

Optimization of Borehole Thermal Energy Storage Systems Using Genetic Algorithm

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Abstract Borehole Thermal Energy Storage (BTES) represents cutting-edge technology harnessing the Earth's subsurface to store and extract thermal energy for heating and cooling purposes. Achieving optimal performance in BTES systems relies heavily on selecting the right operational parameters. Among these parameters, charging and discharging flow rates play a significant role in determining the amount of heat that can be recovered effectively from the system. In this study, we introduce a Genetic Algorithm as an optimization tool aimed at fine-tuning these operational parameters within a baseline BTES model. The BTES model was developed using FEFLOW and simulated over a 3-year period. After each 3-year simulation, the Genetic Algorithm iteratively adjusted the operational parameters to attain the optimal configuration for maximizing heat recovery from the BTES system. Additional analysis was conducted to explore the impact of BTES system size and borehole spacing on heat recovery. Results indicate that the Genetic Algorithm effectively optimized parameters, leading to enhanced heat recovery efficiency. Moreover, the [scenario studies](#) highlighted that closer borehole spacing correlates with higher recovery efficiency.

Keywords Genetic Algorithm · Optimization · Borehole thermal energy storage · Recovery efficiency

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1 1 Introduction

2 Various countries have set targets for achieving net-zero carbon emissions. In the United States, Canada,
3 the UK, and the European Union, a target has been set to achieve net-zero carbon emissions and other
4 greenhouse gases by 2050 (Brown et al. 2023a). However, the dominant role of non-renewable energy such
5 as fossil fuel in meeting the world's energy demands has been a historical constant. Renewable sources of
6 energy are an alternative that will help reduce the growing consumption of fossil fuels. Particularly within
7 the heating sector, excess thermal energy can be temporarily stored and used during peak demand periods
8 (Lanahan and Tabares-Velasco 2017).

9 Thermal Energy Storage (TES) is a technology that has the ability to store extra energy during times
10 of abundance and release it during peaks in demand (Tamme et al. 2012). This makes it a versatile and
11 scalable solution that improves efficiency and dependability. During times of high energy production or
12 in the summer period, excess heat can be captured by solar collectors, waste heat can be recovered from
13 industrial operations, and heat can be captured from various homes. After being captured, this heat is
14 stored in the subsurface for later use. This thermal energy can be released later to heat homes, especially
15 in the cold seasons, and to power turbines. Thermal energy storage can be performed by exploiting the
16 heat in different ways (Zhang 2016); thermo-chemical storage which utilizes reversible chemical reactions
17 to store and release thermal energy, latent heat storage which stores the energy through phase changes,
18 and sensible heat storage which works by increasing/decreasing the temperature of materials such as water,
19 thermal oils, molten salts, or subsurface geological media.

20 The subsurface is increasingly being utilized as a reservoir for storing thermal energy, which is referred to
21 as Underground Thermal Energy Storage (UTES) (Aktaş and Kirçiçek 2021). UTES has been exploited in
22 many areas on both large and small scales. Large scale includes commercial and industrial sectors helping
23 to power turbines as well as heating and cooling buildings. It is used on a small scale as a medium to
24 extract heat during the cold season and cold during the hot periods. During the summer, excess heat is
25 stored underground and retrieved in winter for heating, and vice versa for cooling in the summer. This
26 method helps balance seasonal variations in energy demand, contributing to grid stability. UTES uses a
27 variety of storage media (Casasso et al. 2022), including deep aquifers confined by impermeable geological
28 layers for Aquifer Thermal Energy Storage (ATES), and subsurface rock environments accessed through
29 Borehole Heat Exchangers (BHE) for Borehole Thermal Energy Storage (BTES). ATES became popular
30 for storing and retrieving thermal energy both in small and large quantities and also provided a reliable
31 source of thermal energy due to the stable temperature of aquifers (Possemiers et al. 2014). A major setback
32 ATES systems face is unfavorable hydrogeologic conditions (Shi et al. 2023). Not all geological environments
33 possess suitable aquifers for efficient energy storage, making it difficult to store energy through an ATES
34 system. An alternative option to use is a BTES system. BTES is a seasonal storage system that stores heat

35 in soil or rock environment during the summer months through a BHE embedded in the drilled holes to a
36 desired depth. The stored heat is extracted to meet the heating demands in the winter heating season. A
37 BTES system may consist of a single borehole with a coaxial BHE (Xie et al. 2018; Brown et al. 2023b),
38 where fluid is circulated down the annular space, exchanging heat with the subsurface via conduction
39 through the borehole wall before being circulated back to the surface, or an array of boreholes where heat
40 transfer fluid circulates through the BHEs installed in the boreholes to exchange heat with the surrounding
41 rock or sediment mass (Pourahmadiyan et al. 2023). After drilling the boreholes, a U-Tube or coaxial pipe
42 is installed in the borehole, allowing the fluid to continuously circulate from the bottom of the borehole
43 to the surface. Once the piping is fitted, the borehole may be left un-grouted and filled with groundwater,
44 as is common in Scandinavia (e.g., Sweden and Norway) where boreholes are typically drilled in hard rock
45 with the groundwater table a few meters below ground surface (Gehlin 2002), or grout may be poured into
46 the remaining volume of the drilled borehole to provide structural support and improve heat conductivity
47 between the earth and the heat transfer pipes.

48 While BTES systems offer a sustainable and efficient solution for storing and retrieving thermal energy,
49 one significant area of research is to obtain optimal recovery efficiency (e.g., Woloszyn 2018; Casasso and
50 Sethi 2014). The ratio of the total thermal energy recovered from the subsurface storage system to the total
51 thermal energy injected during a yearly cycle is known as annual efficiency, which also can be termed as
52 recovery efficiency or round trip efficiency. Therefore, optimizing the BTES system is crucial to achieving
53 optimum energy efficiency. This is important because it reduces operational costs associated with heating
54 and cooling, ensures the long-term sustainability of the BTES system by preventing degradation, wear,
55 and inefficiencies over time, and contributes to the reduction of greenhouse gas emissions by promoting
56 the use of renewable energy sources. For this area of renewable energy where optimization is vital to
57 improve energy efficiency, published work reporting on optimization of the BTES systems is very scanty
58 and optimization has been done mostly by calibration of parameters of the BTES system. For example,
59 Rapantova et al. (2016) utilized a calibrated numerical model (FEFLOW finite element model) to simulate
60 various heat injection and extraction cycles. They conducted calibration using data from six monitoring
61 boreholes, measuring temperatures at depths up to 80 meters underground, with the aim of minimizing
62 heat loss through dissipation to the ground. Kumawat et al. (2024) conducted sensitivity studies to evaluate
63 BTES system performance across a range of operational, design, and geological parameters. Fiorentini
64 and Baldini (2021) focused on operational optimization within a framework designed to identify optimal
65 operating conditions for heat pump-driven BTES systems under different electricity CO₂ intensity profiles.
66 Their work introduced a novel linearized control-oriented model describing storage temperature dynamics
67 under varying operational scenarios. Zhu et al. (2019) explored relationships between input parameters and
68 output indicators through global sensitivity analysis coupled with 3D transient numerical methods, revealing
69 significant interactions among different input variables influencing BTES performance. Disadvantages of

70 manual calibration are the time cost of obtaining the optimal parameters used in the design of the BTES
71 system and the uncertainty that the achieved optimum recovery of heat after manual calibration is indeed
72 the true recovery capacity of a particular BTES system.

73 In order to successfully navigate the complexity involved in improving BTES systems, integration of
74 modern optimization techniques is essential. Metaheuristic algorithms have been introduced previously to
75 find optimal and near-optimal solutions for complex optimization problems. One optimization algorithm that
76 falls under the umbrella of the metaheuristic algorithms is particle swarm optimization (PSO), introduced
77 by Kennedy and Eberhart (1995) drawing inspiration from the collective behavior of birds and fish. PSO
78 involves three main controlling parameters—*inertia weight*, *cognitive ratio*, and *social ratio*—where even a
79 slight adjustment can lead to different performances, as demonstrated by Eltamaly et al. (2020) and Harrison
80 et al. (2018). PSO also has some limitations; with regard to complex and large datasets, it produces poor
81 results. When faced with a considerable number of dimensions in the given problem, PSO often struggles
82 to identify the global optimum solution. This issue arises not only from the entrapment in local optima
83 but also from the potential variation in particle velocities, restricting the successive range of attempts to a
84 subset of the overall search space (Pant et al. 2009; Gad 2022).

