

Hydrodynamics and scaling laws for intermittent S-start swimming

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The hydrodynamics of a self-propelling swimmer undergoing intermittent S-start swimming are investigated extensively with varying duty cycle DC , swimming period T , and tailbeat amplitude A . We find that the steady time-averaged swimming speed \bar{U}_x increases directly with A , but varies inversely with DC and T , where there is a maximal improvement of 541.29 % over continuous cruising swimming. Our results reveal two scaling laws, in the form of input versus output relations, that relate the swimmer's kinematics to its hydrodynamic performance: swimming speed and efficiency. A smaller DC causes increased fluctuations in the swimmer's velocity generation. A larger A , on the other hand, allows the swimmer to reach steady swimming more quickly. Although we set out to determine scaling laws for intermittent S-start swimming, these scaling laws extend naturally to burst-and-coast and continuous modes of swimming. Additionally, we have identified, categorized and linked the wake structures produced by intermittent S-start swimmers with their velocity generation.

Key words: swimming/flying

1. Introduction

Predator-prey behaviour on land, in the air and on water is a fascinating natural phenomenon. For example, to escape from a pursuer, a swimming fish can reach more than ten times gravitational acceleration with only a 30 cm long body by using fast-start propulsion (Triantafyllou, Weymouth & Miao 2016). Indeed, humans look up to the

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fast-starting techniques of aquatic animals and hope to apply them to launch bio-inspired underwater robots.

Two fast-starting strategies are usually employed by swimmers, the C- and S-start, according to the bending shape of the swimmer. The S-start propulsion strategy allows the prey to remain in its initial swimming direction and win the pursuit–evasion game. It differs from the C-start strategy, which causes the fish to rotate and change its initial orientation (Weihs 1973). Besides serving as an escape response, S-start behaviour is also used during prey strikes (Domenici & Blake 1997). Recently, much effort has been put into understanding the hydrodynamics of C-start swimming (e.g. Borazjani *et al.* 2012; Gazzola, Rees & Koumoutsakos 2012; Li *et al.* 2014), while little is known about the S-start motion (Triantafyllou 2012).

We conducted a biological experiment on zebrafish (*Danio rerio*) to better understand S-start swimming. The experiment uses zebrafish with body length 3 cm. Experiments are carried out in a cubic water tank of dimensions 50 cm \times 3 cm \times 3 cm. In the water tank, the top is open for taking movies, and the bottom is illuminated by LEDs. High-speed cameras are used to track fish movement. A sufficient amount of lighting is used to create a well-lit environment for recording movies. Interestingly, we find that the locomotion of zebrafish generally consists of two phases: an S-start swimming phase for providing initial acceleration, followed by a gliding phase for saving energy (see supplementary movie 1 available at <https://doi.org/10.1017/fms.2024.103>). In this study, we refer to the combined locomotive gait as intermittent S-start swimming.

The trajectories of swimming zebrafish during intermittent S-start and continuous swimming were analysed, and we observed distinct behaviour (see figure 1(d) and supplementary movie 1). In S-starts, zebrafish displayed more frequent tail undulations (characterized by a shorter period T_s), resulting in pronounced lateral movements. This is followed by a glide phase (T_g) without noticeable lateral motion. The continuous swimming of zebrafish displayed a lower tail undulation frequency or longer swimming period T . For S-start swimming, the total period T can be decomposed into an initial S-start phase of period T_s , which is followed by a gliding phase of period T_g , i.e. $T = T_s + T_g$ (figure 1b). In chase–escape scenarios, zebrafish benefit from a higher undulation frequency in the S-start phase. During the gliding phase, energy may be conserved, or this behaviour may develop passively as a result of elevated oxygen consumption rates during S-starts (Brett & Sutherland 1965).

Intermittent S-start swimming shares kinematic characteristics with burst-and-coast (B-and-C) swimming (Gleiss *et al.* 2011; Dai *et al.* 2018). A B-and-C swimming strategy involves an undulating burst phase with time period T_{burst} , and a non-undulating coast phase with time period T_{coast} (figure 1c). In B-and-C swimming, swimmers employ the same swimming frequency as in continuous swimming, i.e. $T_{burst} = T$ (figures 1a,c). A B-and-C swimming strategy is usually adopted to conserve energy, characterized by low swimming speeds (Chuang 2009; Floryan, Van Buren & Smits 2017; Akoz & Moore 2018; Akoz *et al.* 2019; Liu, Huang & Lu 2020; Ashraf, Wassenbergh & Verma 2021; Gupta *et al.* 2021; Li *et al.* 2021). S-start swimming, on the other hand, is characterized by higher swimming frequencies, with swimmers striving for higher swimming speeds rather than enhanced efficiency.

As S-start swimming plays an important role in predator–prey behaviour, some questions arise naturally. How do kinematic parameters, such as tailbeat amplitude A , duty cycle DC , and swimming period T , influence the swimming performance (speed and energy consumption) of intermittent S-start swimmers? Do scaling laws apply to intermittent S-start swimming as they do to continuous swimming? What kind of wake

