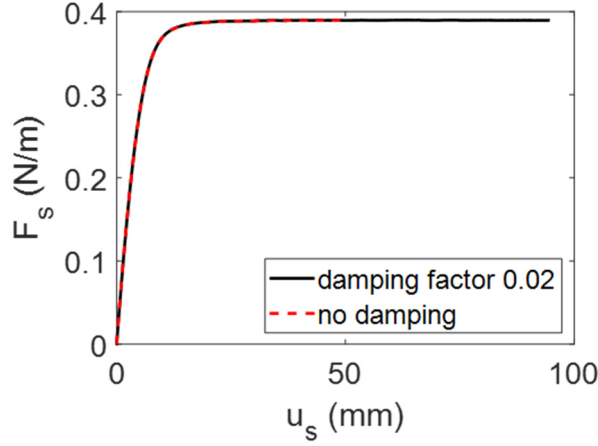
Since  $a/a_0 \leq 1$ , the difference between eq.(19) and eq.(20) can be as large as eight times.

Delavoipière et al. (2018) found that the peak friction force under steady state sliding occurs at  $Pe_1 \sim 1$ . Since  $a/a_0 \approx 1$  near  $Pe_1 \sim 1$ , eq.(19) reduces to  $Pe_1 \approx V_s a / 4D$ . Using this approximation and treating  $a$  as half of the total contact width, we found that the Péclet number associated with the peak friction in Fig.3a is  $Pe_1 = 1 \sim 1.7$ , which is on the same order as the result in Delavoipière et al. (2018). However, due to the large differences in geometry (e.g., plane strain versus spherical indentation; thick substrate versus thin substrate) and loading condition (fixed normal displacement versus fixed normal force) between our model and the experiments in Delavoipière et al. (2018), quantitative comparisons are not pursued here.

## Appendix 2 Additional details of the FE simulations

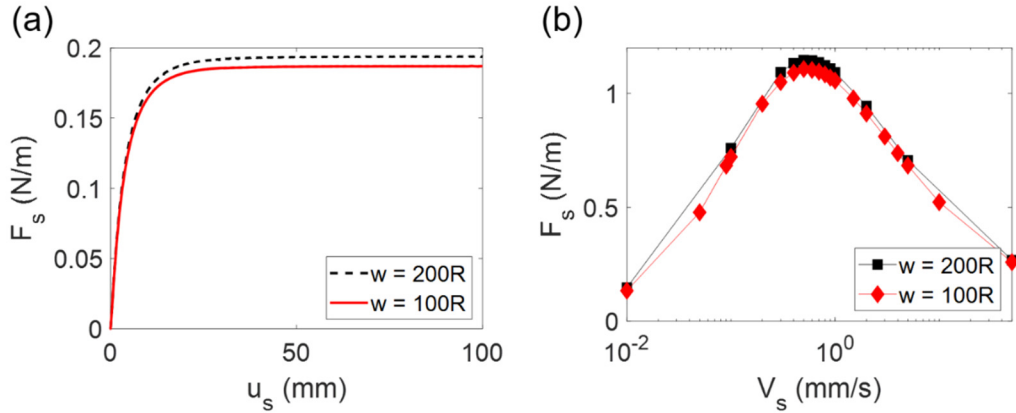
To improve convergence in some cases of FE simulations, we applied the automatic stabilization option in ABAQUS by specifying a constant damping factor ( $= 0.02$ ), which adds a fictitious viscous force to the global equilibrium equation. Such damping factor does not affect the result. In Fig.10, we plot the result of shear force  $F_s$  versus shear displacement  $u_s$  under continuous sliding either using a damping factor of 0.02 or with no damping factor. The results are nearly identical.





**Figure 10** Effect of damping factor on the FE results. Shear force  $F_s$  versus shear displacement  $u_s$  under continuous sliding with a fixed normal displacement  $u_n = 2\text{mm}$  and sliding velocity  $V_s = 5\text{mm/s}$ .

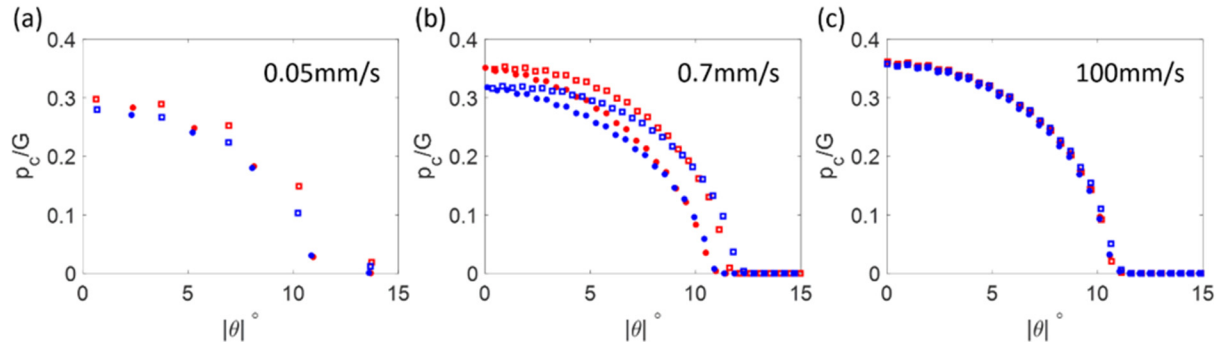
Even though we set the lateral width of the gel substrate  $w$  to be much larger than indenter radius  $R$  (i.e.,  $w = 100R$ ), our FE results still exhibit a minor dependence on  $w$ . This is attributed to the well-known unbounded far field displacements in the Flamant solution and consequently the plane strain contact mechanics solution with an elastic half space (Johnson, 1987). For example, by increasing the  $w$  to be  $200R$ , the shear force  $F_s$  increases slightly, as shown in Fig.11 a. However, the quantitative trend of the function  $F_s$  versus  $V_s$  is independent of  $w$ .



**Figure 11** Effect of the substrate width  $w$  on the FE results. (a) Shear force  $F_s$  versus shear displacement  $u_s$  under continuous sliding with a fixed normal displacement  $u_n = 1\text{mm}$  and sliding velocity  $V_s = 0.6\text{mm/s}$ . (b) Dependence of the steady state friction force  $F_s$  on the sliding velocity  $V_s$  for the normal displacement of  $u_n = 3\text{mm}$ .

### Appendix 3 Effect of poroelastic boundary condition

To illustrate the effect of poroelastic boundary condition, we replaced the zero pore pressure condition at the entire top surface of the gel substrate by a zero flux condition, and reran the simulations of steady state sliding shown in Figs.5a-5c. Results based on the zero pore pressure condition or the zero flux condition are compared in Fig.12. One can see that the zero flux condition leads to a slight increase in contact pressure, but does not qualitatively change the results.



**Figure 12** Normalized contact pressure  $p_c/G$  along the interface for the poroelastic case with  $u_n=3\text{mm}$  and three different  $V_s$ : (a)  $0.05\text{mm/s}$ , (b)  $0.7\text{mm/s}$ , and (c)  $100\text{mm/s}$ . The open square symbols represent points near the leading edge, while the closed circular symbols represent points near the trailing edge. The blue symbols are obtained using a zero pore pressure condition on the top surface of gel substrate, while the red symbols represent the results using a zero flux condition.

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