85 Another type of metaheuristic algorithm is simulated annealing (SA). It stands out as a versatile and
86 potent optimization technique, particularly adept at identifying global optima amidst numerous local optima.
87 The term “annealing” draws an analogy from thermodynamics, specifically the cooling and annealing
88 process observed in metals (Press and Teukolsky 1991). In simulated annealing, the optimization problem’s
89 objective function is employed, instead of using material energy in the metallurgical context. SA is
90 recognized as a straightforward yet highly effective metaheuristic approach in solving global optima problems,
91 especially where the objective function is not explicitly defined and can only be assessed through
92 computationally intensive simulations (Delahaye et al. 2019).

93 Genetic Algorithm (GA) (Holland 1975) is another type of metaheuristic algorithms that emerged as
94 a cutting-edge methodology to navigate the complexity inherent in optimizing BTES systems. It stands
95 as one of the oldest and most widely recognized optimization techniques inspired by natural processes.
96 Within the framework of GA, the exploration of solution space mimics the natural occurrences observed in
97 the environment, incorporating principles from Darwinian theory of species evolution (Slowik and Kwas-
98 nicka 2020). GAs offer several advantages over conventional optimization methods, with two particularly
99 noteworthy strengths: their capability to handle complex problems and their inherent parallelism (Yang
100 2021). GAs exhibit versatility in addressing diverse optimization types, accommodating objective functions
101 that are either stationary or non-stationary (changing over time), linear or nonlinear, continuous or dis-
102 continuous, and those affected by random noise. The parallelism inherent in genetic algorithms stems from
103 multiple offspring within a population acting as independent agents, enabling the exploration of the search
104 space in numerous directions simultaneously. **GAs employ three key operators during the optimization**

105 process: selection, crossover, and mutation. The selection process involves selecting the best individuals
106 from the current generations as parents to form a mating pool for producing offspring for the next genera-
107 tion. Individuals are selected based on their fitness, which is determined by their performance with respect
108 to the objective function. Individuals with higher fitness are selected to form the mating pool. Selection
109 mimics the natural selection process, where individuals with better adaptability are more likely to pass
110 their genetic information to the next generation. After forming the mating pool, a crossover operation is
111 implemented. Crossover involves combining genetic information from two parents to produce offspring with
112 a mix of their traits, thereby exploring new regions of the solution space. Parts of the genetic informa-
113 tion (chromosomes) from two parents are exchanged, creating one or more offspring. Common methods
114 include one-point crossover, two-point crossover, and uniform crossover. Crossover introduces diversity in
115 the population, allowing the algorithm to exploit the beneficial combinations of traits present in the parent
116 individuals. Mutation introduces random changes in the genetic information of individuals, preventing the
117 algorithm from getting stuck in local optima and promoting exploration of the search space. A percentage of
118 genes in an individual's chromosome is randomly changed. The percentage is generally termed as mutation
119 rate which determines the likelihood of a gene being mutated.

120 Optimizing a BTES system entails addressing numerous parameters, including borehole spacing, op-
121 erational parameters such as the flow rates, design parameters such as the thermal conductivity of the
122 grouting material, and other factors. The challenge lies in finding a configuration that maximizes energy
123 storage and recovery, minimizes losses, and adapts to varying thermal demands. GA excel in tackling such
124 multidimensional optimization problems, offering a holistic approach to fine-tuning the intricate parameters
125 of BTES systems. In this study, to be best of our knowledge, GA will be employed for the first time to
126 search for two optimal operational parameters (charging and discharging flow rate) that emerge as critical
127 operating parameters with substantial implications for the performance of a BTES system.

128 2 Methodology

129 2.1 Optimization of parameters

130 In this study, three categories of parameters that influence the recovery efficiency of a BTES system were
131 considered: (a) design, (b) operational, and (c) geological parameters. Design parameters, such as BHE
132 spacing and grout thermal conductivity, influence heat transfer rates, thereby affecting system efficiency.
133 Operational parameters, such as charging and discharging volumetric flow rate, are directly related to
134 energy storage and recovery. Geological parameters consider the thermal conductivity of the ground.

135 The separation distance between BHEs in a BTES model plays a crucial role in determining the system's
136 effectiveness. If the distance is too small, thermal interference can occur between the boreholes, impacting
137 overall system efficiency. This interference involves the heat transfer fluid from one borehole affecting the

138 surrounding ground of another, diminishing the system's overall effectiveness. Conversely, if the spacing is
139 too large, it can result in increased costs and reduced efficiency. Larger land areas would be needed for the
140 same thermal energy storage, and the heat transfer fluid would have to travel a greater distance, requiring
141 more energy for circulation. Consequently, the optimal spacing between two BHEs in a BTES model must
142 be meticulously considered to optimize the system's performance.

143 The thermal conductivity of the grout holds significant importance in the context of a BTES model. It
144 dictates the speed at which heat exchange occurs between the ground and the heat transfer fluid circulating
145 within the borehole. Grout is employed to fill the annular space between the borehole wall and the heat
146 exchanger pipes, influencing the heat transfer between the ground and the heat transfer fluid. In cases
147 where the grout exhibits low thermal conductivity, it acts as a thermal barrier, impeding heat transfer
148 and diminishing the efficiency of the BTES system. Conversely, high thermal conductivity in the grout
149 enhances heat transfer, thereby increasing the overall system efficiency. Consequently, the careful selection
150 of a grout material with appropriate thermal conductivity is pivotal for optimizing the performance of a
151 BTES system.

152 The flow rate during charging and discharging operations emerges as a critical operating parameter with
153 substantial implications for the performance of a BTES system. In the charging phase, the heat transfer
154 fluid circulates through the BTES to store heat in the nearby ground, and the flow rate influences the
155 efficiency of heat transfer from the working fluid to the BTES system. Conversely, in the discharging phase,
156 the heat transfer fluid circulates through the BHEs to release stored heat into a building's heating system.
157 The flow rate during discharging determines the rate at which stored heat is released into a building. While
158 a higher flow rate accelerates heat transfer, impacting the charging and discharging phases, it also could lead
159 to elevated pressure drop, potentially affecting energy consumption and reducing system efficiency. Hence,
160 obtaining an optimal flow rate for both charging and discharging becomes crucial, striking a balance between
161 efficient heat transfer and the energy consumption of the system.

162 The thermal conductivity of solids determines how easily heat can be transferred between the ground
163 and the wall of the borehole. The rate of heat transfer between the borehole wall and the surrounding
164 geological material is directly influenced by the thermal conductivity of the solid. A geological environment
165 with high thermal conductivity facilitates easier heat transfer compared to an environment with low thermal
166 conductivity. Understanding the thermal properties of the subsurface geologic properties is very crucial for
167 attaining the highest recovery efficiency of the BTES system.

168 2.2 Finite element model (FEM)

169 The FEFLOW software ([Diersch 2005](#)), which is a finite element modeling tool, was employed to simulate
170 and compute temperature variations in the subsurface while transferring heat during charging and

171 discharging. The temperature changes in the ground around the borehole can be expressed through the
 172 following equation:

$$\rho \cdot c \cdot \frac{\partial T}{\partial t} = \nabla \cdot (k \cdot \nabla T) + Q \quad (1)$$

173 where ρ (kg/m^3) is the density of the ground, c ($J/(kg * K)$) is the specific heat capacity of the ground, and
 174 T (0C) is the temperature, t is time, k ($W/(m * K)$) is the thermal conductivity of the ground, Q (J/s) is
 175 the heat source and sink in the system.

176 The study considered a BTES system with a circular configuration, primarily because of the favorable
 177 packing density it provides for boreholes within a designated area. This allows for the efficient installation
 178 of more boreholes in a given space, thereby optimizing the overall thermal energy storage capacity of the
 179 system (Skarphagen et al. 2019). To streamline our model, we focused on one quadrant of the circular
 180 BTES system, as shown in Figure 1.