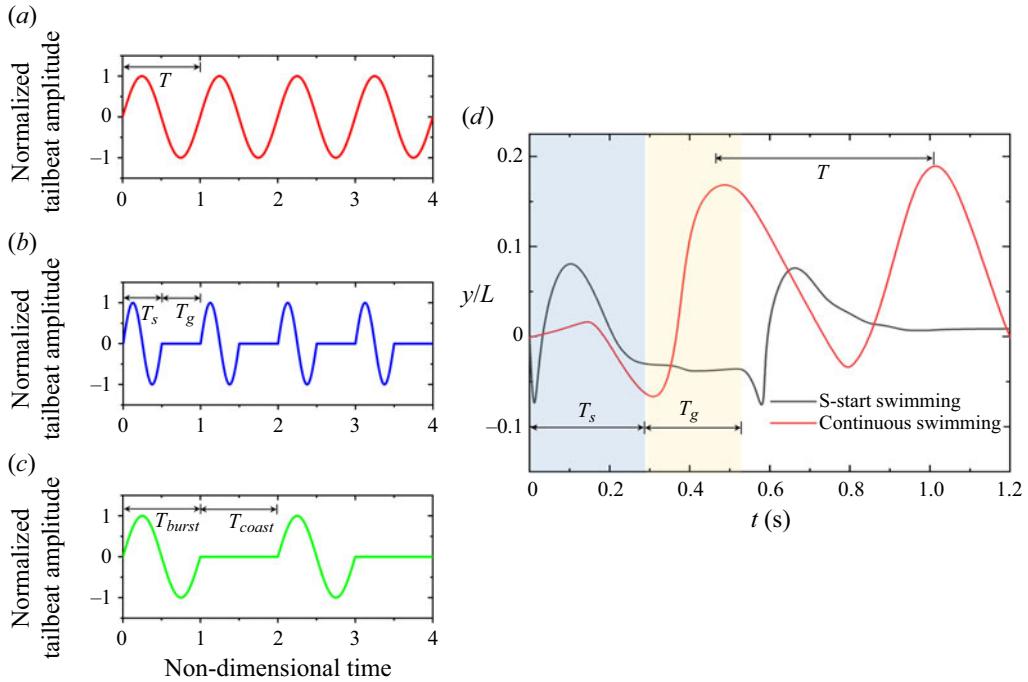


Figure 1. Kinematics of (a) continuous, (b) S-start, and (c) burst-and-coast swimming. (d) Real-time motion of the zebrafish tail tracked during S-start and continuous swimming. In (a,d), T denotes the time period of continuous swimming; T_s and T_g in (b,d) describe the time period of the S-start and gliding phase in the intermittent S-start swimming, respectively; T_{burst} and T_{coast} in (c) refer to the time period of the burst and coast phases in the burst-and-coast swimming, respectively. In (d), y is the time-dependent tailbeat amplitude, $L = 3$ cm is the body length, and t is time.

structures are generated by intermittent S-start swimmers? We attempt to address these questions by simulating numerically self-propelling foils undergoing intermittent S-start swimming. Despite the fact that we set out to determine scaling laws for intermittent S-start swimming, these scaling laws apply to S-start, B-and-C and continuous swimming as well.

2. Problem description and methodology

We consider a computational model in which a fish-like NACA0012 foil self-propels right to left in a rectangular domain. The foil can move freely in both horizontal x and vertical/lateral y directions (figure 2a). Figure 2(b) shows the foil's length as $L = 1$ cm, and its tailbeat amplitude as A . The intermittent S-start swimming is derived from the following kinematics:

$$y(x, t) = \begin{cases} S(t) A_m(x) \sin \left[2\pi \left(\frac{x}{\lambda} - \frac{t}{T_s} \right) - \frac{2\pi}{\lambda} \right], & 0 \leq t \leq T_s, \\ 0, & T_s < t \leq T, \end{cases} \quad (2.1)$$

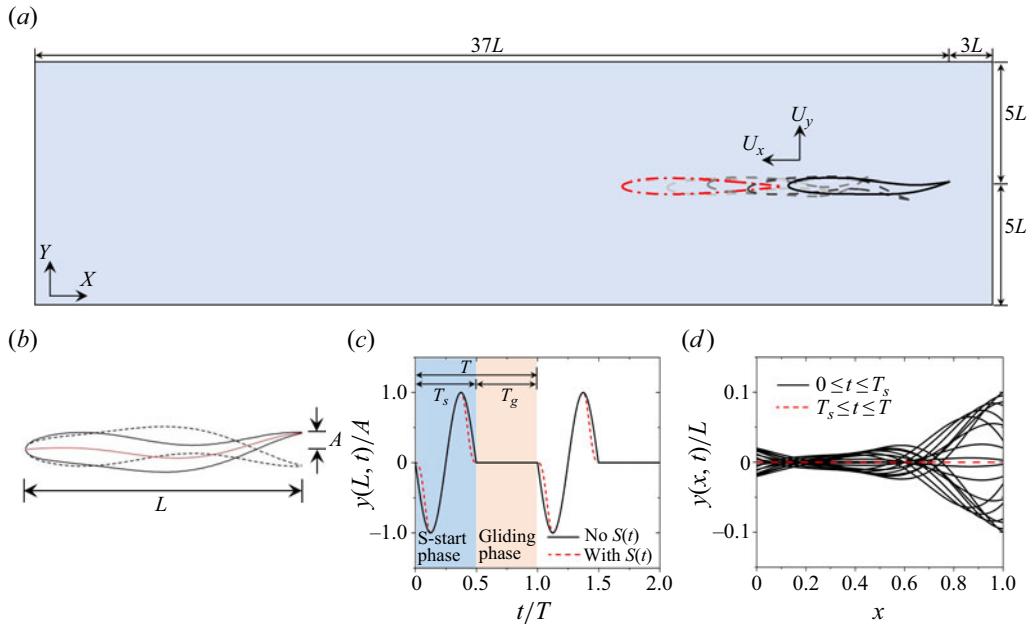


Figure 2. (a) Sketch of the computational domain; (b) NACA0012 foil; (c) intermittent S-start motion; and (d) foil's centreline envelope.

in which the smoothing function $S(t)$, given by

$$S(t) = \begin{cases} 0.5 \left[1 - \cos \left(\frac{4\pi t}{T_s} \right) \right], & 0 \leq t \leq 0.25T_s, \\ 1, & 0.25T_s < t \leq 0.75T_s, \\ 0.5 \left[1 - \cos \left(\frac{4\pi t}{T_s} \right) \right], & 0.75T_s < t \leq T_s, \end{cases} \quad (2.2)$$

is employed to avoid discontinuous accelerations at the junction of the S-start and gliding phase (figure 2c). In the equations above, $A_m(x) = a_0 \times (0.02 - 0.08x + 0.16x^2)$ denotes the amplitude function (Videler & Hess 1984) so that $A = A_m(L)$, t is the time, $\lambda = L$ is the wavelength of the travelling wave, T_s denotes the period of the S-start phase, and T is the total swimming period that includes both the S-start and the gliding phase. The envelope of the foil's centreline using (2.1) is shown in figure 2(d).