181 The FEM model employed is a homogeneous system with a depth of 30 m, consisting of 6 layers with
 182 a vertical discretization of 5 m. The modeling of BHEs employs a quasi-stationary computational method
 183 based on the work by Eskilson and Claesson (1988). This approach is advantageous for long-term simulations
 184 spanning hours or longer, particularly when inflow temperature changes are less frequent and less steep. The
 185 quasi-stationary method offers reasonable accuracy at a lower computational cost. The analytical solution
 186 assumes local thermal equilibrium between all elements of the BHE (pipes, grout, ground) at any point
 187 during the simulation. The model considers a configuration of double U-shaped BHEs connected in parallel.
 188 The borehole diameter is 12 cm, with a pipe separation of 4 cm between each U-shaped BHE. The pipes
 189 have a diameter of 3.2 cm, and the pipe walls are 0.29 cm thick. Figure 2 shows the cross-section of the
 190 borehole heat exchanger. The double U-shaped BHE comprises two pipes with fluid flowing in opposite
 191 directions, providing an increased contact area for enhanced heat transfer. The initial temperature of the
 192 BTES model before simulation was $10^{\circ}C$. The base case model parameters are listed in table 1 below:

Table 1 BTES model parameters

Parameter	Value
Thermal conductivity of solid ($J/m/s/K$)	3
Thermal conductivity of liquid ($J/m/s/K$)	0.65
Volumetric heat capacity of solid ($MJ/m^3/K$)	2.52
Volumetric heat capacity of liquid ($MJ/m^3/K$)	4.2
Grout thermal conductivity ($J/m/s/K$)	3
Inlet pipe thermal conductivity ($J/m/s/K$)	0.42
Outlet pipe thermal conductivity ($J/m/s/K$)	0.42
Inlet temperature during injection (0C)	45
Inlet temperature during recovery (0C)	10
Flow rate during charging/discharging (m^3/day)	20

193 2.3 Genetic Algorithm

194 The implementation of the genetic algorithm is demonstrated in the step below. The algorithm goes through
195 a series of generations to obtain optimal results:

196 Step 1: Generate population:

197 The initial population consists of random candidate solutions (chromosomes) aimed at optimizing recov-
198 ery efficiency. Each chromosome represents a potential solution for achieving the highest recovery efficiency.
199 In this context, each chromosome is defined by two real numbers: the first number represents the flow rate
200 during charging, and the second number represents the flow rate during discharging.

201 **Details of Generation:**

202 – Initialization: An initial set of chromosomes was randomly generated.
203 – Population Size: The population size was set to five, balancing computational complexity with the need
204 for diversity and exploration in the solution space.

$$\mathbf{New}_{pop} = \begin{bmatrix} \mathbf{p}_{1,1} & \mathbf{p}_{1,2} \\ \mathbf{p}_{2,1} & \mathbf{p}_{2,2} \\ \vdots & \vdots \\ \mathbf{p}_{m,1} & \mathbf{p}_{m,2} \end{bmatrix} \quad (2)$$

205 where p represents the model parameters to be inserted into the FEFLOW model in the fitness function.
206 Each row is a set of parameters also known as chromosomes. p_m represents the m_{th} parameter in the m_{th}
207 chromosome. For this study, the first parameter in each row represents the charging flow rate and the
208 second parameter is the discharging flow rate.

209 Step 2: Compute the fitness function:

$$Fitness\ function = f(\mathbf{New}_{pop}) \quad (3)$$

210 where $f(.)$ represents the fitness function. In the fitness function, each chromosome is first inserted into
211 the FEFLOW model. Subsequently, following the model's execution, we calculate the recovery efficiency,
212 $RE = \frac{\nabla H_{recovered}}{\nabla H_{stored}} \times 100\%$ as our objective function. The objective is to maximize this function, which is
213 the RE.

214 Step 3: Selection of parents:

215 In this step, parents (chromosomes) are selected based on their fitness scores to ensure that superior
216 solutions have a higher likelihood of passing their genetic material to the next generation. Consequently, in
217 this step, the chromosome with the lowest RE is excluded, while the remaining chromosomes are retained
218 as parents to generate offspring for the next generation.

219 **Selection Method:** The tournament selection method was employed for this selection process. This
 220 method involves randomly choosing a subset (tournament size) of chromosomes from the population and
 221 selecting the fittest chromosome from this subset as a parent. Tournament selection provides a balanced
 222 approach that is robust to noise and outliers in fitness values.

223 **Process:**

224 – Randomly select a fixed number of chromosomes (tournament size) from the population.
 225 – Evaluate their fitness and select the chromosome with the highest fitness as a parent.
 226 – Repeat the process to select multiple parents for crossover.

$$\text{Parent} = \begin{bmatrix} \mathbf{p}_{1,1} & \mathbf{p}_{1,2} \\ \mathbf{p}_{2,1} & \mathbf{p}_{2,2} \\ \vdots & \vdots \\ \mathbf{p}_{m-1,1} & \mathbf{p}_{m-1,2} \end{bmatrix} \quad (4)$$

227 Step 4: Crossover function:

228 After selecting the parents, the chromosome with the highest RE replaces the eliminated chromosome
 229 to maintain the original size of the population. Subsequently, the crossover function is employed to merge
 230 two chromosomes, exchanging portions of parameters between them. The population generated after the
 231 crossover function is termed the offspring.

232 **Process:**

233 **Crossover Points:** In this case, a single-point crossover was employed because the chromosome contains
 234 only two values, making the crossover point the last value in the chromosome.

235 **Offspring Creation:** Apply crossover to pairs of selected parent chromosomes to generate offspring
 236 that inherit genetic information from both parents.

$$\text{Offspring} = \begin{bmatrix} \mathbf{p}_{1,1}^* & \mathbf{p}_{1,2}^* \\ \mathbf{p}_{2,1}^* & \mathbf{p}_{2,2}^* \\ \vdots & \vdots \\ \mathbf{p}_{m,1}^* & \mathbf{p}_{m,2}^* \end{bmatrix} \quad (5)$$

238 Step 5: Mutation:

239 In the final stages of the genetic algorithm's implementation, the mutation step is applied. During
 240 this step, all chromosomes are selected for mutation. Genes within each chromosome are chosen randomly,
 241 ensuring that each gene has an equal chance of being mutated.

242 **3 Synthetic Cases**

243 In this study, a genetic algorithm is employed to optimize the two operational parameters, while the grout
 244 thermal conductivity and the thermal conductivity of the ground remains fixed. This study considered two
 245 BHE spacings, a 2.5 m spacing and a 10 m spacing. The genetic algorithm ran for 30 generations because,
 246 following a pilot experiment, there was a considerable reduction in recovery efficiency differences from the
 247 25th generation to the 30th generation. The objective was to explore the efficacy of mathematical algorithms
 248 in obtaining optimal parameters for a BTES system design. Algorithm 1 shows the process of implementing
 249 the optimization. During the optimization process, three scenarios were considered, as shown in Table 2.
 250 These scenarios are presented to analyze the impact of the number of boreholes and their spacing on the
 251 overall heat recovery efficiency. The implementation of the genetic algorithm for the optimization process
 252 is illustrated in algorithm 1.

Table 2 Scenarios studies.

Scenarios	BHE spacing (m)	Number of BHEs
1	2.5	127
2	2.5	37
3	10	13

Algorithm 1: Optimization of Parameters using Genetic Algorithm and FEFLOW Model

Set: N_w = Number of parameters
 Set: N_{pop} = Number of population
 Set: N_{gen} = Number of generations

begin

 Generate new population, $\text{New}_{\text{pop}} = (N_{pop}, N_w)$

for $generation = 1, 2, \dots, N_{gen}$ **do**

 Implement the fitness function

for $i = 1, 2, \dots, N_{pop}$ **do**

 Insert parameters into the FEFLOW model

 Run Feflow model and calculate recovery efficiency (RE)

$RE = \frac{\nabla H_{recovered}}{\nabla H_{stored}} \times 100\%$

 Select the best chromosomes in the current population to join the mating pool:

for $i = 1, 2, \dots, N_{pop}$ **do**

 Eliminate the chromosome with the least RE

 Replace it with the chromosome with the highest RE

parent = (N_{pop}, N_w)

 Generate next generation using the crossover function:

offspring = (N_{pop}, N_w)

 Add variation to the offspring using the mutation function:

New_{pop} = (N_{pop}, N_w)

end

253 **4 Results**

254 *4.0.1 Optimization for 2.5 m BHE spacing*

255 The optimization process for the BTES with a 2.5 m BHE spacing considered two scenarios. In the first
256 scenario, a large number of BHEs, approximately 127, were deployed, as shown in Figure 3a. The second
257 scenario involved fewer BHEs, approximately 37, as depicted in Figure 3b.

258 The results of the optimization process for the two scenarios are shown in Figures 4 and 5. For scenario
259 1, after 30 generations, the optimal recovery efficiency reached approximately 65% by the 28th generation.
260 At this point, the charging and discharging flow rates were approximately $18 \text{ m}^3/\text{day}$ and $39 \text{ m}^3/\text{day}$,
261 respectively. Scenario 2, with fewer BHEs, achieved an approximate recovery efficiency of 52.9% at the 29th
262 generation. The charging and discharging flow rates were approximately $20.5 \text{ m}^3/\text{day}$ and $36.9 \text{ m}^3/\text{day}$,
263 respectively. The results of the analysis provide compelling evidence indicating a direct relationship between
264 the increase in discharging flow rates and the corresponding rise in recovery efficiency. Upon closer exam-
265 ination of Figure 4, a discernible trend emerges wherein successive generations witness a notable uptick
266 in both discharging flow rates and recovery efficiency. This suggests a positive correlation between these
267 variables over time. Additionally, Figure 5 further reinforces this observation by highlighting instances
268 where the algorithm generates higher discharging flow rates, coinciding with significant spikes in recovery
269 efficiency. These findings underscore the importance of monitoring and optimizing discharging flow rates as
270 a critical factor in enhancing overall recovery efficiency within our operational processes. By understanding
271 and leveraging this relationship, we can effectively improve our process performance and achieve greater
272 efficiency.

273 One of the primary goals of optimizing the recovery efficiency in a BTES system is to maximize the
274 amount of heat recovered from the subsurface while minimizing costs during the design, construction,
275 operation, and production phases of the heat storage and retrieval process. A significant cost factor is the
276 number of wells drilled for the BTES system. More boreholes result in higher initial and maintenance costs.
277 The results of the recovery efficiency after the optimization process for the two scenarios show that using
278 fewer BHEs in scenario 2 (37 BHEs) compared to scenario 1 (127 BHEs) did not result in poor performance.
279 Scenario 2 achieved a RE of 52.9%, despite having less than a third of the boreholes used in scenario 1.
280 Therefore, the optimization process revealed that a substantial reduction in the number of boreholes can
281 still yield significant heat recovery from the BTES system without incurring the additional costs associated
282 with a larger number of boreholes during the design and operation phases.

283 The analysis also focused on examining the temperature distributions within the BTES model under
284 two different scenarios, as visually represented in Figure 6 and Figure 7. In Scenario 1, characterized by a
285 higher number of 127 BHEs, the temperature distribution appears to be more extensive across the model
286 domain compared to Scenario 2. This disparity arises because Scenario 1 features a greater number of BHEs

287 covering a larger area. Consequently, there are more points within the system where heat exchange occurs,
288 leading to a more even distribution of heat throughout the system. This results in a smoother transition in
289 temperature from the inlet to the outlet, with the temperature change being more gradual and uniform along
290 the length of the BTES system. Conversely, in Scenario 2, which has fewer BHEs covering a smaller area, the
291 distribution of heat exchange points is more limited. As a result, heat transfer becomes more concentrated
292 and localized around these points. This concentration of heat exchange leads to a more abrupt and less
293 gradual temperature change from the inlet to the outlet compared to Scenario 1. Consequently, Scenario 2
294 exhibits a steeper temperature gradient along the length of the BTES system. Moreover, the higher number
295 of BHEs covering a larger area in Scenario 1 results in a greater density of heat exchange locations per unit
296 area of the model domain. This increased density provides more opportunities for heat exchange to occur
297 across a larger portion of the BTES domain compared to Scenario 2, where fewer BHEs cover a smaller
298 portion of the model domain. Leading to scenario 1 achieving higher recovery efficiency than scenario 2.

299 Additionally, another contributing factor to the lower temperature levels observed in Scenario 2 could
300 be the migration of a substantial amount of heat to areas devoid of BHEs. This dispersion of heat poses
301 challenges in its retrieval, thereby increasing the likelihood of heat dissipation. Conversely, Scenario 1
302 benefits from a more extensive coverage of BHEs across a larger portion of the model domain, facilitating
303 more efficient heat retention and retrieval mechanisms. Thus, the interplay between BHE distribution, heat
304 storage capacity, and the dynamics of heat migration in the surrounding environment elucidates the nuanced
305 variations in temperature distributions and the utilization of heat observed between the two scenarios.

306 *4.0.2 Optimization for 10 m BHE spacing*

307 This optimization process involved a 10 m BHE spacing, as illustrated in Figure 8. After 30 generations,
308 the optimal recovery efficiency of approximately 14.9% was achieved in the 30th generation (Figure 9).
309 The charging and discharging flow rates were $24.8 \text{ m}^3/\text{day}$ and $36.2 \text{ m}^3/\text{day}$, respectively. Throughout the
310 process of optimization, there was a little improvement in the overall recovery efficiency. This observation
311 becomes particularly evident upon examining the trends in both charging and discharging flow rates.
312 Across successive generations, there seemed to be minimal changes (increase or decrease) between the
313 previous parameters of charging and discharging flow rates and the subsequent ones generated. However,
314 an interesting pattern emerges when there's a notable difference between the discharging and charging flow
315 rates, specifically when the former exceeds the latter. It is in these instances that a significant increase in
316 recovery efficiency was observed. Moreover, it is worth noting a stark contrast in the flow rate dynamics
317 between this scenario and scenarios 1 and 2. Here, both the charging and discharging flow rates appear to
318 be relatively high. This deviation from the previous scenarios, where the charging flow rates were notably
319 lower compared to discharging flow rates, can be attributed to the larger distances between the BHEs.
320 The greater distance between the BHEs necessitates a higher flow rate to effectively circulate heat through

321 the system and extract it efficiently. Consequently, the requirement for higher flow rates in this scenario
322 highlights the intricate interplay between system design parameters, such as BHE spacing, and operational
323 variables like flow rates, all of which impact the overall efficiency and effectiveness of the system.

324 Similar to Scenario 1, the cross-section of the model domain reveals a broad temperature distribution.
325 Additionally, the wider spacing between BHEs leads to extensive temperature migration and interaction
326 with surrounding materials, resulting in a decrease in temperature. As depicted in Figure 10, there is a
327 noticeable temperature reduction between successive BHEs. Specifically, the surrounding temperature of
328 the last BHE is approximately 13°C , which is close to the boundary temperature.

329 The temperature profile and recovery efficiency for a 3-year cycle are depicted in Figure 11 for both
330 2.5 m (scenario 1) and 10 m spacings. The analysis reveals distinct trends. During charging, the 10m
331 spacing exhibits a rapid initial temperature rise followed by a gradual increase, while the 2.5 m spacing
332 shows a steady incremental change throughout the charging phase. Throughout the 3-year cycle, the 2.5 m
333 spacing records consistently higher temperatures during the charging phase, with the temperature difference
334 between the two spacings increasing in successive cycles. During discharging, the 2.5 m spacing also records
335 higher temperatures compared to the 10m spacing, with both systems experiencing significant temperature
336 increases in successive cycles, however the 2.5 m spacing scenario recovered more heat than the 10m spacing.
337 These temperature profile trends are reflected in the recovery efficiency, with the 2.5 m spacing consistently
338 achieving higher heat recovery across the 3-year cycle. The difference in recovery efficiency between the two
339 BHE spacings widens at each successive year, indicating that the 2.5 m spacing will continue to recover
340 more heat than the 10m spacing in additional cycles.

341 4.0.3 Base case vs optimum case

342 The BTES system for scenario 1 underwent simulation spanning a six-year cycle to conduct a comparative
343 analysis between a base case scenario, characterized by a charging and discharging flow rate of $20 \text{ m}^3/\text{day}$,
344 and the optimized configuration of scenario 1. Throughout this simulation, the dynamic interplay between
345 charging and discharging cycles and their consequent impact on outlet temperature variations highlighted
346 the critical importance of optimizing operational parameters for achieving peak system performance.

347 Figure 12 provides a comparative overview of the recovery efficiency between the base and optimized
348 scenarios. Within the initial year, the base case attained a recovery efficiency of 34.1%, while the optimized
349 scenario showcased a notably higher efficiency of 62.2%. This difference in recovery efficiency signifies
350 that a larger proportion of the energy injected is recovered during the discharging phase in the optimized
351 scenario, thereby enhancing the energy efficiency and cost-effectiveness of the BTES system. Furthermore,
352 it also implies a diminished environmental footprint, as the system operates with greater efficiency, thereby
353 minimizing energy wastage.

354 To comprehensively evaluate the long-term efficacy of the BTES system, it's imperative to consider the
355 recovery rates across multiple cycles. Over the span of the six-year cycle, the base case peaked at a recovery
356 efficiency of 60.6%, whereas the optimized scenario consistently outperformed, achieving a significantly
357 higher recovery efficiency of 96.5%. The widening gap in recovery efficiency between the optimized and
358 base scenarios with each successive cycle underscores the optimized scenario's ability to extract more heat
359 per cycle compared to the base scenario, indicating sustained superior performance over time.