Three non-dimensional numbers are considered in this work: duty cycle (Akoz & Moored 2018)

$$DC = \frac{\text{S-start period}}{\text{total swimming period}} = \frac{T_s}{T}, \quad (2.3)$$

swimming number (Gazzola, Argentina & Mahadevan 2014)

$$Sw = \frac{2\pi L A}{T \nu}, \quad (2.4)$$

and energy consumption coefficient

$$C_E = \frac{CoT}{\rho \nu^2} \left(\frac{L}{W} \right). \quad (2.5)$$

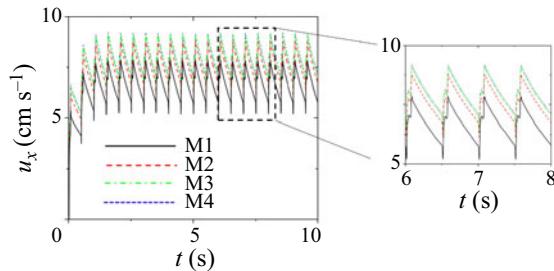


Figure 3. Grid convergence study using four grids, M1 to M4.

| Grid | $\Delta x/L$ | \bar{U}_x (cm s ⁻¹) | $\Delta \bar{U}_x$ (%) |
|------|-----------------------|-----------------------------------|------------------------|
| M1 | 5.00×10^{-3} | 6.6808 | 12.376 |
| M2 | 2.00×10^{-3} | 7.6244 | 4.016 |
| M3 | 1.25×10^{-3} | 7.9434 | 0.950 |
| M4 | 1.00×10^{-3} | 8.0196 | — |

Table 1. Grid convergence study with $\Delta t = 10^{-4}T$.

Here, $\nu = 0.0091$ poise is the fluid kinematic viscosity, $\rho = 1$ g cm⁻³ is the density of the fluid, and W is the width of the swimmer into the plane of the paper. In (2.5), *CoT* denotes the cost of transport, which describes the energy consumption of a self-propelling body. There are two common definitions of *CoT* – metabolic and mechanical. Here, we focus on the mechanical *CoT*, which is the ratio of time-averaged total power and time-averaged swimming speed (in the horizontal x direction) (Bale *et al.* 2014). The Reynolds number is defined as $Re = \bar{U}_x L / \nu$.

Numerical investigations are conducted using the open-source IBAMR software, which is a distributed-memory parallel implementation of the immersed boundary (IB) method that incorporates the Cartesian grid adaptive mesh refinement (AMR) technique (Griffith 2009; Griffith & Patankar 2020). The IBAMR software has been used extensively to study fish-like swimming (e.g. Bhalla, Griffith & Patankar 2013b; Tytell *et al.* 2016; Hoover *et al.* 2018). The computational domain is taken to be a rectangular box of size $40L \times 10L$ with periodic boundary conditions along the axial direction and no-slip boundary conditions in the lateral direction (figure 2a).

A grid convergence study is conducted for an intermittent S-start swimmer with $(DC, T, A/L, \lambda/L) = (0.2, 0.5$ s, $0.2, 1.0)$. Here, s denotes seconds. The simulations are conducted on four grids (M1 to M4) with uniform mesh spacings $\Delta x/L = \Delta y/L = 5.00 \times 10^{-3}$ (M1), 2.00×10^{-3} (M2), 1.25×10^{-3} (M3) and 1.00×10^{-3} (M4) in the finest level. The time step size is fixed at $\Delta t = 1.00 \times 10^{-4}T$. In figure 3, the time history of swimming velocity u_x is presented for the four grids. Table 1 lists the steady swimming speed \bar{U}_x . There is a relatively small difference between M3 and M4 in terms of \bar{U}_x (0.95%). For the remainder of the simulations, mesh M3 is selected.

Using M3, a time step size convergence study is performed by selecting four values of Δt : $\Delta t_1 = 5.00 \times 10^{-4}T$, $\Delta t_2 = 2.50 \times 10^{-4}T$, $\Delta t_3 = 1.00 \times 10^{-4}T$ and $\Delta t_4 = 0.50 \times 10^{-4}T$. Table 2 lists the corresponding \bar{U}_x values. The difference between \bar{U}_x using Δt_3 and Δt_4 is only 0.01%. Grid M3 and time step size Δt_3 are selected for the remainder of the simulations based on accuracy and computational resources. More convergence studies

| Time step size | $\Delta t/T$ | \bar{U}_x (cm s $^{-1}$) | $\Delta \bar{U}_x$ (%) |
|----------------|-----------------------|-----------------------------|------------------------|
| $\Delta t1$ | 5.00×10^{-4} | 7.9475 | 0.034 |
| $\Delta t2$ | 2.50×10^{-4} | 7.9448 | 0.018 |
| $\Delta t3$ | 1.00×10^{-4} | 7.9434 | 0.010 |
| $\Delta t4$ | 0.50×10^{-4} | 7.9426 | — |

Table 2. Time step size convergence study using grid M3 with $\Delta x/L = \Delta y/L = 1.25 \times 10^{-3}$.

| $\frac{Sw}{A/L}$ | T(s) | | | | | | |
|------------------|----------|----------|----------|----------|---------|---------|---------|
| | 0.50 | 0.75 | 1.00 | 1.25 | 1.50 | 1.75 | 2.00 |
| 0.05 | 6283.19 | 4188.79 | 3141.59 | 2513.27 | 2094.40 | 1795.20 | 1570.80 |
| 0.10 | 12566.37 | 8377.58 | 6283.19 | 5026.55 | 4188.79 | 3590.39 | 3141.59 |
| 0.15 | 18849.56 | 12566.37 | 9424.78 | 7539.82 | 6283.19 | 5385.59 | 4712.39 |
| 0.20 | 25132.74 | 16755.16 | 12566.37 | 10053.10 | 8377.58 | 7180.78 | 6283.19 |

Table 3. Sw values for various A and T .

related to the IB method can be found in our previous works (e.g. Bhalla *et al.* 2013a; Patel, Bhalla & Patankar 2018).

3. Results and discussions

The hydrodynamics of intermittent S-start swimming is investigated by measuring the swimming speed and the energy consumption associated with a self-propelling foil for $DC = 0.20\text{--}1.00$ with $\Delta DC = 0.20$, $T = 0.50\text{--}2.00$ s with $\Delta T = 0.25$ s, and $A = 0.05\text{--}0.20L$ with $\Delta A = 0.05L$. The calculated values of Sw corresponding to different pairs of A and T are listed in table 3. We discuss first hydrodynamic performance, then scaling laws, and finally classify the wake structures of an intermittent S-start swimmer. The trends of hydrodynamic performance (swimming speed and CoT) are plotted in dimensional units. This avoids confusion related to non-dimensionalization and ensures reproducibility. Eventually, hydrodynamic performance metrics are expressed non-dimensionally through scaling laws in § 3.2.