360 **5 Discussion**

361 Several sensitivity analysis studies have been conducted to determine the impact of design, operational,
362 and geological parameters on BTES system efficiency. These studies focus on optimizing the heat transfer
363 process, minimizing heat loss, and enhancing overall system performance. Han and Yu (2016) conducted
364 a sensitivity analysis on geological attributes (initial ground temperature, groundwater velocity, and ther-
365 mal properties of materials), design factors (borehole depth), and operational parameters (working fluid
366 velocity, inlet temperature, and intermittent mode) for a vertical Ground Source Heat Pump (GSHP). The
367 results showed that geological properties, borehole depth, and working fluid velocity significantly influence
368 performance, while specific heat capacity showed no noticeable impact. Wołoszyn and Golaś (2014) ana-
369 lyzed BTES efficiency with single U-tube boreholes, focusing on geological thermal properties, including
370 thermal conductivity, specific heat, and density of the rock mass and grout material. The study found that
371 thermal conductivity of the rock mass had the most significant impact. Wołoszyn (2018) conducted a global
372 sensitivity analysis (GSA) on BTES efficiency during long-term operation, assessing the influence of BHE
373 arrangement parameters (distance between BHE axes in the x-direction and y-direction, and the angle
374 between the top surface of the rock mass and borehole axes). The results showed that BHE inclination
375 crucially impacts BTES efficiency. Baser and McCartney (2015) studied the influence of different variables
376 on a BTES model with five boreholes, including heat injection rate, duration, ground thermal conductivity,
377 and borehole spacing. The study indicated that soil with lower thermal conductivity had less lateral heat
378 loss, and arrays with smaller borehole spacing allowed more concentrated heat storage at higher tempera-
379 tures. Kumawat et al. (2024) conducted a sensitivity analysis of BTES modeling on a wedge-shaped model,
380 examining five parameters: well spacing, grout thermal conductivity, charging and discharging rates, and
381 soil thermal conductivity. The analysis revealed that BHE spacing and volumetric flow rate had the highest
382 impact on roundtrip efficiency.

383 Other optimization techniques for BTES systems have also been studied. Schulte et al. (2015) presented
384 an approach for simulating and optimizing borehole thermal energy storage systems, using a software tool
385 to optimize the number and length of borehole heat exchangers based on specific annual heat demand.
386 The tool effectively determined the ideal size of the thermal energy storage, showing that BTES systems

387 operate efficiently in large-scale applications. Fiorentini and Baldini (2021) proposed a control-oriented
388 numerical optimization model to determine the best operating conditions for a heat pump-driven BTES,
389 aiming to reduce yearly CO₂ emissions by shifting the heat pump's electricity load based on CO₂ intensity
390 profiles. The study demonstrated that this boundary condition is crucial for optimal system operation.
391 Rapantova et al. (2016) optimized the lengths of charging and discharging cycles to minimize heat loss
392 due to ground dissipation, finding borehole depth optimization crucial for reducing heat exchange surface
393 area and heat losses. Lanini et al. (2014) established a 1D analytical model and a 3D multilayer numerical
394 model, validated against experimental data, to simulate different configurations over many years. The study
395 included a full-scale experiment evaluating the energetic potential of BTES, showing that the heat transfer
396 fluid lost 15% of its energy at a depth of 100 m and 25% at 150 m.

397 The conclusions from these studies primarily analyze the influence of various BTES parameters on
398 system performance. While geological parameters (e.g., ground thermal conductivity) significantly affect
399 performance, they cannot be easily altered to optimize heat recovery. However, operational and design
400 parameters, as noted in the literature, also have a significant influence and can be regulated for optimal
401 recovery. Existing theoretical research on optimizing BTES parameters for optimal heat recovery is insuffi-
402 cient. In this paper, we propose a method using a genetic algorithm to optimize BTES recovery efficiency
403 by automatically adjusting the operational parameters (charging and discharging flow rate) of the BTES
404 system. The results were promising, indicating that by using the genetic algorithm to calibrate the BTES
405 operational parameters, and adjusting the BHE spacing, optimal recovery efficiency was achieved after the
406 optimization process.

407 6 Conclusion

408 This research study aimed to optimize the operation of BTES systems by utilizing a genetic algorithm to
409 compute optimal parameters for maximizing energy recovery. The study focused on design, operational, and
410 geological parameters. Design parameters included thermal conductivity of the grout and BHE spacing, with
411 two spacing configurations considered. The geological parameter was the subsurface's thermal conductivity,
412 of which both the subsurface's thermal conductivity and the grout thermal conductivity were held constant.
413 The optimization algorithm focused on operational parameters, such as charging and discharging flow rates.
414 Finite element simulation was employed to simulate the BTES system and compute recovery efficiency as
415 the performance metric for testing the optimized parameters.

416 The results highlighted the intricate relationship between recovery efficiency, BHE spacing, and the
417 volumetric charging and discharging flow rates within the BTES system. It was evident that smaller BHE
418 spacing correlated with higher recovery rates, while, within a specific BHE spacing, higher discharging flow
419 rates contributed to enhanced recovery efficiency. However, an interesting observation emerged regarding

420 the interplay between BHE spacing and flow rates during charging. It was noted that when the BHE spacing
421 was smaller, the flow rate during charging tended to be lower compared to configurations with larger BHE
422 spacing. This suggests a trade-off between BHE spacing and flow rates during charging, indicating the
423 need for careful consideration when optimizing these parameters. Moreover, an optimal BTES model was
424 developed and compared against the base case. The findings revealed that the optimal model achieved an
425 impressive recovery efficiency of 96.5% over a simulated six-year period. This highlights the effectiveness of
426 optimizing system parameters in maximizing energy recovery and overall system performance.

427 By delving into the impact of various parameters on system performance, this study contributes to the
428 use of mathematical algorithms for obtaining optimal parameters, ultimately enhancing the efficient design
429 and operation of BTES systems.

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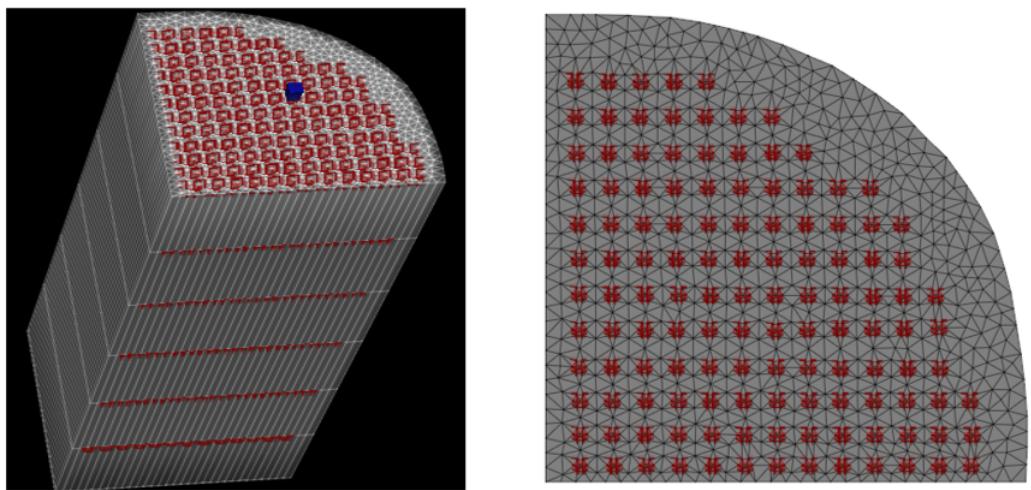


Fig. 1 Finite element model grid for this study

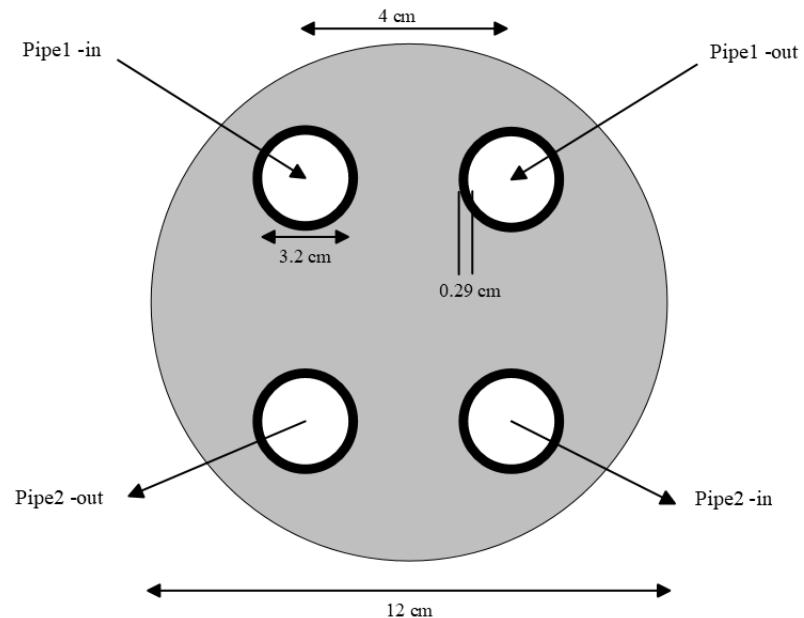


Fig. 2 Cross-section of the borehole heat exchanger

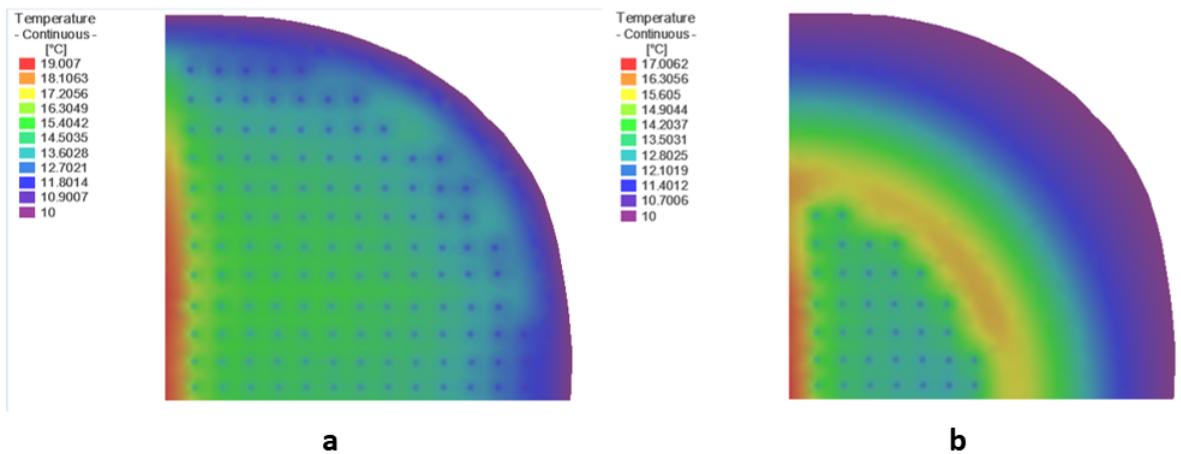


Fig. 3 Top view of a simulated BTES system containing: a. 127 BHEs and b. 37 BHEs