3.1. Hydrodynamic performance

Figure 4 illustrates how DC , T and A affect the steady time-averaged swimming speed \bar{U}_x of the foil. At a given A , \bar{U}_x decreases with increasing T and DC , with maximal and minimal values achieved at $(DC, T, A/L) = (0.20, 0.05\text{ s}, 0.20)$ and $(DC, T, A/L) = (1.00, 2.00\text{ s}, 0.05)$, respectively. Depending on DC , as much as 122.89 % to 541.29 % can be gained with intermittent S-start swimming ($DC < 1.0$) compared to the continuous swimming case ($DC = 1.0$); a smaller DC results in higher \bar{U}_x . For a given $T_s = T \times DC$, it can also be observed that the intermittent S-start swimming with a larger DC and smaller T produces higher \bar{U}_x than that with a smaller DC and larger T ; for example, $\bar{U}_x = 1.47$ cm s $^{-1}$ derived from $(DC, T, A/L) = (0.40, 0.50\text{ s}, 0.05)$ is larger than $\bar{U}_x = 1.02$ cm s $^{-1}$ for $(DC, T, A/L) = (0.20, 1.00\text{ s}, 0.05)$, as shown in figure 4(a). Due to a longer deceleration period in the gliding phase, a smaller DC may not be an optimal strategy for the intermittent S-start swimmer at a specific undulating period T_s . According

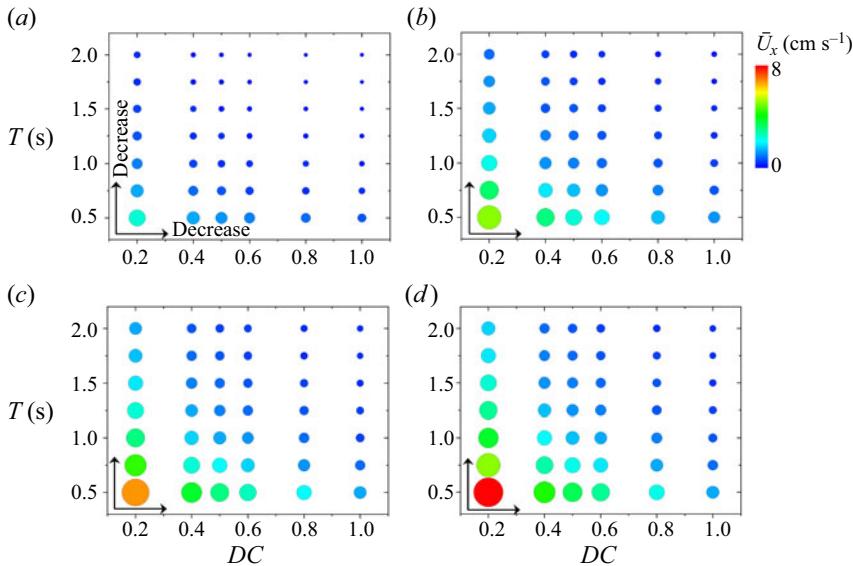


Figure 4. Influence of DC and T on \bar{U}_x with (a) $A = 0.05L$, (b) $A = 0.10L$, (c) $A = 0.15L$, and (d) $A = 0.20L$.

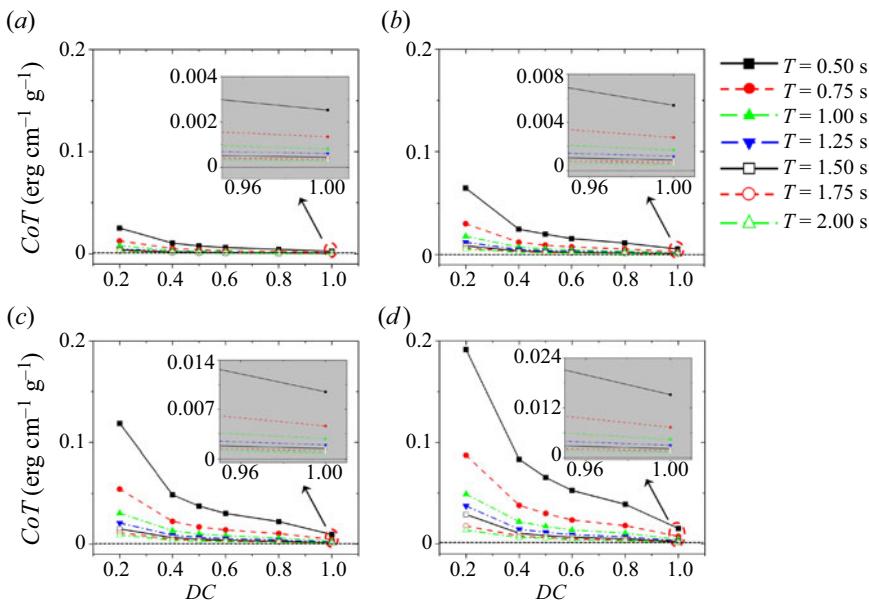


Figure 5. Influence of DC and T on CoT at (a) $A = 0.05L$, (b) $A = 0.10L$, (c) $A = 0.15L$, and (d) $A = 0.20L$.

to figure 4, a larger A corresponds to higher swimming speeds, which concurs with previous studies on oscillating, undulating and self-propelling foils (e.g. Floryan, Van Buren & Smits 2019). Overall, \bar{U}_x correlates strongly with A but inversely with DC and T .

The effects of DC , T and A on CoT are shown in figure 5, where CoT decreases with increasing DC and/or T , but it increases with increasing A . As discussed in Chao, Alam & Cheng (2022), an increase in tailbeat amplitude A requires more input power. A larger

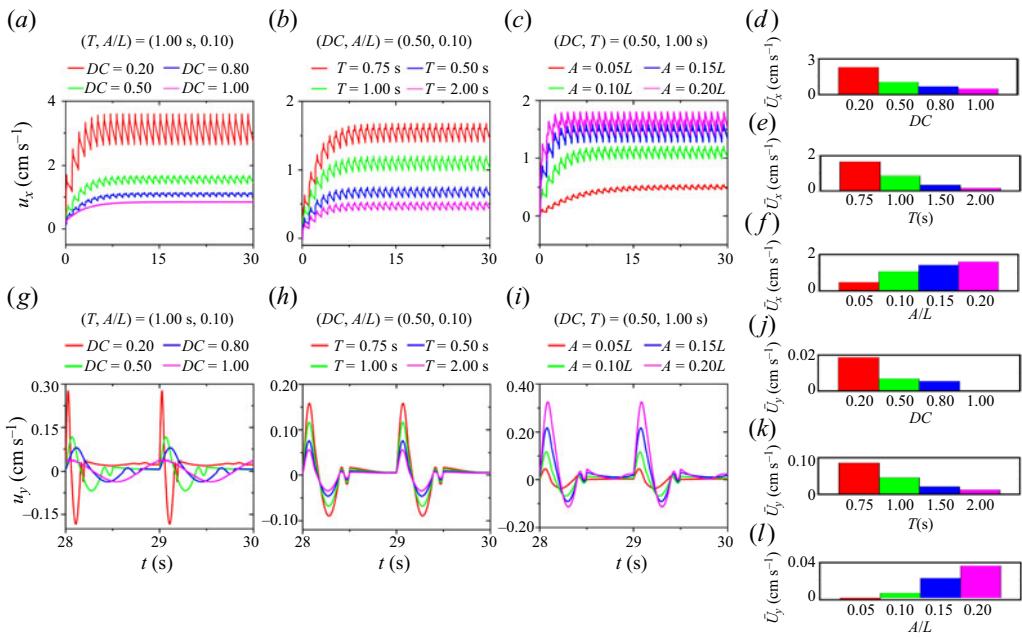


Figure 6. The effect of (a) DC , (b) T , and (c) A on u_x , and (d–f) the corresponding steady-state swimming velocities \bar{U}_x . The effect of (g) DC , (h) T , and (i) A on u_y , and (j–l) the corresponding steady-state lateral velocities \bar{U}_y .

value of A also hinders the flow passing around the foil. As a result, increasing A leads to inefficient swimming, i.e. higher CoT . An inverse relationship forms between CoT and DC at specific T and A . This is because a higher undulating frequency (smaller DC) requires more input power. Given a set of (DC, A) , the decline of CoT slows down with increasing T . It is desirable to have a lower CoT because it indicates lower energy consumption, thus greater mechanical efficiency. According to figure 5, intermittent S-start swimming results in inefficient propulsion. Floryan *et al.* (2019) found that at a given Strouhal number, the foil efficiency can be increased by increasing the trailing edge amplitude. However, this is not the case for the present self-propelling foil, where CoT increases with an increase in A .

Figure 6 illustrates how examined parameters DC , T and A affect the instantaneous swimming speed in horizontal and vertical directions. As expected, a smaller DC causes significant fluctuations in u_x and u_y at given (T, A) (figures 6a,g). A continuous swimming mode generates steady forward motion with fewer fluctuations, whereas an intermittent S-start gait leads to higher swimming speeds with larger fluctuations around the mean velocity. In a similar way, a smaller T and a larger A lead to more fluctuating u_x and u_y velocities (figures 6b,c,h,i). Interestingly, the u_x curve is stabilized easily with a higher A , whereas $A = 0.05L$ leads to a negative \bar{U}_y (figure 6l). This result suggests that an S-start swimmer may adopt smaller DC and T , but a larger A to achieve higher instantaneous and time-averaged speeds (figures 6a–f). However, it has been observed from biological experiments that the fish does not tend to produce large amplitudes when adopting an S-start motion. A plausible explanation is that larger tailbeat amplitudes produce high-speed flows (that could reveal the fish's whereabouts) and are costlier energetically. The decision-making behaviour behind this observation deserves further exploration.

3.2. Scaling laws applicable to intermittent S-start, B-and-C and continuous swimming

To better understand the effects of swimming parameters DC , T and A on \bar{U}_x and CoT (figures 4 and 5), scaling laws for swimming speed and efficiency are derived. In the previous subsection, we considered variations in both S-start and B-and-C swimming kinematics because T_s and T were varied independently. In terms of scaling laws, we do not distinguish explicitly between the two gaits, and refer to them collectively as S-start or intermittent motions. The scaling laws for hydrodynamic performance metrics are derived by balancing the work done by thrust and drag forces. We note that our starting point for deriving the scaling laws differs from previous studies on continuous swimming that consider force balance instead of energy balance. Due to the differences in duration and intensity of thrust and drag forces between S-start and glide phases of intermittent swimming, a force balance approach is not appropriate. Using the principle of energy balance, we can account explicitly for S-start and glide periods (T_s and T_g , respectively) in scaling laws. According to this, during steady swimming, the energy spent/work done by the thrust force during the S-start phase is used in overcoming the hydrodynamic drag throughout the swimming period. In steady state and over the course of a swimming cycle, the swimmer's kinetic energy remains the same. This is because kinetic energy gained through thrust forces is lost through drag forces. At the end of the cycle, the swimmer achieves the same swimming velocity as when it started (see figure 6). As a result, the kinetic energy term does not appear in the energy balance statement that follows next. We note that the scaling laws that we derive have an input versus output relationship. Inputs to the laws are the foil kinematic parameters, which are known beforehand and do not require computational fluid dynamics simulation. This is different from the velocity scale given in Akoz & Moored (2018) (for an intermittent swimmer), which has a mixed form – that is, it contains both input (foil kinematic parameters) and output (thrust force) quantities, depends on drag and thrust decomposition of the hydrodynamic force, and relies on potential flow theory.

3.2.1. Scaling laws based on overcoming the viscous drag

An intermittent swimmer undulates its body at an effective angular frequency $\omega^{eff} = \omega/DC = 2\pi/(DC \times T) = 2\pi/T_s$, with $DC < 1$. Here, $\omega = 2\pi/T$ denotes the angular frequency, T is the total swimming period, and T_s is the flapping time period. During steady swimming in a low to moderate Reynolds number flow regime, the thrust energy is expended overcoming viscous drag, i.e.

$$E_T \sim E_D. \quad (3.1)$$

Here, E_T (henceforth called thrust energy) represents the energy used or work done by thrust forces during the flapping period T_s , and E_D (henceforth called viscous drag energy) represents the energy spent during the entire swimming period, $T = T_s + T_g$, in overcoming viscous drag (figure 1b).