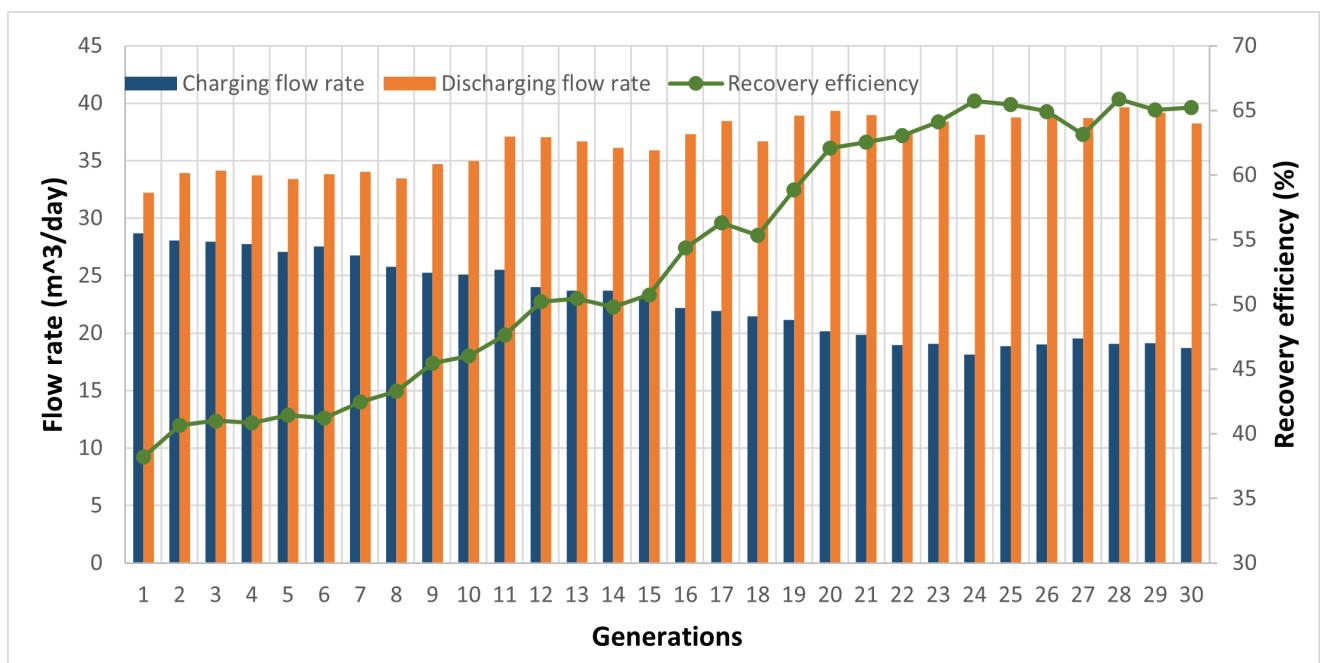


Fig. 4 Optimal parameters and recovery efficiency for scenario 1

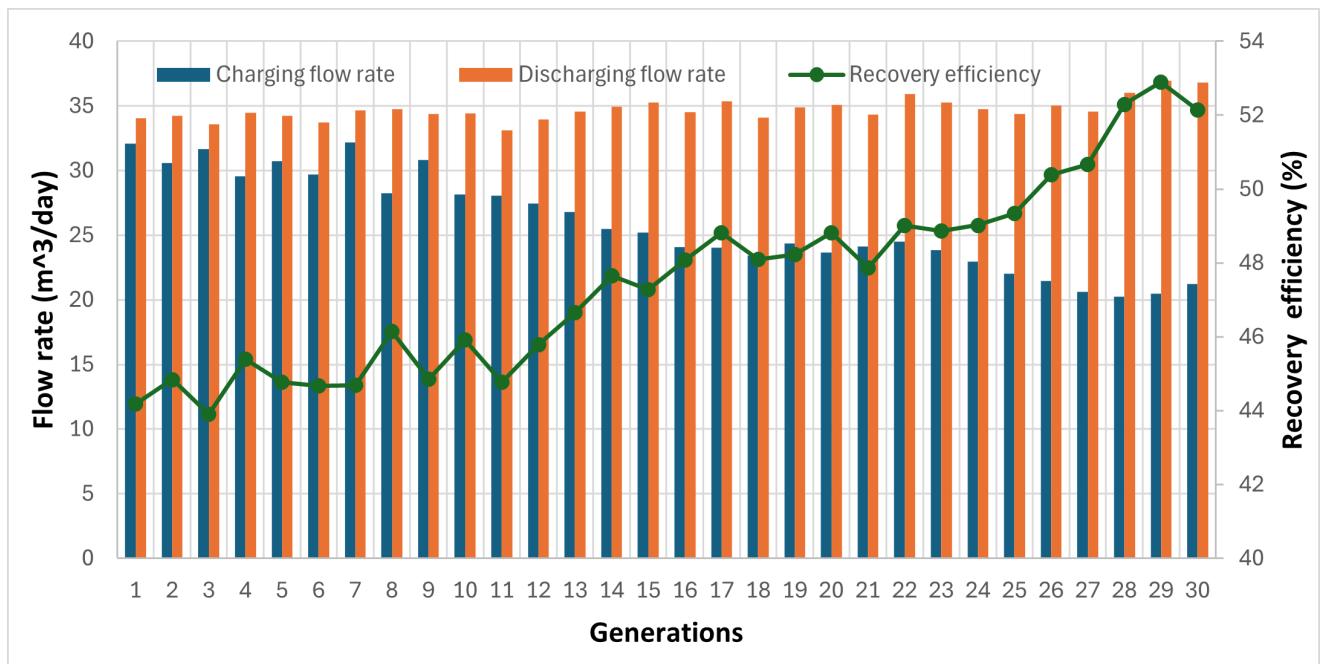


Fig. 5 Optimal parameters and recovery efficiency for scenario 2

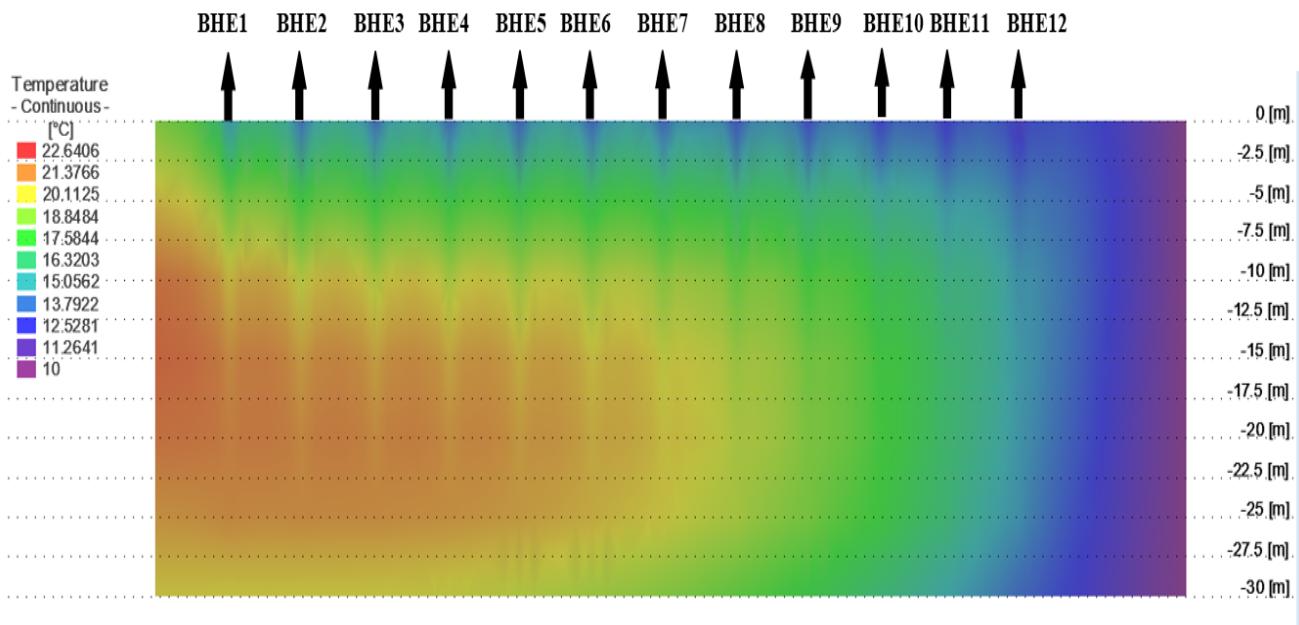


Fig. 6 Cross-section of the BTES model showing spatial distribution of temperature after 3-year cycle for scenario 1

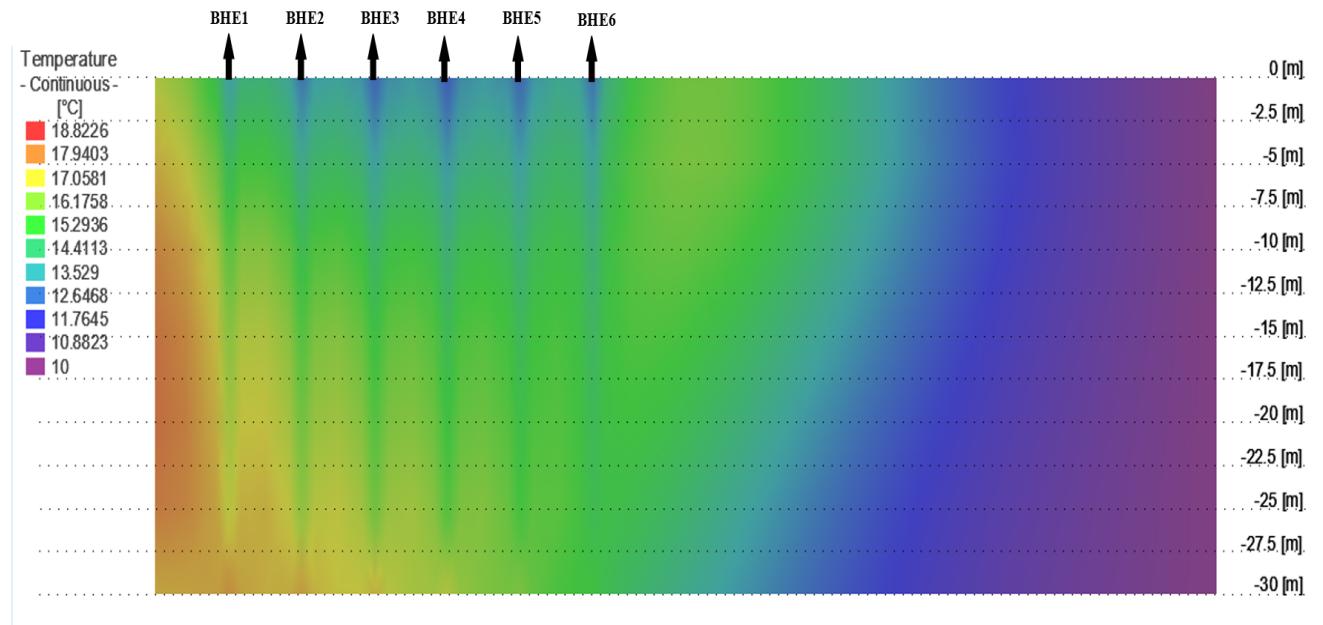


Fig. 7 Cross-section of the BTES model showing spatial distribution of temperature after 3-year cycle for scenario 2