The thrust energy can be estimated through thrust force $F_T \sim \rho U_{lateral}^2 A_p$ as $E_T \sim F_T \bar{U}_x T_s$. Here, $U_{lateral} = A\omega/DC$ is the deformation velocity in the lateral direction, A is the tailbeat amplitude, and A_p is the projected area of the swimmer into the plane of the paper. Thus, $A_p \sim L \times W$, in which L is the length of the swimmer, and W is the width of the swimmer (into the plane of the paper). Gazzola *et al.* (2014) provide a geometric argument for the thrust force scale for undulatory swimmers. This scale has also been used in prior studies of aquatic locomotion (Bale *et al.* 2014; Floryan, Van Buren & Smits 2018; Floryan *et al.* 2019). The viscous drag energy can be estimated from the viscous

drag force F_D as $E_D \sim F_D \bar{U}_x T$. The viscous drag force can be estimated from the Blasius solution $F_D \sim \mu(\bar{U}_x/\delta)A_p$. Here, \bar{U}_x is the steady time-averaged swimming speed, and $\delta \sim L/\sqrt{Re}$ is the boundary layer thickness, with $Re = \bar{U}_x L / \nu$ the Reynolds number. By equating thrust to viscous drag work, we obtain

$$\left. \begin{aligned} E_T &\sim E_D, \\ \rho(A\omega/DC)^2 A_p \bar{U}_x T_s &\sim \frac{\mu \bar{U}_x A_p}{L} \left(\frac{\bar{U}_x L}{\nu} \right)^{1/2} \bar{U}_x (T_s + T_g) \\ \hookrightarrow \bar{U}_x &\sim (A\omega)^{4/3} (DC)^{-2/3} L^{1/3} \nu^{-1/3}, \end{aligned} \right\} \quad (3.2)$$

in which we used the definition $DC = T_s/(T_s + T_g)$. Equation (3.2) can be non-dimensionalized and rewritten succinctly as

$$Re \sim Sw^{4/3} / DC^{2/3}. \quad (3.3)$$

The scaling law for the cost of transport CoT can be derived using its definition and velocity scale (3.2) as

$$\left. \begin{aligned} CoT &= \frac{E_L}{\bar{U}_x T} \\ &\sim \frac{\rho(A\omega/DC)^3 A_p T_s}{(A\omega)^{4/3} (DC)^{-2/3} L^{1/3} \nu^{-1/3} T} \\ \hookrightarrow CoT &\sim \rho v^2 (W/L) (Sw)^{5/3} (DC)^{-4/3}, \end{aligned} \right\} \quad (3.4)$$

The numerator of the cost of transport E_L in (3.4) represents the energy spent or work done by the swimmer deforming its body in the lateral direction. Following Bale *et al.* (2014), the lateral power scales as $P_L \sim \rho U_{lateral}^3 A_p$. (According to Bale *et al.* (2014), most of the muscle work is done in producing lateral deformations. Consequently, $P_{muscle} \approx P_L$.) Consequently, work done in generating body deformations during the swimming period is $E_L = P_L T_s$. The denominator $\bar{U}_x T$ represents the distance travelled by the swimmer during the swimming period. Equation (3.4) can be non-dimensionalized and rewritten succinctly as

$$C_E = \frac{CoT}{\rho v^2} \left(\frac{L}{W} \right) \sim Sw^{5/3} / DC^{4/3}. \quad (3.5)$$

For a two-dimensional swimmer, a unit width is taken into the plane of the swimmer, i.e. $W = 1$.

Equations (3.3) and (3.5) provide a relationship between (non-dimensional) swimming speed Re and energy consumption C_E of the foil as a function of its key kinematic parameters Sw and DC . To see how well the above scaling laws fit the simulated data of the previous section, we plot the foil's measured outputs, Re and C_E , against its kinematic inputs, $Sw^{4/3} / DC^{2/3}$ and $Sw^{5/3} / DC^{4/3}$ in figures 7(a,b), respectively. It can be observed that the output data for different values of DC collapse well onto a single straight line, suggesting that (3.3) and (3.5) can be used as scaling laws to estimate the hydrodynamic performance of an intermittent S-start swimmer. Furthermore, these two scaling laws also explain the results of the previous section very well. For example, considering a swimmer of fixed length L and time period T , (3.3) suggests that a larger A (thus Sw) leads to a higher \bar{U}_x (figures 6c,f). Similarly, a smaller DC generates a higher (time-averaged)

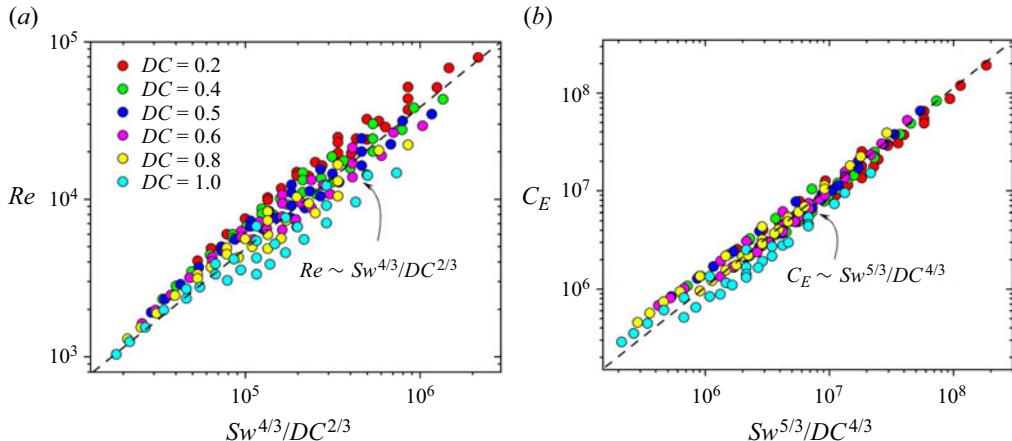


Figure 7. Scaling laws for (a) swimming speed Re , and (b) efficiency C_E , based on overcoming the viscous drag. The fitted dashed lines are $Re = 0.037 Sw^{4/3}/DC^{2/3}$ and $C_E = 1.032 Sw^{5/3}/DC^{4/3}$, and the coefficients of determination are $R^2 = 0.889$ and 0.987 for (a,b), respectively.

swimming speed \bar{U}_x (figures 6a,d). Equation (3.5) also shows that C_E is inversely related to DC , which confirms the hypothesis that intermittent S-start swimming ($DC < 1.0$) is less efficient than the continuous one ($DC = 1.0$).

3.2.2. Scaling laws based on overcoming the pressure drag

For high-speed flows, it can be argued that the thrust energy/work E_T is spent in overcoming the pressure drag F_P instead of the viscous drag F_D . For inviscid fluids, this is indeed the case. The pressure force scales as the square of the body's velocity, i.e. $F_P \sim \rho \bar{U}_x^2 A_p$. For a geometric argument for the F_P scale for undulatory swimmers, see Gazzola *et al.* (2014). Thus, by equating thrust to pressure drag work, we obtain

$$\left. \begin{aligned} E_T &\sim E_P, \\ \rho (A\omega/DC)^2 A_p \bar{U}_x T_s &\sim \rho \bar{U}_x^2 A_p \bar{U}_x (T_s + T_g) \\ \hookrightarrow \bar{U}_x &\sim A\omega/DC^{1/2}. \end{aligned} \right\} \quad (3.6)$$

Through non-dimensionalization, we obtain

$$Re \sim Sw/DC^{1/2}. \quad (3.7)$$

The corresponding CoT scale is obtained as

$$\left. \begin{aligned} CoT &= \frac{E_L}{\bar{U}_x T} \\ &\sim \frac{\rho (A\omega/DC)^3 A_p T_s}{A\omega (DC)^{-1/2} T} \\ \hookrightarrow CoT &\sim \rho v^2 (W/L) Sw^2 / DC^{3/2}. \end{aligned} \right\} \quad (3.8)$$

Equation (3.8) can be non-dimensionalized and rewritten as

$$C_E = \frac{CoT}{\rho v^2} \left(\frac{L}{W} \right) \sim Sw^2 / DC^{3/2}. \quad (3.9)$$

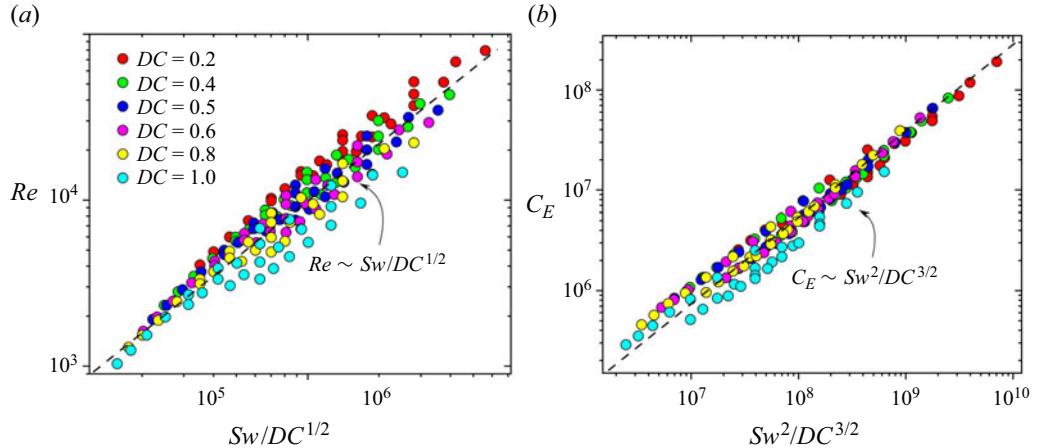


Figure 8. Scaling laws for (a) swimming speed Re , and (b) efficiency C_E , based on overcoming the pressure drag. The fitted dashed lines are $Re = 1.279Sw/DC^{1/2}$ and $C_E = 0.029Sw^2/DC^{3/2}$, and the coefficients of determination are $R^2 = 0.883$ and $R^2 = 0.980$ for (a,b), respectively.

Equations (3.7) and (3.9) are compatible with inviscid flows as the effect of $\nu \rightarrow 0$ is eliminated. According to figure 8, scaling laws based on overcoming the pressure drag are also able to fit the data reasonably well. We attribute this match to the moderate flow regime ($10^3 < Re < 10^5$) considered in this study. At very-low-speed ($Re \sim 1$) and very-high-speed ($Re > 10^6$) flows, we expect to see a more distinct trend between viscous-based and pressure-based scaling laws. At low Re , the fluid–structure interaction (FSI) equations are coupled strongly, while at high Re , explicit turbulence models are required to resolve the turbulent flow features. In our flow solver, we solve the FSI equations in a split manner without explicit turbulence models, which is suitable for simulating moderate- Re FSI cases. The two scaling laws will be tested in low- and high-speed flow regimes in a future study. Because we are considering moderate Re flows in this study, we prefer viscous scaling laws over pressure-based ones to describe the hydrodynamic performance of intermittent S-start swimmers (speed and efficiency). To validate whether the data statistically support the different scaling laws that have been presented, we have conducted t-tests on the scaling laws.

In order to check whether the data support the different scaling laws presented in this section, we performed t-tests on the data. We assume that the data fit power laws of the form $Re = c_1Sw^{\alpha_1}DC^{\alpha_2}$ and $C_E = c_2Sw^{\alpha_3}DC^{\alpha_4}$. Using ordinary least squares regression on the data, the coefficients obtained were $\alpha_1 = 0.9829$, $\alpha_2 = -0.8928$, $\alpha_3 = 1.7833$ and $\alpha_4 = -1.2453$. After that, two-sample t-tests were performed on the statistical values of Re and C_E , as well as their simulation values, at a significance level of 0.05. As expected, the means of the data sets were equal and the statistics converged to the simulation values. We accept the null hypothesis at significance level 0.05. Table 4 and table 5 present the least squares regression fit and two-sample t-test results, respectively.

3.2.3. Special cases of $DC \rightarrow 0$ and $DC \rightarrow 1$

The two cases $DC \rightarrow 0$ and $DC \rightarrow 1$ warrant separate discussion. The swimmer reaches $DC \rightarrow 0$ if: (i) it does not flap its body at all; (ii) its gliding period is much longer than its flapping period, i.e. $T_g \gg T_s$; or (iii) it flaps infinitely fast in an infinitely short period of time. The first two scenarios imply that $Sw \rightarrow 0$. It follows that the body's steady

| | α_1 | α_2 | α_3 | α_4 |
|----------------|------------|------------|------------|------------|
| Value | 0.9829 | -0.8928 | 1.7833 | -1.2453 |
| Standard error | 0.0173 | 0.0192 | 0.0159 | 0.0147 |

Table 4. Values and standard errors in $\{\alpha_i\}$ obtained using the ordinary least squares regression.

| | t-statistic | Degrees of freedom | Standard error | p-value |
|-------|-------------|--------------------|----------------|----------|
| Re | 0.025673 | 334 | 0.001405 | 0.979533 |
| C_E | 0.032521 | 334 | 0.001779 | 0.974076 |

Table 5. Results of t-tests between the simulation and statistical values of Re and C_E .

swimming velocity and energy expenditure should also approach zero when $DC \rightarrow 0$. The viscous and pressure scaling laws discussed in §§ 3.2.1 and 3.2.2 lead naturally to this conclusion. These laws are in the form Sw^m/DC^n , with $m > n$ and $m, n \in \mathbb{R}^+$. As $DC^n \rightarrow 0$, $Sw^m \rightarrow 0$ at a faster rate, which leads to the expected result. The third scenario is unphysical, as there is a practical limitation to how rapidly a body can flap. For example, Sanchez-Rodriguez, Raufaste & Argentina (2023) mentioned that an undulatory swimmer's tailbeat frequency is generally less than 20 Hz. Also, a very fast oscillation in a very short amount of time breaks the local thermodynamic equilibrium assumption of fluid mechanics, which requires that flow quantities (velocity, pressure) change at a reasonable rate (Jakobsen 2008). Thus scenario (iii) is not considered in the scaling laws derived in this work. (Moreover, it seems impossible to obtain experimental or simulation data to validate some candidate scaling law in this situation.)