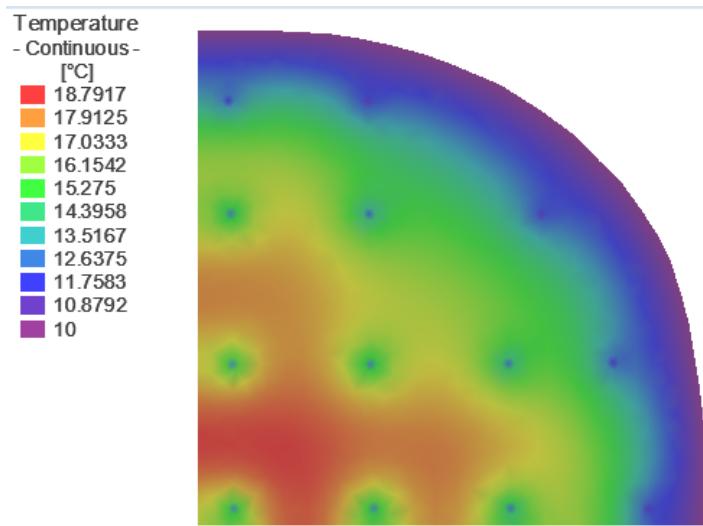


Fig. 8 Top view of the simulated BTES system for 10m BHE spacing

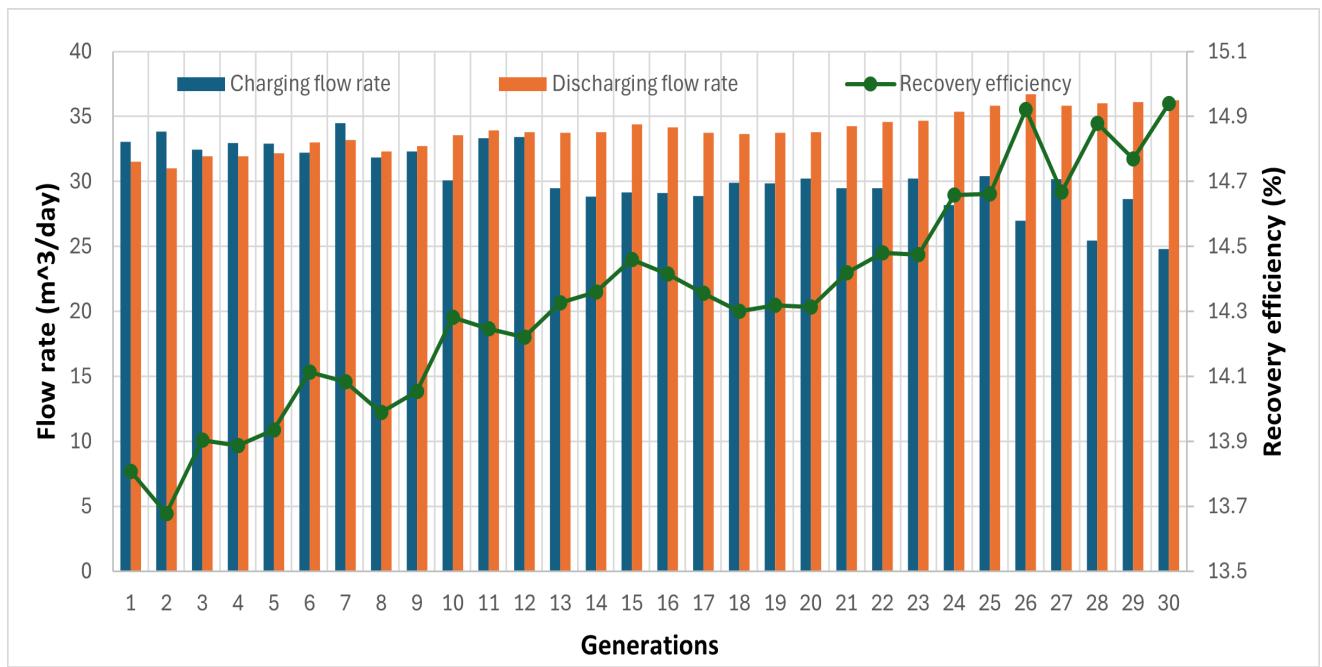


Fig. 9 Optimum parameters and recovery efficiency for scenario 3

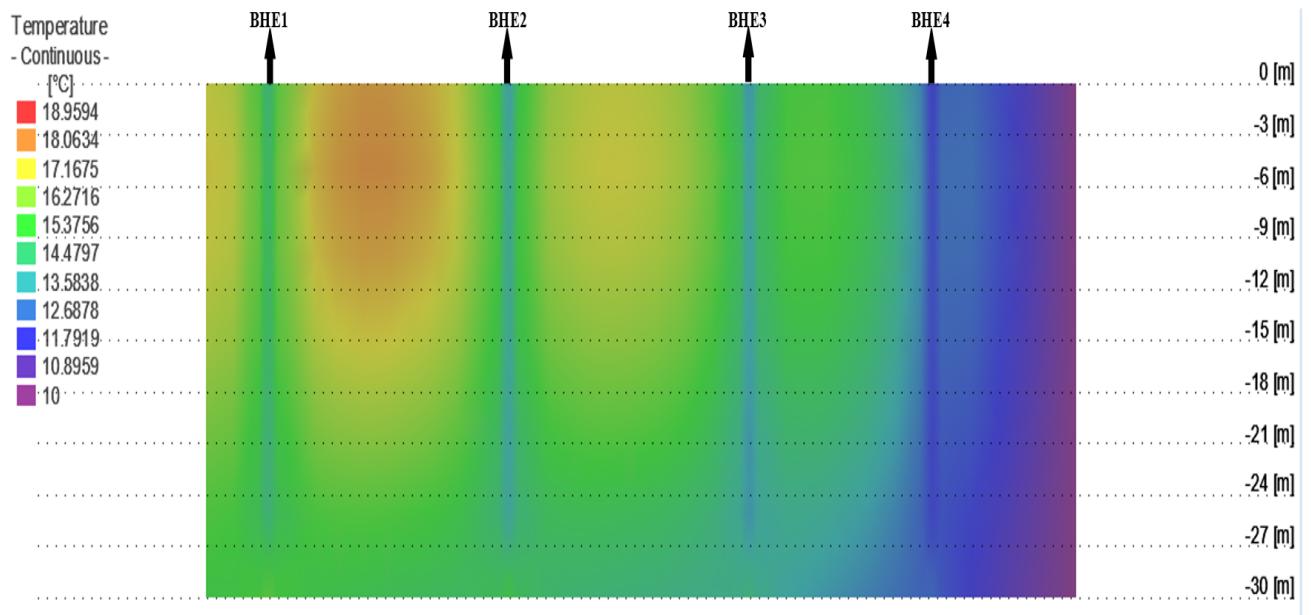


Fig. 10 Cross-section of the BTES model showing spatial distribution of temperature after 3-year cycle for scenario 3

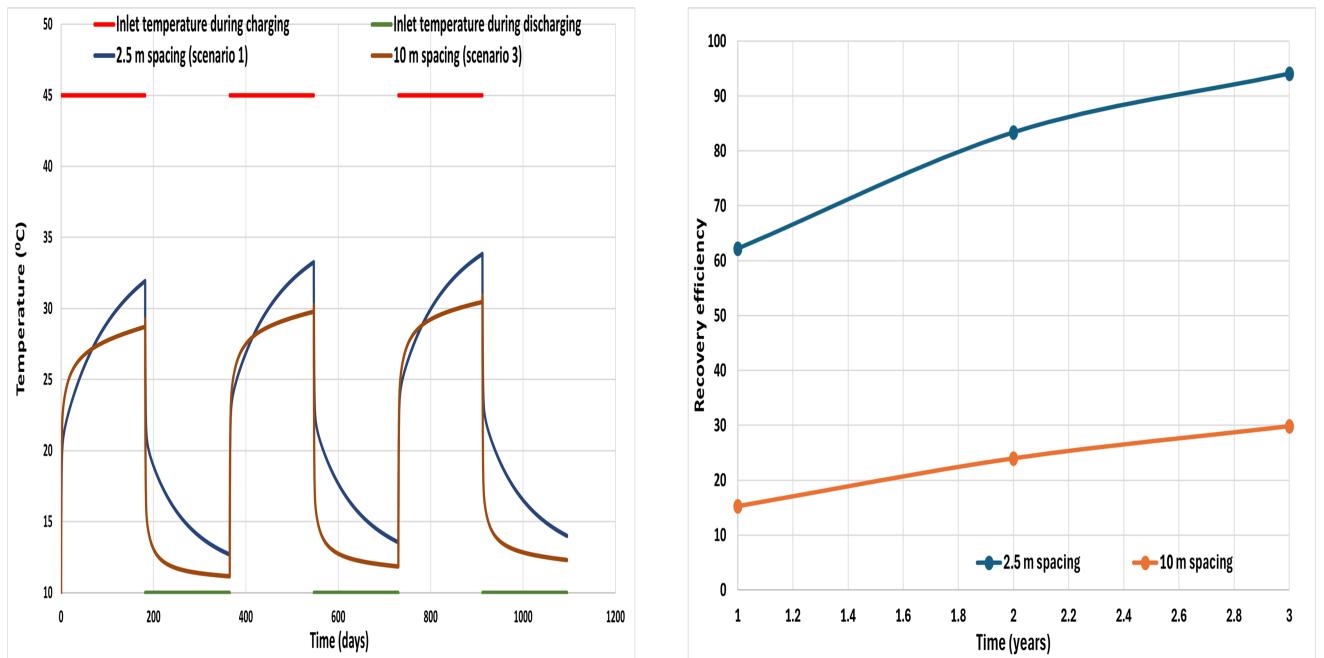


Fig. 11 Temperature profile and recovery efficiency for a 3-year cycle

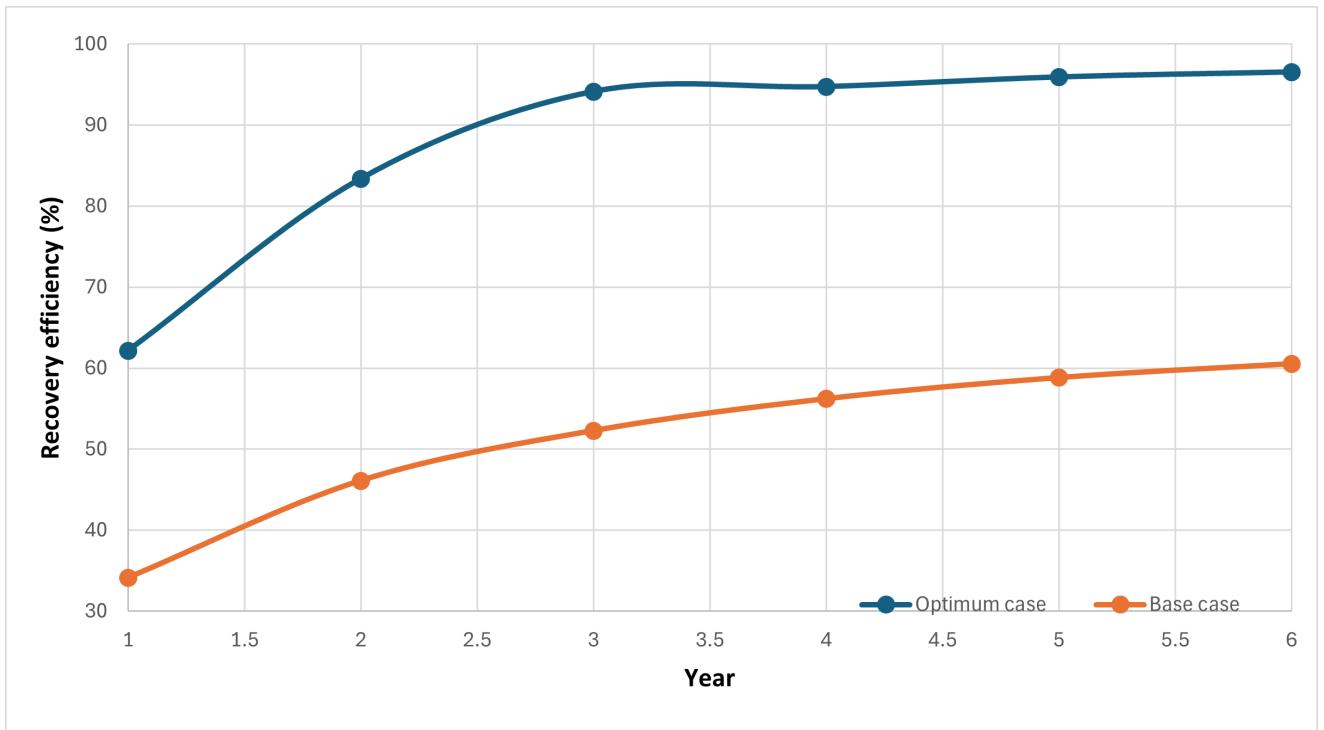


Fig. 12 Recovery efficiency of base case and optimum case