The other end of the DC range, $DC \rightarrow 1$, implies continuous swimming. It can be observed that as $DC \rightarrow 1$, the scaling laws derived for the inertial flow regime reduce to $Re \sim Sw$ and $C_E \sim Sw^2$, whereas those obtained for the viscous flow regime reduce to $Re \sim Sw^{4/3}$ and $C_E \sim Sw^{5/3}$. For both flow regimes, our results are consistent with the scaling laws for steady, continuous swimming velocity derived in Gazzola *et al.* (2014).

3.3. Wake structures

In figure 9(a), six different flow patterns are illustrated, including 2P (\diamond), 2P+S (\square), 2S (\blacksquare), Mode I (\circ), Mode II (\triangle) and Mode III wake (∇). At smaller DC and Sw , i.e. the bottom left corner in the DC – Sw plane, two vortex pairs (2P) are generated by the S-start phase, and a single vortex (S) is generated by the boundary layer separation in the gliding phase (figure 9b). Increasing DC decreases the time lag between adjacent S-start phases, thereby suppressing boundary layer separation. Consequently, only two vortex pairs are observed as 2P wakes (figure 9c) when $Sw \leq 3000$ and $0.5 \leq DC < 1.0$. Further, for the 2P+S wake, the single vortex located in the middle of two vortex pairs gradually approaches and merges with the upper vortex pair (figure 9b). In the far field, therefore, the 2P+S wake would eventually become a 2P wake. Figure 9(a) illustrates a bifurcation caused by the 2P+S and 2P wake behind the foil at smaller Sw . A typical 2S reverse Kármán vortex street appears at $DC = 1.0$ (figure 9d).

By increasing Sw (by increasing either A or T), a blurry flow pattern, i.e. Mode I and Mode II wakes, appears gradually at $DC \leq 0.5$. There are four types of vortex structures

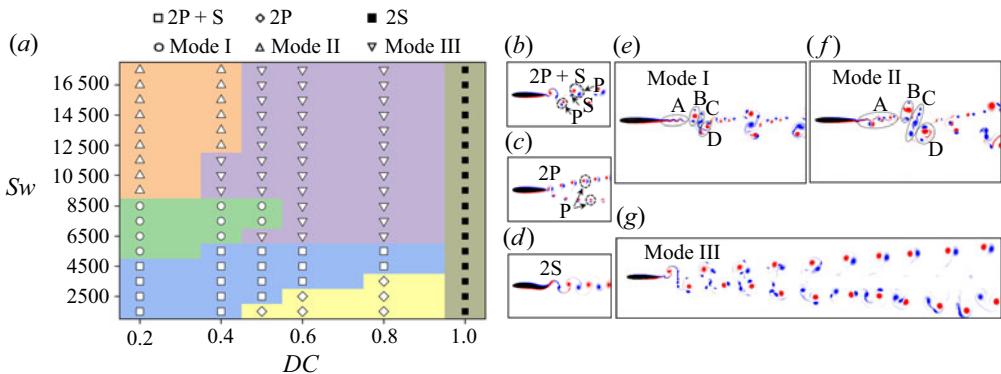


Figure 9. (a) Flow patterns in the DC - Sw plane. (b–g) Typical wake structures. Red and blue denote positive and negative vorticity, respectively.

in the Mode I and Mode II wakes: A, B, C and D (figures 9e,f). The vortex structure A corresponds to the boundary layer separation in the gliding phase, forming nS and mP+nS structures in the Mode I and Mode II wakes, respectively (Williamson & Roshko 1988). The vortex structures B, C and D are generated during the S-start phase. The vortex structure B in the Mode I and Mode II wakes is similar, where a strong positive vortex is surrounded by two weak negative vortices. The vortex structure C contains several negative vortices, with the Mode II wake providing more single vortex structures compared to the Mode I wake; see figure 9(f). The blurry vortex structure D describes the interaction between vorticity produced by the S-start phase and the one produced by the gliding phase. The evolution of Mode I and Mode II wakes due to the intermittent motion of a pitching foil has also been reported in Akoz *et al.* (2019), where it is mentioned that vortex structures B and D increase the swimming velocity, whereas structure C causes a deceleration. The Mode III wake occupies a large portion of the DC - Sw plane. After a series of vortex mergers, the Mode III wake eventually becomes a 2P wake, as shown in figure 9(g).

4. Conclusions

In this work, intermittent S-start swimming is studied in a systematic manner. The numerical study reveals that the time-averaged swimming velocity \bar{U}_x and the cost of transport CoT increase with decreasing duty cycle DC . This suggests that although intermittent S-start swimming leads to higher swimming speeds (which is good for both escaping predators and striking prey), it is an inefficient swimming gait compared to continuous cruising. We also presented two scaling laws to characterize the hydrodynamic performance of S-start swimmers, $Re \sim Sw^{4/3}/DC^{2/3}$ and $C_E \sim Sw^{5/3}/DC^{4/3}$, which are suitable for moderate speed flows ($10^3 < Re < 10^5$) considered in this work. During swimming, hydrodynamic performance can be measured by Re and C_E , which represent swimming speed and energy consumption. We can use Sw and DC to measure kinematic inputs for the swimmer. At high $Re > 10^6$, we expect the hydrodynamic performance metrics to scale as $Re \sim Sw/DC^{1/2}$ and $C_E \sim Sw^2/DC^{3/2}$, although this needs to be verified. Additionally, we also classified the wake structures of self-propelling intermittent S-start swimmers in this work. Finally, we note that the plots relating to scaling results appear to be stratified according to DC : points with higher DC values lie beneath points with lower DC values. A similar stratification is also found in the work of Akoz & Moored

(2018) (see their figure 10). Perhaps the scaling relations do not quite capture DC effects, and this motivates further investigation.

Supplementary movie. A supplementary movie is available at <https://doi.org/10.1017/jfm.2024.103>.

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